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Aerodynamics performance of a ventilation system for droplets controlling by coughing in a hospital isolation ward

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Abstract

Over 270 million people have been infected by the coronavirus disease 2019 (COVID-19) in the past two years, resulting in 5 million deaths. The virus primarily transmitted through droplets or aerosols produced by coughing, sneezing, and talking. A full-scale isolation ward in Wuhan Pulmonary Hospital is modeled in this work, and water droplet diffusion is simulated using the Computational Fluid Dynamics (CFD) approach. In an isolation ward, a local exhaust ventilation system is intended to avoid cross-infection. The existence of a local exhaust system increases turbulent movement, leading to a complete breakup of the droplet cluster and improved droplet dispersion inside the ward. When outlet negative pressure is 4.5 Pa, the number of moving droplets in the ward decreases by approximately 30% compared to the original ward. The local exhaust system could minimize the number of droplets evaporated in the ward; however, the formation of aerosols cannot be avoided. Furthermore, in six different scenarios there are 60.83%, 62.04%, 61.03%, 60.22%, 62.97% and 61.52% droplets produced by cough deposited on the surface of the patient. However, the local exhaust ventilation system has no apparent influence on the control of surface contamination. This study provides several suggestions for the optimization of ventilation in the wards, and scientific evidence to ensure the air quality of hospital isolation wards.

Keywords Computational fluid dynamics (CFD), Virus airborne diffusion, Local exhaust ventilation system

32 **Introduction**

33 The study of airborne diseases has been increasingly significant, recently, with the emergence of
34 severe acute respiratory syndrome, Ebola virus, tuberculosis, coronavirus disease 2019 (COVID-
35 19), and other infectious diseases (Hathway et al. 2011). With the outbreak of the COVID-19
36 worldwide (as of January 20, 2022), there have been 332,617,707 people diagnosed with COVID-
37 19, including 5,551,314 deaths globally (WTO 2022). Some countries or regions adopted closed-
38 off management to prevent the spread of the disease, which seriously affected economic growth and
39 the normal operation of the whole city. Over the last two years, countries have prioritized preventing
40 and controlling the epidemic. Every day, 10,000 people worldwide die as a result of COVID-19. It
41 can affect anyone anywhere, especially the healthcare workers exposed to the environment existing
42 virus droplets. Therefore, the study of virus airborne diffusion in hospital isolation rooms is
43 significant.

44 Viral respiratory infections are diffused by contact and droplet transmission, fomite transmission,
45 and other types of transmission (WTO 2020). Contact transmission occurs when a healthy man is
46 in close contact with a person who is infected (direct contact) or when an infected individual exhaust
47 fomites containing viral droplets (indirect contact) (Morawska et al. 2020). Viruses spread through
48 the air via droplets or aerosols produced during coughing, sneezing, and talking (Jones et al. 2015).
49 As an intense process of droplet generation for transient time, recently the focus of study has been
50 on cough. Small virus droplets expelled into the air can exist as airborne for a long time (WTO 2009)
51 and can fill the whole room under the effect of an air conditioner (Liu 2020). Li et al. (2020)
52 investigated the spread of droplets in the tropical outdoor environment and found that at wind speeds
53 of 2 m/s, 100 μm droplets can move up to 6.6 m. Wei et al. (2015) modeled the coughing process as
54 a turbulence jet to investigate diffusion of droplets and obtained travel distances with different
55 droplet sizes. Lindsley et al. (2016) compared aerosol droplets samples of 61 adult volunteer
56 outpatients through experiments and found that more aerosol droplets containing viruses were
57 released by coughing. Nicas et al. (2005) analyzed the pathogen-containing droplet emission from
58 an infected person, and results indicated that the droplets are composed of 98.2% water and 1.8%
59 of nonvolatile solid compounds.

60 Furthermore, COVID-19 outbreak can spread from one room to another via a ventilation system.
61 Nissen et al. (2020) investigated the ventilation system in a COVID-19 isolation unit at Uppsala
62 University Hospital through an experiment. They found that the virus could spread long distances
63 via central ventilation systems. And other researches had also found viral RNA in samples from
64 ventilation systems in isolation rooms (Guo et al. 2020; Liu et al. 2020). Thus, long distances
65 airborne dispersal of COVID-19 virus in hospital wards exist and significantly increases cross-
66 infection risk.

67 Computational fluid dynamics (CFD) has been widely utilized to analyze the mobility of droplets
68 indoors. Davardoost and Kahforoushan (2019) simulated the dispersion of volatile organic
69 compounds indoor via CFD modeling to investigate the performance of ventilation. Liu et al. (2020)
70 used a full-experimental and numerical simulation to investigate the transmission and deposition of
71 bioaerosols in a hospital isolation ward in which the Re-Normalisation Group (RNG) $k-\epsilon$ model was
72 selected. Bhattacharyya et al. (2020) studied the flow field in an isolation ward when both air-
73 conditioning system and sanitizing machine kept working through CFD method in which transition

74 SST $k-\varepsilon$ model was applied. They found that the high turbulent fields generated in the ward may be
75 conducive to the spread of sanitizer. Borro et al. (2020) used the CFD method to investigate the
76 droplet diffusion from a coughing event in the waiting room and recovery ward of Vatican State
77 Children's Hospital, and the RNG $k-\varepsilon$ model was employed in numerical simulation. Results
78 indicated that ventilation system accelerated the spread of infected droplets across in the whole
79 waiting room and the local exhaust ventilation system could help to remove the contaminated air.
80 Li et al. (2020) investigated the diffusion of droplets generated from the patient's wound under four
81 different ventilation systems during the surgery through CFD method and obtained the optimal
82 arrangement of the ventilation system. Cho (2018) evaluated the flow field of contaminated droplets
83 exhaled from patients in an isolation ward at three different ventilation systems and obtained an
84 improved ventilation system using numerical simulation and field measurement, which efficiently
85 removed contaminated droplets.

86 Although extensive study has been conducted on the simulation of droplets dispersion from a
87 coughing episode and contaminant dispersal throughout the ward, there has been little research on
88 the fast removal of contamination sources in an isolation ward. The main objective of this work is
89 to analyze propagation characteristics of droplets containing viruses induced by coughing in the
90 ward and to design a local exhaust ventilation system to control droplets diffusion. A full-scale
91 experiment is conducted to verify the flow field. The temporal and spatial distribution of droplets
92 in six scenarios are simulated through the CFD method. Additionally, the number of moving,
93 evaporated, and deposited droplets is quantitatively analyzed. This study provides several guidelines
94 for optimizing ventilation in hospital wards, and scientific evidence for ensuring the air quality of
95 hospital isolation wards.

96 **Problem description**

97 The main transmission route of infection is that viruses spread through the air via droplets or
98 aerosols produced during coughing, sneezing, and talking (Jones et al. 2015). A high concentration
99 of contaminated droplets in the hospital epidemic ward raises the danger of infection from patients
100 to health care staff as well as patient cross-infection. A ventilation system ensures the air quality
101 indoors and protects healthcare workers and patients from contaminants (Liu et al. 2020). Therefore,
102 it is important to design an effective ventilation system to reduce droplet diffusion in the hospital
103 wards. Based on the above questions, coughing and droplets release processes in the isolation ward
104 of Wuhan Pulmonary Hospital are simulated through the CFD method to improve the ventilation
105 system's performance. The geometric model is established based on the actual measurement of the
106 isolation ward in Wuhan Pulmonary Hospital as shown in Fig. 1.

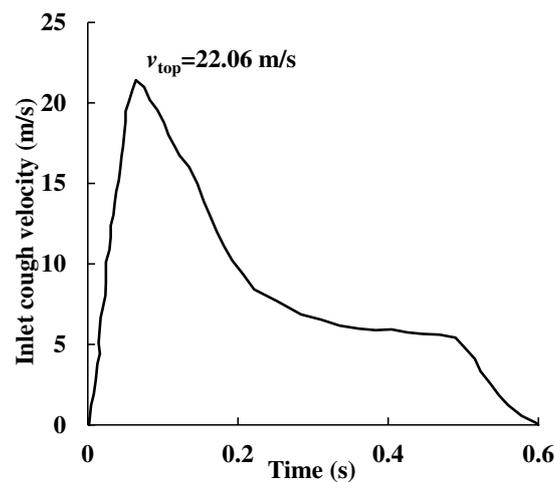
107 According to Zhu et al. (2020), Yao et al. (2020), and Ke et al. (2020), COVID-19 was an
108 enveloped virus with a large nucleoprotein, and virions ranging in size from 60 to 140 nm.
109 Furthermore, Bi et al. (2021) discovered that the diameter of cough followed by the Rosin-Rammler
110 distribution, with an average diameter of 0.02 mm. Compared with the droplets, the size of virions
111 can be ignored since they are tiny. We assume that all droplets contain viral droplets to simplify the
112 model in the simulation. The material of droplets is simplified as water according to the investigation
113 of Nicas et al. (2005). The droplets are injected from the patient's mouth in the time range of 0.042-
114 0.136 s and the cough velocity can reach 22.06 m/s in a short period (Borro et al. 2020; Gupta et al.
115 2011), as shown in Fig. 2. Droplets properties used in the simulation are shown in Table 1.



