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Title page

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ORIGINAL ARTICLE(or REVIEW)

Multi-material Topology Optimization Based on Multiple SIMP of Variable Density Method

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Abstract: In this paper a multiple SIMP (solid isotropic material with penalization model) of variable density method is proposed to solve the problem of multi-material topology optimization. All candidate materials including void material are arranged in descending order of elastic modulus. The material conversion scheme of multiple SIMP is based on the elastic modulus of the candidate material after interpolation. The iterative criterion of multi-material topology optimization is derived from the Kuhn-Tucker condition using the guide-weight method. Three examples show that it is effective and moderate to use the proposed method to solve the problem of multi-material topology optimization.

Keywords: Topology optimization • Multi-material • Multiple SIMP • Variable density method

1 Introduction

As a kind of structural optimization, topology optimization is a mathematical method for optimizing the distribution of materials in a given design domain according to given load conditions, constraints and performance indicators [1-3]. In the past decades, a variety of topology optimization methods such as level set method [4-5], phase field method [6-7] bi-directional evolutionary structural optimization (BESO) method [8-9], and variable density method [10-11] has been extensively and deeply researched. These topology optimization methods gradually expanded from single-material to multi-materials [12-15].

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Guo et al. [16] proposed a level set-based approach to solve the stress-related topology optimization problem. The absence of gray areas in the design domain is the most attractive feature of this approach during the optimization process. The approach is extended to stress-related topology optimization for multi-materials [17]. In comparison with SIMP-based optimization approaches, the level set-based approach has lower solution efficiency. This means that it needs more iteration to get the optimal solution.

Tavakoli et al. [18] introduced a new algorithm to solve multi-materials topology optimization problem. The core idea of the algorithm is to divide the multi-material topology optimization problem into a sequence of binary phase topology optimization sub-problems. These sub-problems are solved in a sequential manner according to the traditional binary phase topology optimization problem. There are a large number of binary phase topology optimization sub-problems to be solved in the proposed algorithm. The computational cost of this algorithm increases dramatically. It usually takes over 1000 iterations to find an optimal solution.

To solve the multi-material topology optimization problem, Xia et al. [19] integrate the material conversion scheme of the BESO with the level set method. This method inherits the advantages of the BESO method and the level set method, and overcomes the shortcomings of the two methods.

Among the various popular interpolation schemes of variable density method, SIMP is a commonly used density-stiffness interpolation model. The interpolation model uses the element relative density as the design variable, and the material properties are simulated by the exponential function of the element relative density. The

elastic modulus in SIMP for single material [20] can be formulated as

$$E(x) = x^p E_0, \quad (1)$$

where $E(x)$ is the elastic modulus after interpolation; x is the element relatively density; p is penalization factor and E_0 is the elastic modulus of the material.

For the two-phase material problem, the SIMP model can be expressed as

$$E(x) = x^p E_1 + (1 - x^p) E_2, \quad (2)$$

where E_1 and E_2 are the elastic modulus of the two materials, respectively.

The three-phase material usually contains two material and void material. The SIMP model of three-phase material requires two design variables to describe. It can be further expressed as

$$E(x_1, x_2) = x_1^{p_1} (x_2^{p_2} E_1 + (1 - x_2^{p_2}) E_2), \quad (3)$$

According to this law, the SIMP model m -phase material requires $(m-1)$ design variables to describe. Blasques [21] applied the SIMP-like interpolation model to multi-phase material topology optimization problem of composite beam cross sections. It also requires $(m-1)$ design variables.

Zuo et al. [22] proposed an ordered SIMP interpolation method to solve multi-phase material topology optimization problem. This method interpolates the elastic modulus for multiple materials with respect to the normalized density variables by introducing power functions with scaling and translation coefficients. The biggest feature of this method is that there is no need to introduce any new design variables. Yin et al. [23] proposed a new material interpolation model based on SIMP method, which is named the peak function model. This model can easily include multi-phase materials in the design by using a linear combination of normal distribution functions. The outstanding feature of this model is also that there is no need to add any new design variables.