116
117 **Fig. 1** Layout of an isolation ward in Wuhan Pulmonary Hospital.

118 **Table 1.** Droplets properties used in discrete phase model (Nicas et al. 2005; Borro et al. 2020;
119 Gupta et al. 2011)

Material	Diameter distribution	Diameter (mm)	Velocity (m/s)	Temperature (K)
H ₂ O	Rosin-Rammler	0.002~0.1	22.06	310



120
121
122 **Fig. 2** Velocity profile of one cough by patient [13].

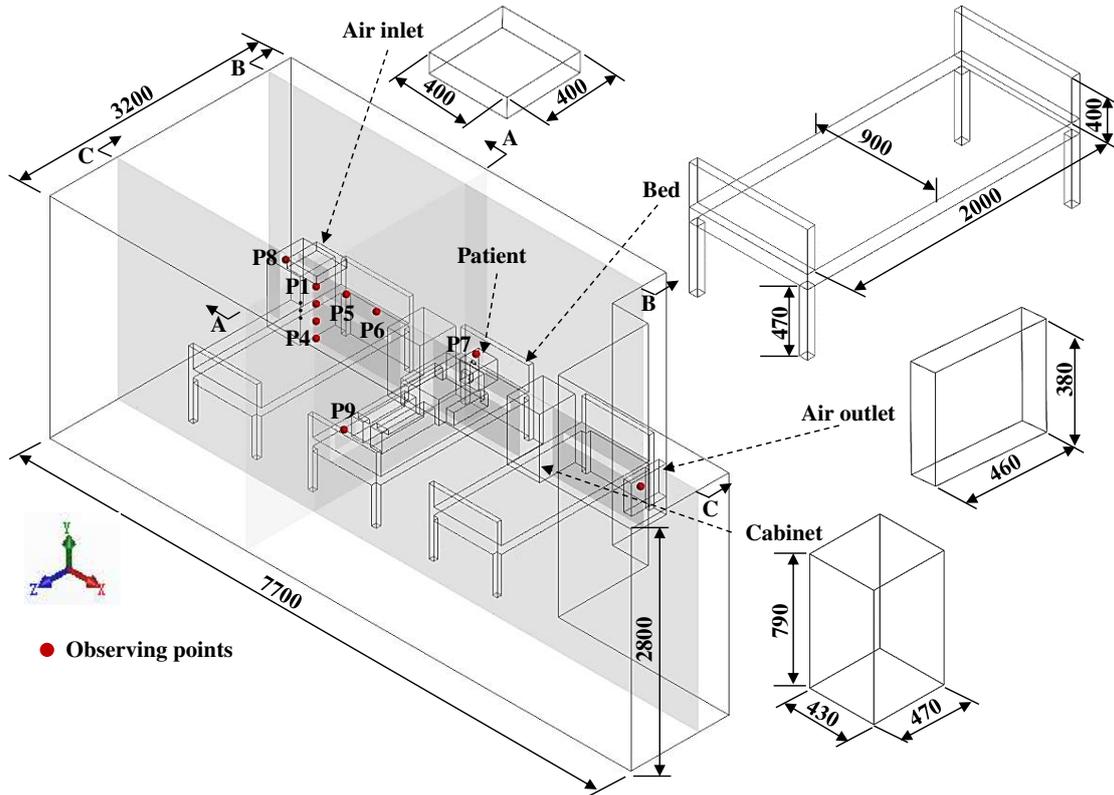
123 **Method**

124 **Geometry and mesh**

125 The geometric model is established based on the actual measurement of the isolation ward in Wuhan
126 Pulmonary Hospital. The patient's geometry has made some necessary simplifications to reduce the
127 computational cost. There is a naturally ventilated air outlet near the bed, and a ventilation system
128 located on the ceiling delivers fresh air to the ward. The patient is placed in the middle bed in this
129 model. Fig. 3 plots the schematic of the ward and the furniture's dimension.

130 The meshes are generated using the Gambit (ANSYS, Gambit 2.4.6, Canonsburg, PA, USA). The
131 hybrid grid method is used in the gridding drawing process. The meshes of the furniture zone are

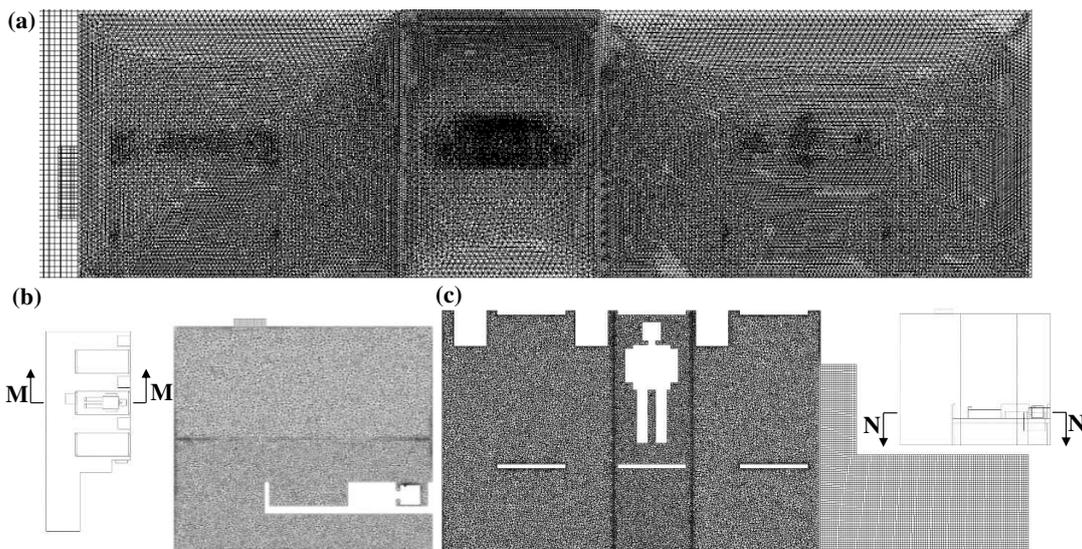
132 generated using tetrahedron elements to ensure the quality of the mesh, which is shown in Fig. 4(a).
 133 Furthermore, the structured grid is used in the regular fluid zone to reduce the amount of calculation.
 134 Therefore, the meshes surrounding the patient are defined as plotted in Fig. 4(b) and Fig. 4(c) from
 135 the cross-section M-M and N-N respectively. The whole zone of fluid consists of tetrahedron and
 136 hexahedron elements with 5,807,108. In our previous work (Cai et al. 2016; Zhang et al. 2015; Song
 137 et al. 2021), the same method generates the model's mesh. In other words, the validity of the mesh
 138 has already been verified.



139

140

Fig. 3 Schematic of the ward and the dimension of furniture (unit: mm).



141

142

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Fig. 4 Meshes for the CFD simulation (a) mesh of the furniture, (b) mesh of the manikin on the cross-section M-M, (c) mesh of the manikin on the cross-section N-N.

144 Airflow modeling

145 The flow field is simulated using the CFD solver ANSYS Fluent (Fluent 2020 R2, ANSYS,
146 Canonsburg, PA, USA). To simulate the indoor turbulence flow field, three turbulence models are
147 extensively used: Reynolds Average Navier-Stokes (RANS), direct numerical simulation (DNS),
148 and large eddy simulation (LES) (Choi et al. 2012; Wang et al. 2018). In the indoor flow field
149 simulation, the DNS and LES require longer computation times and more computer capacity than
150 the RANS. Considering the accuracy and amount of the calculation, the RANS equations are used
151 (Chen et al. 1995).

152 The continuity and the Navier-Stokes equation can be written as follows:

$$153 \quad \frac{\partial \bar{u}_m}{\partial x_m} = 0 \quad (1)$$

$$154 \quad \frac{\partial u_m}{\partial t} + \frac{\partial(\bar{u}_m \bar{u}_n)}{\partial \bar{u}_n} = -\frac{1}{\rho} \frac{\partial \bar{P}}{\partial x_m} + \nu \frac{\partial^2 \bar{u}_m}{\partial x_n \partial x_n} + \frac{\partial \tau_{mn}}{\partial x_n} \quad (2)$$

155 where u_m is the fluid velocity in the m direction, x_m is the coordinate in the m direction, and P
156 represents the pressure of the fluid. The overbar in the equations denotes the mean variables. And
157 the τ_{mn} is the Reynolds stress.

158 Because of cough, a turbulent jet event (Bourouiba et al. 2014), the RNG k - ε model in the RANS
159 method is used in the simulation, which has been proved effective in the simulation of indoor
160 turbulent flow field (Liu et al. 2020; Borro et al. 2020; Liu et al. 2017). This model is derived from
161 renormalization theory, in which an additional term is added in the epsilon equation to improve the
162 accuracy of turbulence flow (Abraham et al. 1997; Yakhot et al. 1986). For solving method, the
163 pressure-implicit with splitting of operators algorithm is selected due to the suitability for transient
164 flows.

165 Discrete phase

166 To simulate the diffusion of droplets from a cough, the discrete model is used. The droplet force
167 balance equation in the discrete phase model theory is defined as (Fluent ANSYS 2020):

$$168 \quad m_d \frac{d\vec{u}_d}{dt} = m_d \frac{\vec{u}_f - \vec{u}_d}{\tau_r} + \frac{m_d \vec{g}(\rho_d - \rho)}{\rho_d} + \vec{F} \quad (3)$$

169 where m_d represents the mass of droplet, \vec{u}_f is the velocity of the fluid, \vec{u}_d is the velocity of a
170 droplet, \vec{F} is an additional force and τ_r is the relaxation time of a droplet.

171 In the right-hand side of the equation, the term is the drag force; the second part represents a force
172 under the influence of gravity; the third part is additional forces under various circumstances, such
173 as forces in moving reference frames, thermophoretic force, Brownian force, Saffman's lift force,
174 Magnus lift force and user-defined force.

175 Drag force and gravity have the greatest influence on the dynamic behavior of droplets formed
176 by cough occurrences. The droplet motion is assumed as a stochastic collision. Other forces,
177 including thermophoresis and Brownian forces, are too small to be ignored in the simulation.
178 Therefore, droplets are mainly affected by the drag forces, generation of turbulence associated with

179 the ventilation system, and gravity force.

180 Evaporation of droplets

181 After exhaling infectious droplets through cough, small suspended droplets could form short-
182 duration droplets cloud. Then, droplets could suspend via air in the ward for a long time. The surface
183 of the droplet exchanges a lot of heat and mass with the ambient. In other words, a portion of the
184 droplets evaporates into water vapor suspended indoors for a long time. Therefore, the evaporation
185 process of the suspension droplets is of great significance in analyzing the risk of infectious droplet
186 transmission. Kukkonen et al. (1989) proposed a model of pure water droplets evaporation and
187 dispersion, driven by the partial pressure difference between the droplet surface and the ambient air.
188 Many studies about droplets evaporation are currently based on this work (Mao et al. 2020).

189 The droplet is assumed to be uniform in this study, which means that there is no gradient in
190 concentration or temperature within the droplet. Evaporation and convection are the main heat
191 exchangers between the droplet surface and the ambient air. The radiation heat transfer can be
192 neglected due to the minimal temperature difference. The temperature of the droplet changes is
193 determined by Eq. (4) (Parienta et al. 2011).

$$194 \quad m_d C_d \frac{dT_d}{dt} = \pi h_c D_d^2 (T_\infty - T_d) + L_t \frac{dm_d}{dt} \quad (4)$$

195 where m_d is the mass of the droplet, C_d is the heat capacity of water, T_d is the droplet temperature,
196 D_d is the droplet diameter, T_∞ is the ambient temperature and L_t is the latent heat of vaporization.
197 The left part of the equation indicates the total enthalpy change in a droplet. On the right-hand side
198 of the equation, the first part is the convection heat transfer between the air and the droplet's surface;
199 the second part represents the latent by droplet vaporization.

200 And the heat transfer coefficient h_c in Eq. (4) can be obtained as (Chao et al. 2006):

$$201 \quad Nu = \frac{h_c D_d}{k_g} = 2.0 + 0.6 Re_d^{1/2} + Pr^{1/3} \quad (5)$$

202 where k_g is the thermal conductivity of air, Nu is the Nusselt number, Pr is the Prandtl number, and
203 Re_d is the droplet Reynolds number.

204 Boundary condition

205 Before simulation, the inlet speed of the ventilation system is tested in Wuhan Pulmonary Hospital,
206 and the speed obtained is 1.045 m/s. The air is set to 295 K, and the walls are assumed as adiabatic.
207 Droplets are expected to be absorbed by surfaces (beds, cabinets, walls, and patient) as they collide
208 with the surfaces due to their low momentum. Therefore, we select the trap as the boundary
209 conditions of the surfaces. The other boundary conditions for CFD simulation are summarized in
210 Table 2.

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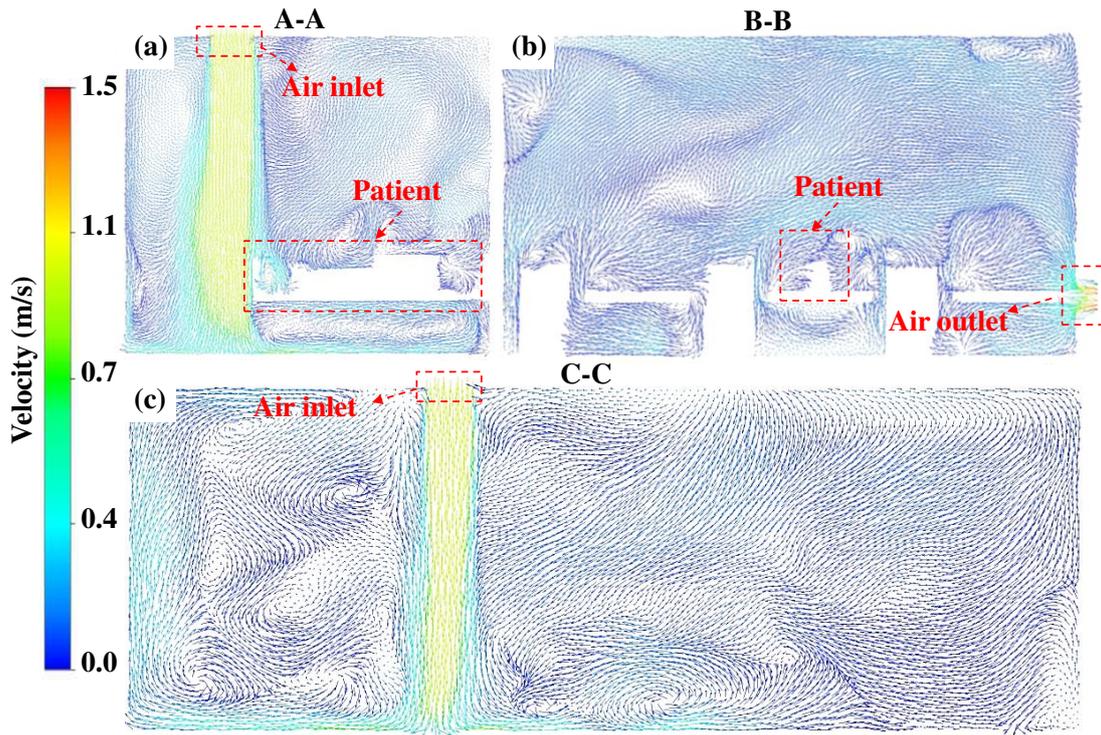
Table 2. Boundary conditions in the CFD simulation.