Either the previous method for multi-material topology optimization has more iteration or additional design variables need to be added. In this paper, under the premise of without increasing design variables, we aim to solve the problem of multi-material topology optimization.

2 Multiple Variable Density Method

The structure is composed of m candidate materials μ_j , with corresponding densities ρ_j , and elastic modulus $E_j (j=1,2,\dots,m)$. The candidate material μ_m is the void material. Let all the m candidate materials be arranged in

descending order of elastic modulus such that $E_1 > E_2 > \dots > E_m$. The structure is divided into n elements and the initial material of all elements is candidate material μ_1 .

In the multiple SIMP of variable density method, the material properties are still simulated by the exponential function of the element relatively density. The elastic modulus in multiple SIMP of variable density method can be formulated as

$$E(x_i^j) = (x_i^j)^p E_j, \quad (4)$$

where $E(x_i^j)$ is the elastic modulus after interpolation by the j th material μ_j . $x_i^j (i=1,2,\dots,n)$ is the relative density of the i th element whose material is the j th material μ_j . E_j is the elastic modulus of the j th material μ_j .

In the multiple SIMP of variable density method, the material of an element can change from one to another, as shown in Fig.1. When an element changes from the j th material μ_j to the $(j+1)$ th material μ_{j+1} , the elastic modulus after interpolation does not change. It is expressed as

$$E(x_i^j) = E(x_i^{j+1}), \quad (5)$$

We can get Eq. (6) from Eqs.(4) and (5).

$$(x_i^j)^p E_j = (x_i^{j+1})^p E_{j+1}, \quad (6)$$

The Eq. (7) is expressed in another form as

$$x_i^j = x_i^{j+1} p \sqrt[p]{\frac{E_{j+1}}{E_j}}, \quad (7)$$

The maximum value of x_i^{j+1} is 1. When $x_i^j \leq p \sqrt[p]{\frac{E_{j+1}}{E_j}}$,

the element changes from the j th material μ_j to the $(j+1)$ th material μ_{j+1} . At this time, the relative density of the i th element whose material is the $(j+1)$ th material μ_{j+1} is calculated as

$$x_i^{j+1} = x_i^j \sqrt[p]{\frac{E_j}{E_{j+1}}}, \quad (8)$$

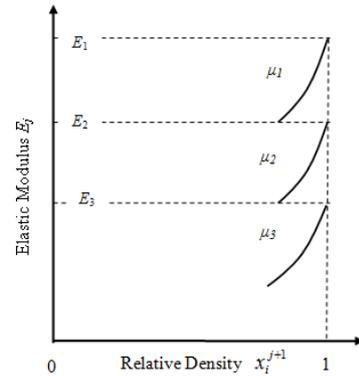


Figure 1 Elastic modulus in multiple SIMP of variable density method

3 Multi-material Topology Optimization using Multiple SIMP of Variable Density Method

3.1 The Mathematical Model

A matrix \bar{X} is defined to store the relative density of elements. The number of matrix rows is the number of elements n , and the number of matrix columns is the number of candidate materials m . It is expressed as

$$\bar{X} = (x_{1,1}, x_{1,2}, \dots, x_{i,j}) \in R^{n \times m}, \quad (9)$$

$$i = 1, 2, \dots, n \quad j = 1, 2, 3, \dots, m$$

An element can only be assigned one of m materials. It means that only one element in each row is (0, 1] and the other elements are equal to zero.

A column matrix I is defined, whose elements are equal to 1. It is expressed as

$$I = [1, 1, \dots, 1]_{m \times 1}, \quad (10)$$

We are setting a matrix X which is equal to \bar{X} times I . It is expressed as

$$X = \bar{X}I, \quad (11)$$

For each i

$$x_i^j = x_{i,j}, \quad (12)$$

The mathematical model of multi-material topology optimization under the single working condition can be expressed as

$$\begin{aligned} X &= (x_1^j, x_2^j, \dots, x_i^j) \in R^n \\ i &= 1, 2, \dots, n \quad j = 1, or2, or3, \dots, orm \\ f(X) &\rightarrow \min \\ s.t \quad g(X) &\leq 0 \\ x_{\min} &\leq x_i^j \leq x_{\max} \end{aligned} \quad (13)$$

where vector X is an n -dimensional vector of the design variables; $f(X)$ is the objective function and $g(X)$ is the constraint function. n is the number of finite element meshes and m is the number of candidate materials. x_{\min} and x_{\max} are the lower and upper limits of the design variable, respectively.