	Original	Ventilation_1	Ventilation_2	Ventilation_3	Ventilation_4	Ventilation_5
		Outlet pressure (Pa)				
Ventilation outlet	/	-2.5	-3.0	-3.5	-3.5	-3.5
		Outlet temperature: 300 K				
		DPM condition: Escape				
		Inlet velocity: 1.045 m/s				
Air inlet		Inlet temperature: 295 K				
		DPM condition: Escape				
		Outlet pressure: zero-gauge pressure				
Air outlet		Outlet temperature: 300 K				
		DPM condition: Escape				
		Droplet velocity: 22.06 m/s (in <i>Y</i> direction)				
Droplet inlet		Temperature: 310 K				
		DPM condition: Escape				
Surfaces (beds, cabinets, walls, and patient)		Temperature: 300 K				
		DPM condition: Trap				

216 Results and discussion

217 Validation of the flow filed

218 This study investigates the flow field and droplet diffusion in the isolation ward. Before conducting
 219 transient simulations, a stable scenario is simulated without the coughing process as the initial value
 220 to assure the airflow stability in the room. The CFD simulation results in the isolation ward are
 221 plotted below. Fig. 5 shows streamlines formed by the isolation ward's ventilation system. As shown
 222 in Fig. 5(a), the airflow from the inlet impacts the flow field in the isolation room. After the airflow
 223 hits the floor, one part of the airflow moves up the wall to the ceiling and forms a backflow zone,
 224 while the other part moves toward the empty area and returns. And after the vertical air reaches the
 225 floor, the airflow hits the bed and produces a recirculation region. Fig. 5(b) indicates flow field on
 226 the cross-section B-B. The airflow in the zone is observed to move toward the outlet, and a portion
 227 of the air hits the furniture and forms an eddy current area. Fig. 5(c) shows flow field on the cross-
 228 section C-C. After the air is delivered to the floor, the split flow reaches the floor and forms a
 229 backflow.



230

231

Fig. 5 Flow fields on the cross-sections (a) A-A, (b) B-B, and (c) C-C.

232

An experiment is conducted to measure velocities of 10 points to verify the flow field simulation. The measuring apparatus (Hot ball anemometer WS-40, Hongrui, China) is selected for measurement, placed in different positions to measure the velocity. The location of the observing points is illustrated in Fig. 2. And parameters of the WS-40 are summarized in Table 3. The test data can be collected in real-time and output to the laptop, as shown in Fig. 6.

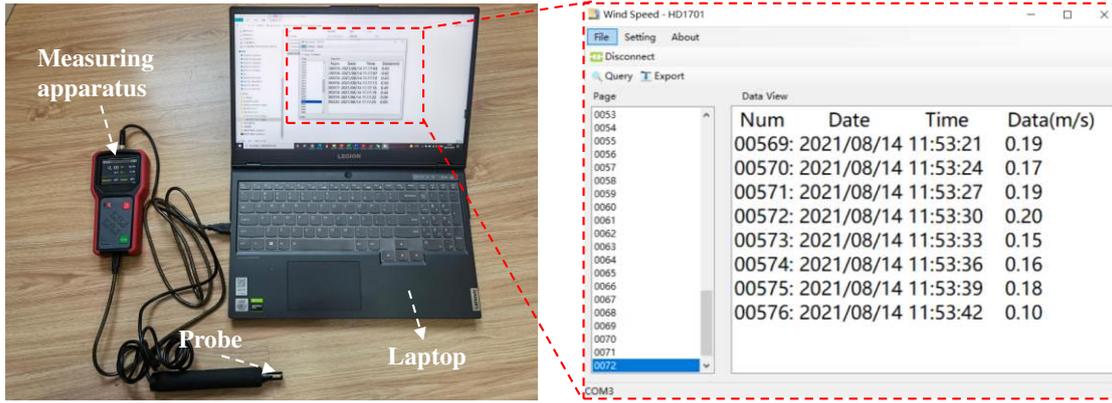
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During the experiment, the ventilation system is kept running. Furthermore, doors and windows are closed to separate the space from the outside world. In the process of the experiment, only the air-conditioning is turned on, resulting in no temperature change.

240

The velocity validation results at selected points are shown in Fig. 7. Results of the experiment and simulation are consistent. The mean absolute percentage error is less than 10%, indicating that the CFD simulation's boundary condition are suitable. Because the actual airflow is turbulent, using anemometers to measure velocity is difficult. In addition, in order to lower the computing cost, the simulation is simplified, which results in certain errors. But in this model, the errors between the simulation and the experiment are acceptable.

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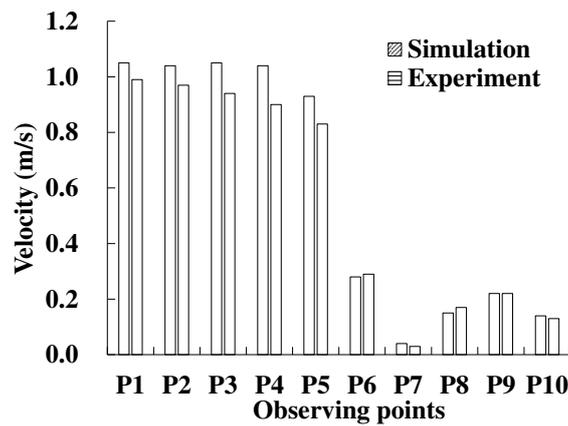
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Fig. 6 Real-time acquisition of experiment data.

248

Table 3. Parameters of hot-bulb anemometer

	Operating	Temperature	Measurement	Accuracy
	voltage	stability	range	
	3.7 V	0~50°C	0~2 m/s	0.01 m/s

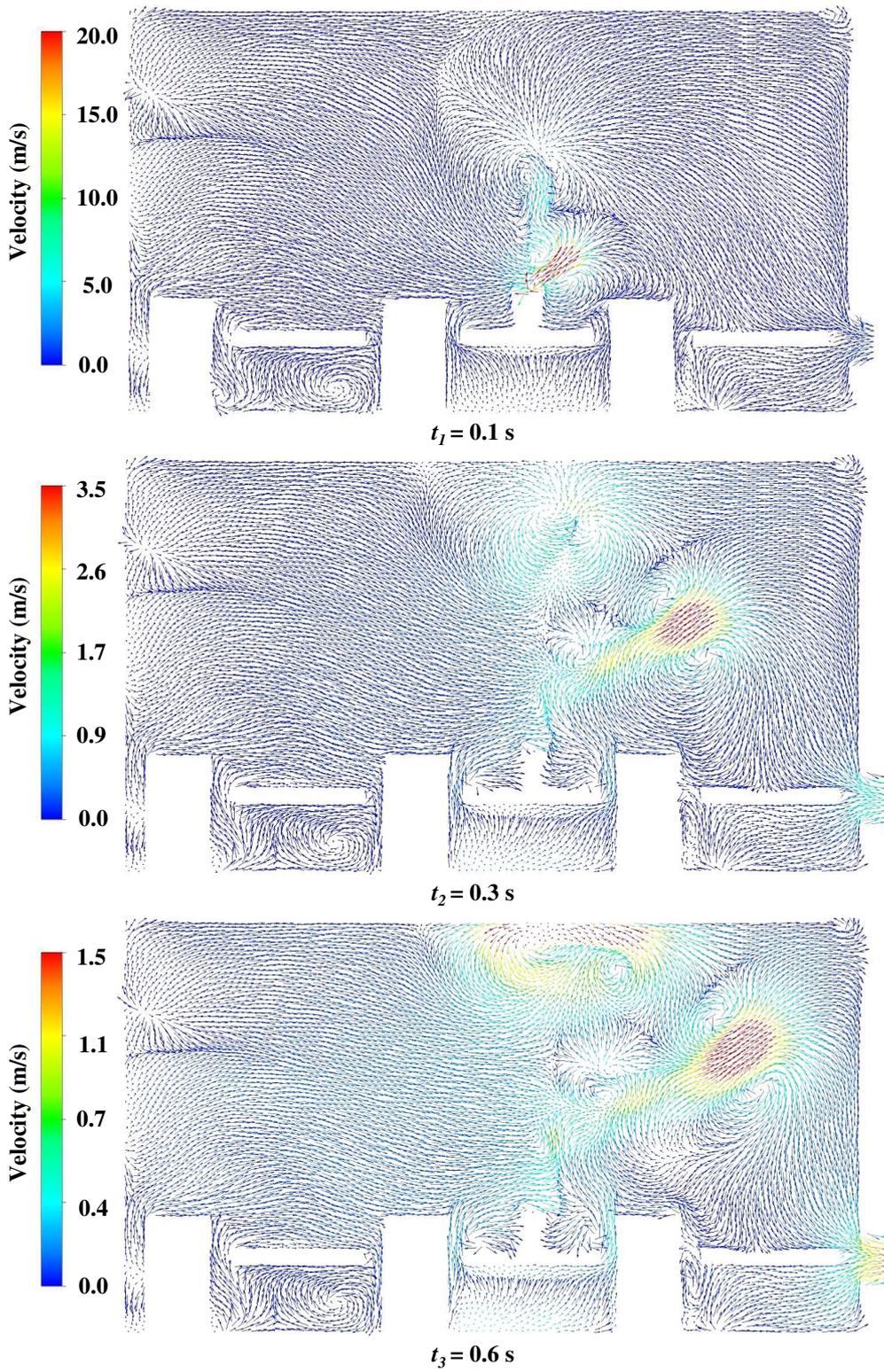


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Fig. 7 Velocity validation at selected points.

251 **Diffusion and distribution of Droplets**

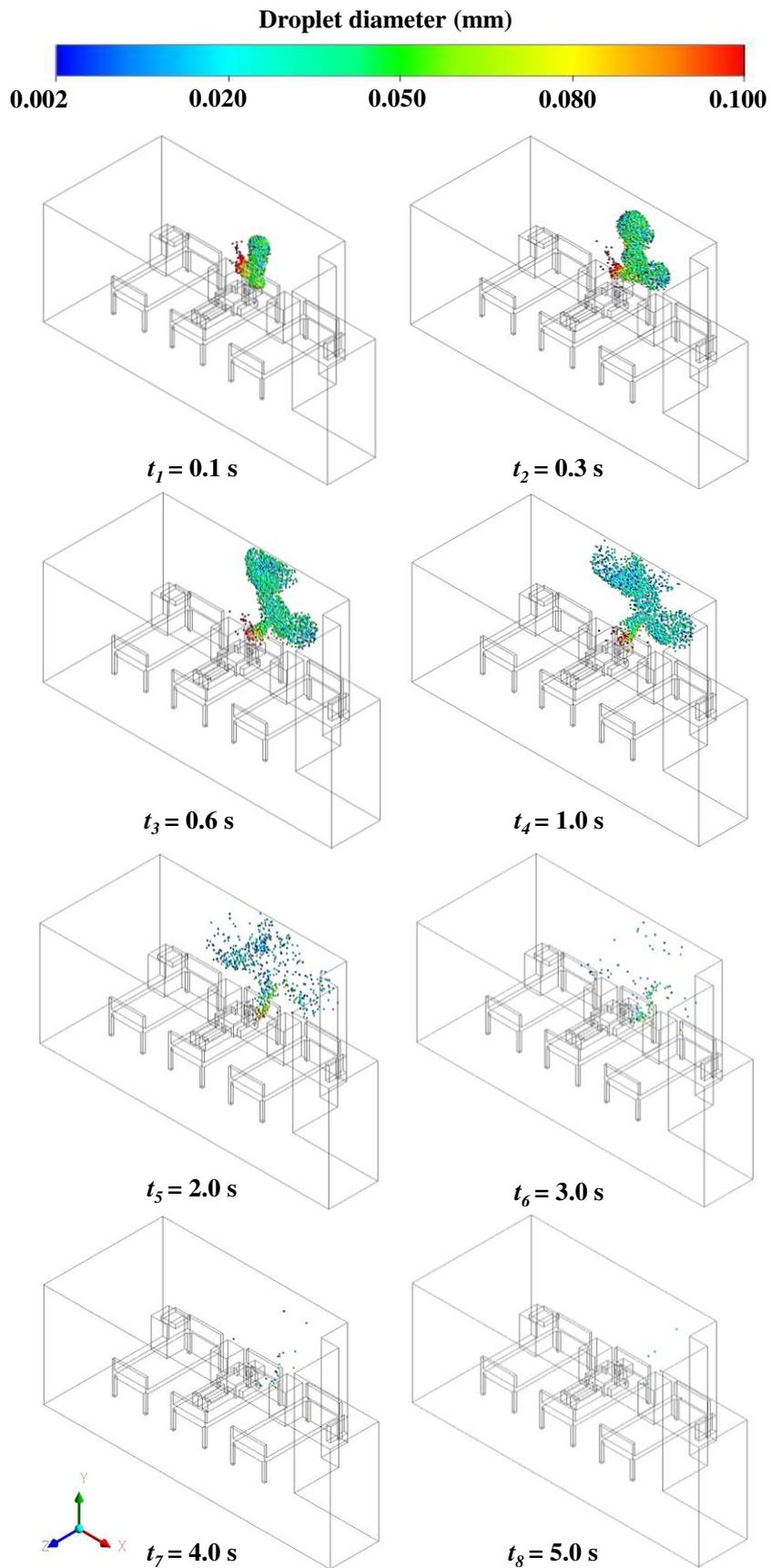
252 To investigate the droplets diffusion in the isolation ward, a steady scenario is simulated without the
 253 coughing process to ensure airflow stability in the room before releasing droplets. When the flow
 254 field in the room tends to be stabilized, droplets begin to be released (which means the patient starts
 255 coughing), and at this time $t = 0$ s. Fig. 8 shows the flow fields of one cough on the cross-section B-B
 256 in the original ward and Fig. 9 plots the temporal and spatial distributions of the droplets in the
 257 ward of Wuhan Pulmonary Hospital. At 0.1 s, a small portion is diffused to exhaust under the action
 258 of the flow field, which can be seen in Fig. 9; most droplets gather to form a droplet cloud. Droplets
 259 disperse gradually because of the vortex around the patient. These droplets hit the ceiling and
 260 continue to move along the wall, which can be observed at 0.3-2.0 s in Fig. 9. And flow fields on
 261 the cross-section B-B at 0.3 s and 0.6 s in Fig. 8 can explain the diffusion of droplets. Droplets begin
 262 to spread to the entire room at 3.0 s, affected by the eddy and buoyancy. Most droplets are eventually
 263 deposited on the patient surface and the walls, and a small portion can be observed to spread in the
 264 ward, which could carry virous and suspend indoors for a long time.



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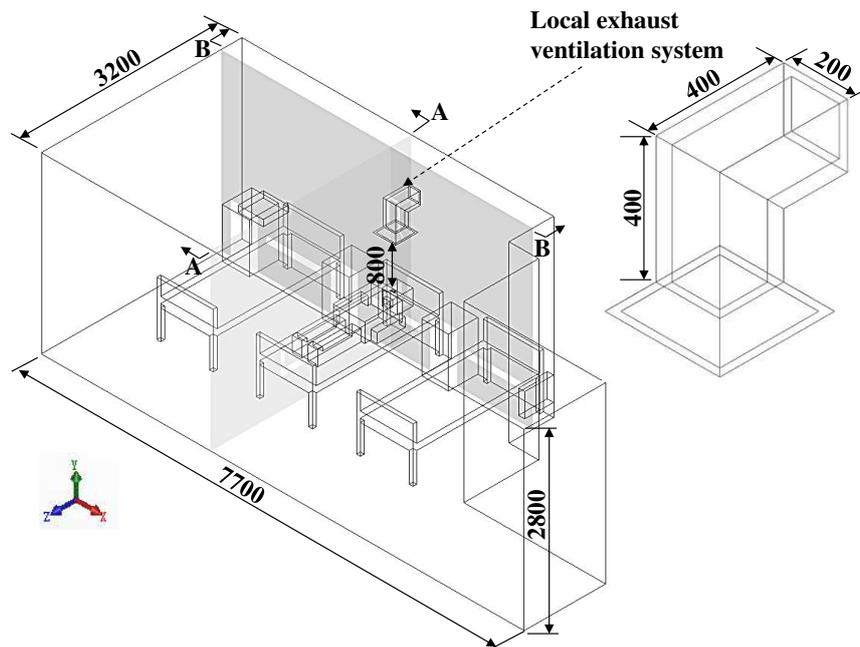
Fig. 8 Flow fields of one cough on the cross-section B-B in the original ward.