3.2 Guide-weight Method

In order to solve the multi-material topology optimization problem, we first need to construct the Lagrange equation as follows

$$L = f(X) + \lambda g(X), \quad (14)$$

where λ is a Lagrange multiplier.

Based on the Kuhn-Tucker condition, the optimal value X^* must be satisfied

$$\begin{cases} \frac{\partial L}{\partial x_i^j} = \frac{\partial f}{\partial x_i^j} + \lambda \frac{\partial g}{\partial x_i^j} \begin{cases} \leq 0 & x_i^j = x_{\max} \\ = 0 & x_{\min} < x_i^j < x_{\max} \\ \geq 0 & x_i^j = x_{\min} \end{cases} \\ \lambda \geq 0 \\ \lambda g(X) = 0 \begin{cases} \lambda = 0 & g(X) < 0 \\ \lambda \geq 0 & g(X) = 0 \end{cases} \\ g(X) \leq 0 \\ x_{\min} \leq x_i^j \leq x_{\max} \\ i = 1, 2, \dots, n \quad j = 1, or2, or3, \dots, orm \end{cases}, \quad (15)$$

When the value of x_i^j is between the upper and lower limits of the design variable, we can obtain the following equation as

$$\frac{\lambda \frac{\partial g}{\partial x_i^j}}{-\frac{\partial f}{\partial x_i^j}} = \frac{\lambda x_i^j \frac{\partial g}{\partial x_i^j}}{-x_i^j \frac{\partial f}{\partial x_i^j}} = 1 \quad (16)$$

We mainly discuss the problem of multi-material topology optimization under mass constraint in this paper. So $f(X)$ represent multi-material structural performance and $g(X)$ represent the total volume or mass of multi-material structure.

Using the guide-weight method [24-28], the four key formulas are defined as follows:

$$H_i = \frac{\partial g}{\partial x_i^j}, \quad (17)$$

$$W_i = x_i^j H_i, \quad (18)$$

$$G_i = -x_i^j \frac{\partial f}{\partial x_i^j}, \quad (19)$$

$$G = \sum_{i=1}^n G_i, \quad (20)$$

where H_i is the heap density of x_i^j . W_i is the equivalent mass of x_i^j . G_i is the guide-weight of x_i^j and G is the total guide-weight.

Substituting equations (17) and (19) into equation (16), then we have

$$x_i^j = \frac{G_i}{\lambda H_i} \quad i = 1, 2, \dots, n \quad j = 1, or2, or3, \dots, orm \quad (21)$$

The above formula is the iterative criterion which can solve the problem of multi-material topology optimization taking advantage of the guide-weight method. When the multi-material topology optimization is performed, the iterative criterion can be written as

$$(x_i^j)^{(k+1)} = \left(\frac{G_i}{\lambda H_i}\right)^{(k)} \quad i = 1, 2, \dots, n \quad (22)$$

A step factor α is introduced into the iterative criterion. The purpose is to ensure convergence of optimization results. Considering the lower and upper limits of the design variable, the iterative criterion is finally indicated as

$$\text{Case(1): If } (x_i^j)^{(k+1)} > p \sqrt{\frac{E_{j+1}}{E_j}}, \text{ then}$$

$$(x_{i,j})^{(k+1)} = \begin{cases} \alpha \left(\frac{G_i}{\lambda H_i}\right)^{(k)} + (1-\alpha)(x_i^j)^{(k)} & x_{\max} > (x_i^j)^{(k+1)} > p \sqrt{\frac{E_{j+1}}{E_j}} \\ x_{\max} & (x_i^j)^{(k+1)} \geq x_{\max} \end{cases} \quad (23)$$

In this case, the element relative density will change and the material of the elements does not change.

$$\text{Case(2): If } (x_i^j)^{(k+1)} \leq p \sqrt{\frac{E_{j+1}}{E_j}}, \text{ then}$$

$$(x_{i,j+1})^{(k+1)} = (x_i^j)^{(k)} p \sqrt{\frac{E_j}{E_{j+1}}} \quad j = 1, \text{ or } 2, \text{ or } 3, \dots, \text{ or } m-1 \quad (24)$$

$$(x_{i,j})^{(k)} = x_{\min} \quad (25)$$

In this case, the i th element changes from the j th material μ_j to the $(j+1)$ th material μ_{j+1} . The relative density of the i th element whose material is the $(j+1)$ th material $x_{i,j+1}$ is computed by Eq.(24). Because an element can only be assigned one of m materials. The relative density of the i th element in the j th $x_{i,j}$ is assigned a value of x_{\min} as Eq.(25).

3.3 Lagrange Multiplier λ

When the design variable X takes the optimal value X^* , the Lagrange multiplier λ can be obtained by means of

$$\lambda = \frac{G_i}{x_i^j H_i} = \quad i = 1, 2, \dots, n \quad (26)$$

Hence

$$\lambda = \frac{G_1}{W_1} = \frac{G_2}{W_2} = \dots = \frac{G_n}{W_n} = \frac{\sum_{i=1}^n G_i}{\sum_{i=1}^n W_i} \quad (27)$$

W_i is the equivalent mass of multi-material structure. When performing a multi-material topology optimization, Lagrange multiplier λ can be calculated as

$$\lambda = \frac{G}{W_0} \quad (28)$$

where W_0 is the mass or volume constraint of multi-material structure.

Using equations (22-25), and (28), multi-material

topology optimization using multiple SIMP of variable density method can be solved.

4 The Minimum Compliance Problem of Multi-material Topology Optimization

In this section, we mainly discuss the minimum compliance problem of multi-material topology optimization by virtue of the above proposed method. The element relative density is selected as the design variable. The structural mass is as the constraint function. It can be formulated as

$$f(\mathbf{X}) = C, \quad (29)$$

$$g(\mathbf{X}) = M - f_0 \cdot M_0 \leq 0, \quad (30)$$

where C is the compliance of multi-material structure. M_0 is the initial mass of multi-material structure. M is the mass of multi-material structure after optimization. f_0 is a mass fraction.

How to get the explicit expression between structural compliance and the design variable is the key to solving multi-material topology optimization. The structural compliance C can be expressed as

$$C = \mathbf{F}^T \mathbf{U} = \mathbf{U}^T \mathbf{K} \mathbf{U}$$

$$= \sum_{i=1}^n c_i = \sum_{i=1}^n \mathbf{u}_i^T \mathbf{k}_i \mathbf{u}_i, \quad (31)$$

where \mathbf{U} is the displacement vector. \mathbf{F} is the load vector. \mathbf{K} is the structural stiffness matrix. \mathbf{u}_i is the element displacement vector. \mathbf{k}_i is and the element stiffness matrix.

According to multiple SIMP of variable density method, we know that

$$\mathbf{k}_i = (x_i^j)^p \mathbf{k}_i^j, \quad (32)$$

where \mathbf{k}_i is the stiffness matrix of the i th element after penalization. \mathbf{k}_i^j is the initial stiffness matrix of the i th element assigned to the j th material μ_j .