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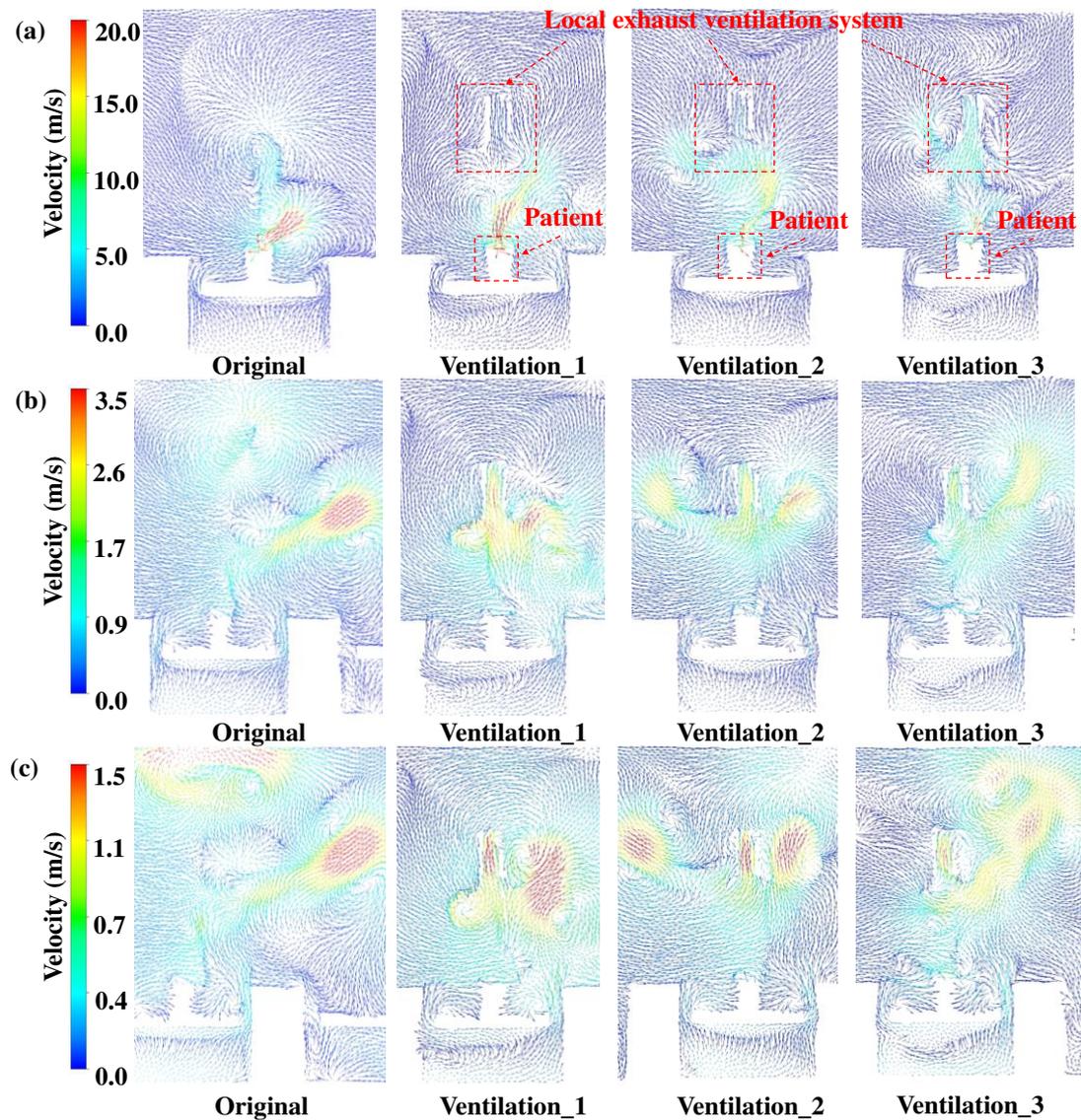
Fig. 9 Temporal and spatial distributions of droplets in the original ward.

270 As a fast-spreading infectious disease, the droplets containing the virus can remain in the room
 271 for a long time. The negative pressure ventilation systems isolate pathogenic droplets in hospitals,
 272 such as Wuhan Leishenshan Hospital (Luo et al. 2020). The layout of negative pressure wards can
 273 also minimize the exposure of patients and health care professionals to COVID-19 (Al-Benna 2021).
 274 According to the US Ventilation of Health Care Facilities, the minimum differential pressure
 275 between wards and relatively clean areas is 2.5 Pa (ANSI/ASHRAE/ASHE 2017). A local exhaust
 276 ventilation system is designed to ensure the air quality of the isolation ward, which is placed above
 277 the patient's head. The geometry of the ward furnished with the ventilation system and the
 278 dimension of the system is shown in Fig. 10.



279
 280 **Fig. 10** Schematic of the isolation ward (unit: mm) furnished with the local exhaust ventilation
 281 system.

282 The initial outlet negative pressure of the local exhaust ventilation system is set at 2.5 Pa in
 283 scenario ventilation_1, and the other boundary conditions for CFD simulation are summarized in
 284 Table 2. The purpose of the simulation is to analyze the effect of the local exhaust system on droplet
 285 diffusion. Fig. 11 shows the flow field on the cross-section B-B. Coughed stream of contaminated
 286 air spread droplets, affect by the local exhaust system at 0.1 s. The droplet swarm is divided into
 287 two streams affected by the local exhaust device and the air outlet. The local exhaust system
 288 enhances the turbulent transport throughout the ward, compared with the original ward.

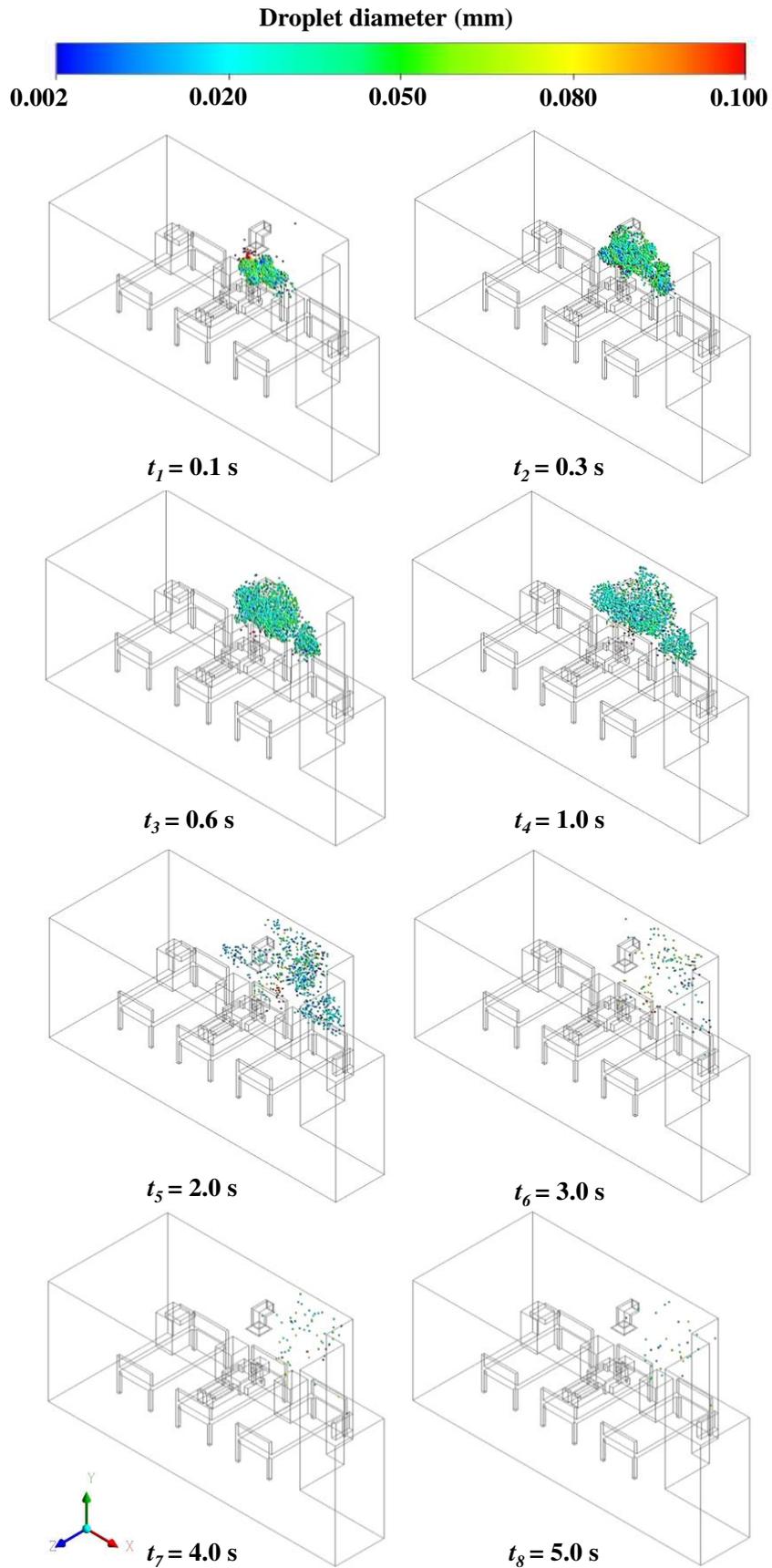


289

290 **Fig. 11** Flow field of one cough on the cross-sections B-B in four scenarios at (a) $t_1 = 0.1$ s, (b) $t_2 =$
 291 0.3 s, and (c) $t_3 = 0.6$ s.

292 The temporal and spatial distributions of droplets in the ward furnished a local exhaust ventilation
 293 system is plotted in Fig. 10. Interestingly, the existence of the local exhaust system promotes
 294 turbulent transport around the patient's head, leading to a complete breakup of the droplet cluster
 295 and enhancing the droplets' dispersion indoors. At 5.0 s, few droplets can be tracked moving in the
 296 ward, which means that most droplets are deposited or escape from the outlet.