The structural compliance C is rewritten based on multiple SIMP of variable density method. It can be formulated as

$$C = \sum_{i=1}^n (x_i^j)^p \mathbf{u}_i^T \mathbf{k}_i^j \mathbf{u}_i, \quad (33)$$

Derivation of structural compliance C with respect to design variables is obtained as

$$\frac{\partial C}{\partial x_i^j} = \mathbf{F}^T \frac{\partial \mathbf{U}}{\partial x_i^j} + \left(\frac{\partial \mathbf{F}}{\partial x_i^j} \right)^T \mathbf{U}, \quad (34)$$

The load vector \mathbf{F} has nothing to do with the relative density x_i^j , so the above equation is simplified as below.

$$\frac{\partial C}{\partial x_i^j} = \mathbf{F}^T \frac{\partial \mathbf{U}}{\partial x_i^j}, \quad (35)$$

From the equation $\mathbf{K}\mathbf{U}=\mathbf{F}$, the following equation is obtained as

$$\frac{\partial \mathbf{K}}{\partial x_i^j} \mathbf{U} + \mathbf{K} \frac{\partial \mathbf{U}}{\partial x_i^j} = \frac{\partial \mathbf{F}}{\partial x_i^j} = \mathbf{0}, \quad (36)$$

From equations (35) and (36), we have

$$\begin{aligned} \frac{\partial \mathbf{C}}{\partial x_i^j} &= -\mathbf{U}^T \frac{\partial \mathbf{k}}{\partial x_i^j} \mathbf{U} = -\sum_{i=1}^n \mathbf{u}_i^T \frac{\partial \mathbf{k}_i}{\partial x_i^j} \mathbf{u}_i, \quad (37) \\ &= -p(x_i^j)^{p-1} \mathbf{u}_i^T \mathbf{k}_i^j \mathbf{u}_i \end{aligned}$$

Let's substitute equations (37) and (31) into equation (19), and then sort to get the following equation.

$$\begin{aligned} G_i &= -x_i^j \frac{\partial f}{\partial x_i^j} = -x_i^j \frac{\partial \mathbf{C}}{\partial x_i^j}, \quad (38) \\ &= p(x_i^j)^p \mathbf{u}_i^T \mathbf{k}_i^j \mathbf{u}_i = p c_i \end{aligned}$$

The total guide-weight G can be calculated as

$$G = \sum_{i=1}^n G_i = p \sum_{i=1}^n c_i = p \mathbf{C}, \quad (39)$$

The mass of multi-material structure M is expressed as

$$M = \sum_{i=1}^n \sum_{j=1}^m x_i^j v_i \rho_j, \quad (40)$$

Derivation of the mass of multi-material structure M with respect to design variables x_i^j is obtained as [29]

$$\frac{\partial M}{\partial x_i^j} = v_i \rho_j, \quad (41)$$

Let's substitute equations (30) and (41) into equation (17), then we have

$$H_i = \frac{\partial g}{\partial x_i^j} = \frac{\partial M}{\partial x_i^j} = v_i \rho_j, \quad (42)$$

Equations (38) and (42) are plugged into equation (22), and then it can be written as

$$x_i^j = \frac{p c_i}{\lambda v_i \rho_j} \quad i=1, 2, \dots, n \quad j=1 \text{ or } 2 \text{ or } \dots \text{ or } m, \quad (43)$$

The above formula is the iterative criterion which can solve the multi-material topology optimization problem of minimum compliance taking advantage of the guide-weight method. When the multi-material topology optimization is performed, the iterative criterion can be written as

$$(x_i^j)^{(k+1)} = \left(\frac{p c_i}{\lambda v_i \rho_j} \right)^{(k)} \quad i=1, 2, \dots, n \quad j=1 \text{ or } 2 \text{ or } \dots \text{ or } m \quad (44)$$

Taking into account the convergence of optimization and the limits of design variables, the iterative equation is finally expressed as

$$\text{Case(1): If } (x_i^j)^{(k+1)} > p \sqrt{\frac{E_{j+1}}{E_j}}, \text{ then}$$

$$(x_{i,j})^{(k+1)} = \begin{cases} \alpha \left(\frac{p c_i}{\lambda v_i \rho_j} \right)^{(k)} + (1-\alpha) (x_i^j)^{(k)} & x_{\max} > (x_i^j)^{(k+1)} > p \sqrt{\frac{E_{j+1}}{E_j}} \\ x_{\max} & (x_i^j)^{(k+1)} \geq x_{\max} \end{cases} \quad (45)$$