297 In addition, we simulate droplet diffusion under different outlet negative pressures in scenario
 298 ventilation_2 and scenario ventilation_3. Fig. 13 and Fig. 14, respectively illustrates the temporal
 299 and spatial distributions of the droplets indoors. With an increase in the negative pressure, the local
 300 exhaust ventilation system can exhaust more droplets containing pathogenic microorganisms.
 301 However, the existence of the local exhaust ventilation system cannot remove all droplets
 302 thoroughly. Vortices can form in the area around the ventilation system, resulting in a portion of
 303 droplets contained in the room and moving with the indoor air.

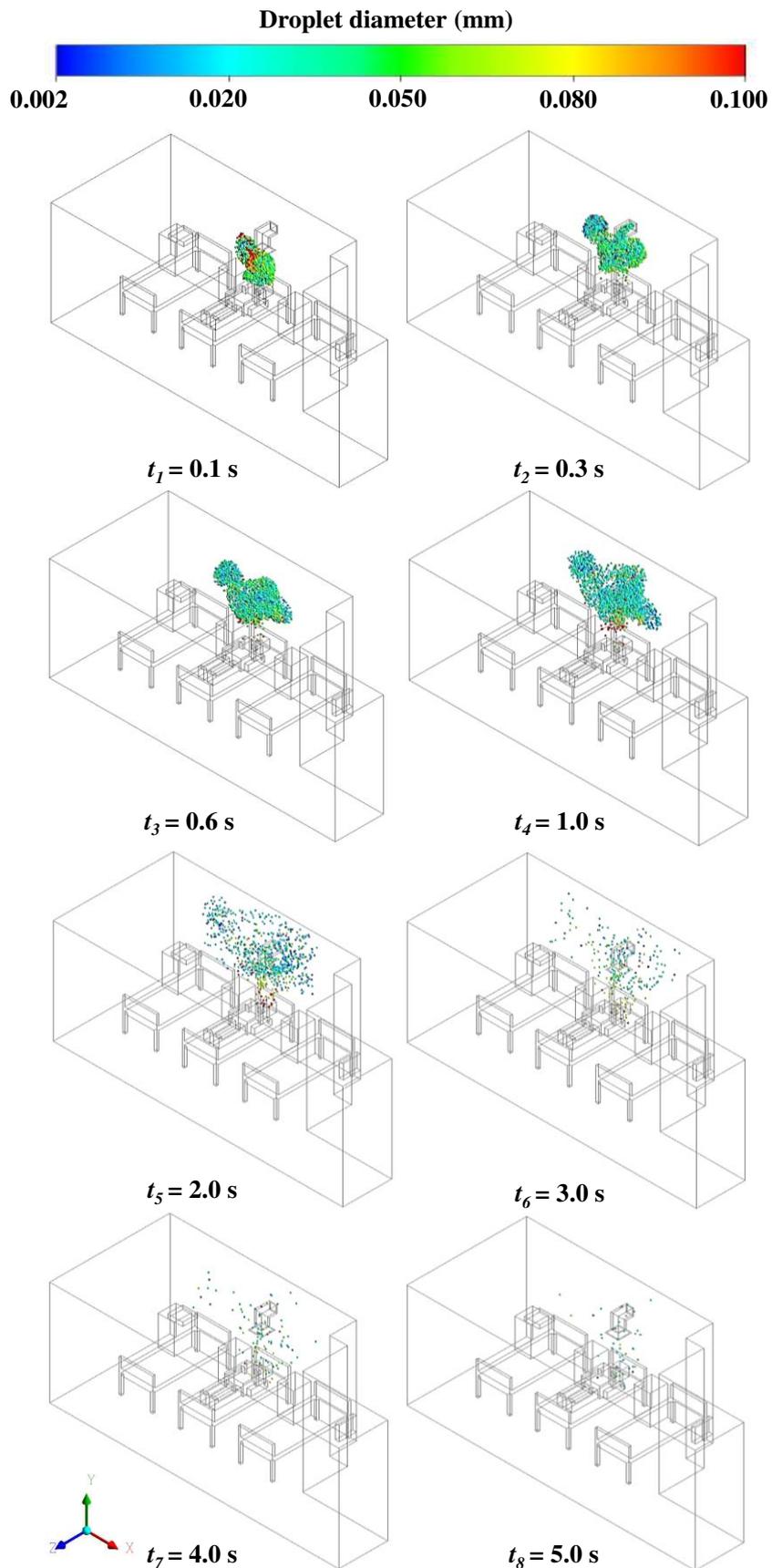


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Fig. 12 Temporal and spatial distributions of droplets with designed local exhaust ventilation system when outlet negative pressure is 2.5 Pa.

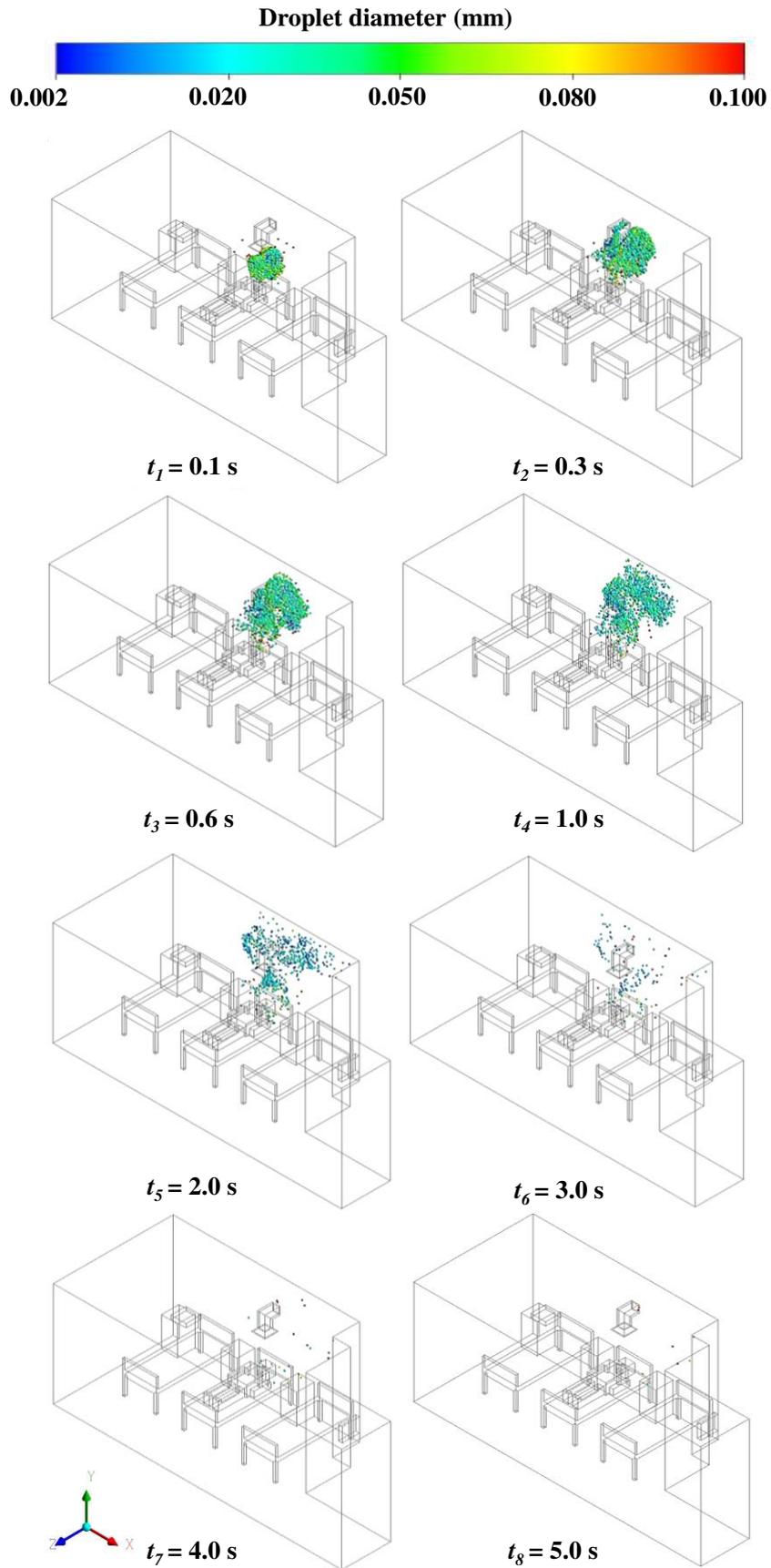


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Fig. 13 Temporal and spatial distributions of droplets with designed local exhaust ventilation system when outlet negative pressure is 3.0 Pa.



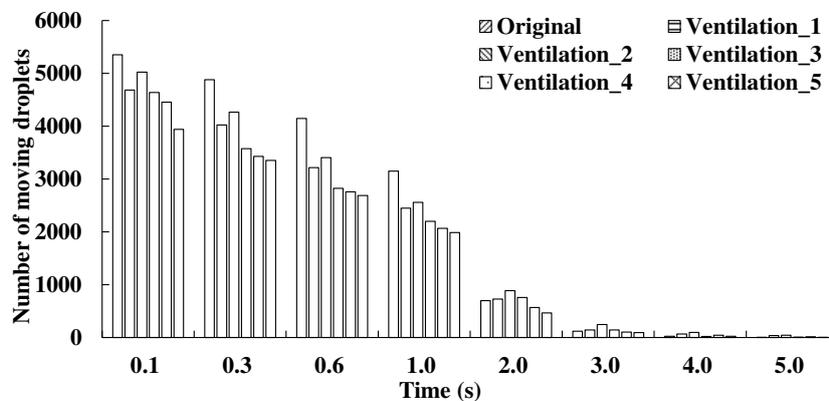
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311 **Fig. 14** Temporal and spatial distributions of droplets with designed local exhaust ventilation system

312 when outlet negative pressure is 3.5 Pa.

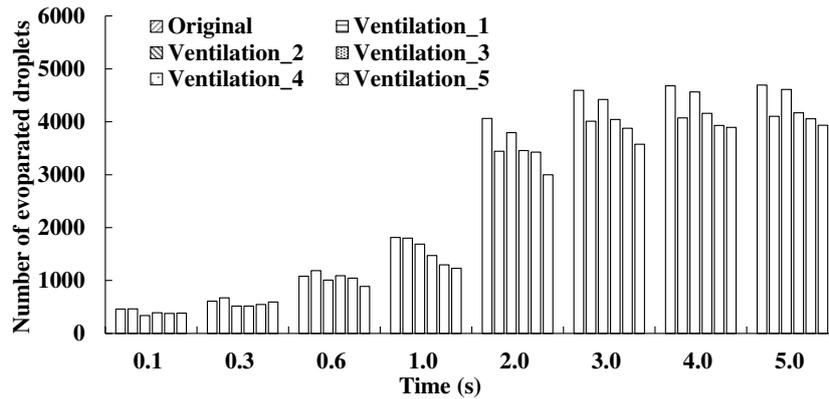
313 To obtain the influence of the outlet negative pressure on droplet diffusion, two more scenarios
 314 are added, in which the outlet negative pressure is 4.0 Pa (ventilation_4) and 4.5 Pa (ventilation_5),
 315 and we quantitatively analyze the droplet number at different moments in six scenarios. Results
 316 show that the local exhaust ventilation system can reduce the number of droplets, which can be seen
 317 in Fig. 15. When the negative outlet pressure is 2.5 Pa, the number of moving droplets in the ward
 318 reduces by nearly 15% compared with the original situation. Interestingly, as the negative outlet
 319 pressure rises from the initial value to 3.0 Pa, more droplets remain in the room than in the original
 320 ward. It can be explained that the existence of the local exhaust system promotes turbulent transport,
 321 leading to a complete disruption of the droplet cloud and enhancing the dispersion of droplets in the
 322 ward. As increasing the outlet negative pressure, we find that the number of droplets staying in the
 323 room gradually decreases. When the negative outlet pressure increases to 4.5 Pa, the local exhaust
 324 ventilation system can capture and exhaust more droplets than other scenarios. The number of
 325 droplets tracked in the ward reduces reduced by nearly 30% compared with the original ward. This
 326 is because the mass of the droplets is light, and when the negative outlet pressure value reaches a
 327 certain value, droplets caused by a coughing event can be removed more effectively. However, the
 328 effect of ventilation on patient comfort remains to be studied.

329 In addition to the droplets that spread with the air in the ward, a portion of droplets evaporates
 330 into water vapor according to Eq. (4-5). Fig. 16 illustrates the number of droplets evaporated in the
 331 ward over time. Over time, more droplets evaporate into water vapor and exist in the room in the
 332 form of water vapor, which is more conducive to the formation of aerosols. This allows pathogenic
 333 microorganisms to remain indoors for a long time, which increases the exposure of COVID-19 to
 334 healthcare staffs. The local exhaust system can minimize the number of droplets evaporated in the
 335 ward. But the formation of aerosols cannot be avoided.



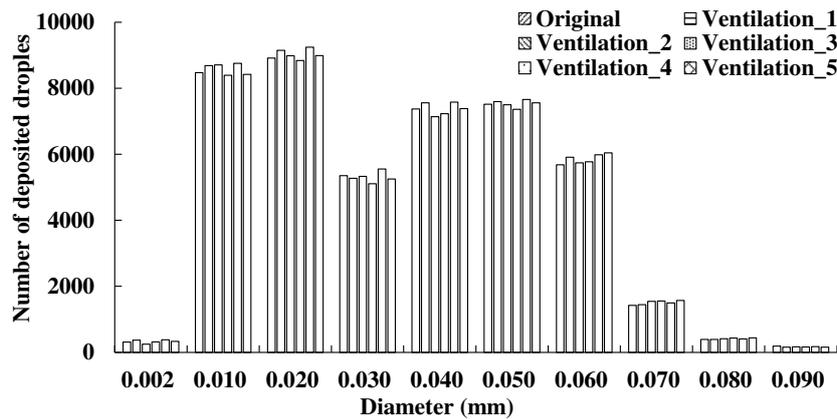
336
 337 **Fig. 15** Number of droplets moving in the ward over time.

338



339
340 **Fig. 16** Number of droplets evaporated in the ward over time.

341 As for the number of deposited droplets, there are 60.83%, 62.04%, 61.03%, 60.22%, 62.97%
342 and 61.52% of droplets produced by cough deposited on the area around the patient in six scenarios.
343 Fig. 17 illustrates the number of droplets deposited on the patient surface with different diameters.
344 The number of droplets with diameters of 0.01-0.03 mm accounts for nearly 38.1% of the deposited
345 droplets. According to the results, the local exhaust system has no obvious impact on the prevention
346 and control of surface contamination.



347
348 **Fig. 17** Number of droplets deposited on the surface of the patient in the ward with different
349 diameters.

350 **Conclusion**

351 In this study, the full-scale geometry of the isolation ward in Wuhan Pulmonary Hospital is
352 modeled and the droplet diffusion process from a coughing event in the ward is simulated. An
353 experiment is conducted to validate the CFD model, and a local exhaust ventilation system is
354 designed to ensure the air quality indoors.

355 (1) Although the local exhaust ventilation system reduces the number of airborne contaminants,
356 this also leads to a substantial increase in turbulent air motions. Results show that a negative outlet
357 pressure of 2.5 Pa and 3.5 Pa led to a decrease in evaporated droplets and airborne contaminants
358 remaining indoors, and the negative outlet pressure of 3.0 Pa increases the number of droplets
359 tracked in the ward of the excessively turbulent flow. It is found that the number of droplets
360 remaining indoors on the outlet negative pressure is not linear. The exhaust system can effectively

361 extract contaminants when the negative pressure value is small. But the negative pressure is
362 increased, the local exhaust system promotes turbulent transport, leading to a complete breakup of
363 the droplet cluster and enhancing the dispersion of droplets indoors. The mass of the droplets is light,
364 and when the outlet negative pressure value reaches a certain value, droplets can also be removed
365 effectively.

366 (2) The local exhaust system can minimize the number of droplets evaporated in the ward.
367 However, the formation of aerosols cannot be avoided.

368 (3) There are 60.83%, 62.04%, 61.03%, 60.22%, 62.97% and 61.52% droplets produced by cough
369 deposited on the patient surface in six scenarios. According to the results, the local exhaust system
370 has no obvious influence on the prevention and control of surface contamination.

371 Nevertheless, the simulation in the isolation ward proved that the local exhaust ventilation system
372 allows reducing the number of airborne contaminants remaining indoors. The compound of the
373 contaminant droplets is complicated to simulate in CFD, and the droplets are assumed to be
374 absorbed on the surface when droplets collide with the surfaces. The spread of infectious diseases
375 is complex. Therefore, it is difficult to simulate through the CFD method to restore the real process.
376 The simulation of the simplified model can only provide some reference to analysis on contaminant
377 diffusion in the room.

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384 Wang. Conceptualization, Supervision and Project administration were performed by Chengqing
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393 **Reference**

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