Case(2): If $(x_i^j)^{(k+1)} \leq p \sqrt{\frac{E_{j+1}}{E_j}}$, then

$$(x_{i,j+1})^{(k+1)} = (x_i^j)^{(k)} p \sqrt{\frac{E_j}{E_{j+1}}} \quad j=1, \text{ or } 2, \text{ or } 3, \dots, \text{ or } m-1, \quad (46)$$

$$(x_{i,j})^{(k)} = x_{\min}, \quad (47)$$

Lagrange multiplier λ can be computed by

$$\lambda = \frac{G}{W_0} = \frac{p \mathbf{C}}{f M_0}, \quad (48)$$

According to equations (44-48), the minimum compliance problem of multi-material topology optimization can be solved by virtue of the above proposed method.

5 Convergence Criteria

The relative error τ of the two adjacent optimization results is less than the given convergence accuracy τ_{\max} for five times in succession, that is, 5 consecutive times satisfying equation (49), it is considered that the multi-material topology optimization has converged. The convergence criteria can be formulated as

$$\tau = \frac{\left| \sum_{k=1}^N (C^{(k)} - C^{(k-1)}) \right|}{\sum_{k=1}^N C^{(k-1)}} \leq \tau_{\max}, \quad (49)$$

6 Numerical Experiments and Discussions

6.1 Bridge Structure under Mass Constraint

The design domain of the bridge structure [30] is 200×100 rectangular, as shown in Fig.2. The design domain is divided into 200×100 elements during multi-material topology optimization analysis. The left lower endpoint of the design domain is fully constrained. In the vertical direction of the design domain, the right lower endpoint is constrained. The vertical load F_1 is 20 N, located on the midpoint A of lower boundary. The vertical load F_2 is 10 N, located on the $\frac{1}{4}$ and $\frac{3}{4}$ of lower boundary, respectively. The other necessary parameters corresponding to these test cases are listed in Table 1.

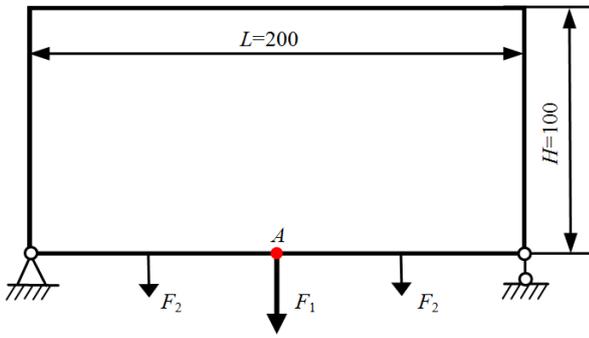


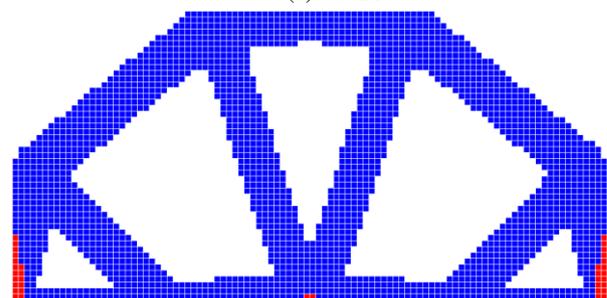
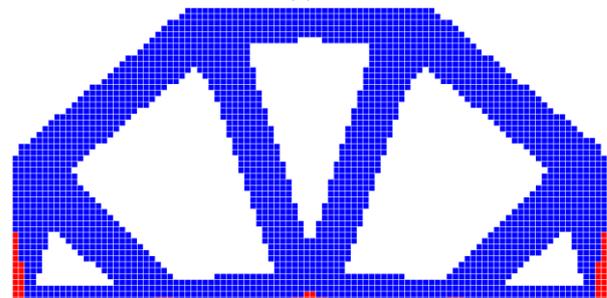
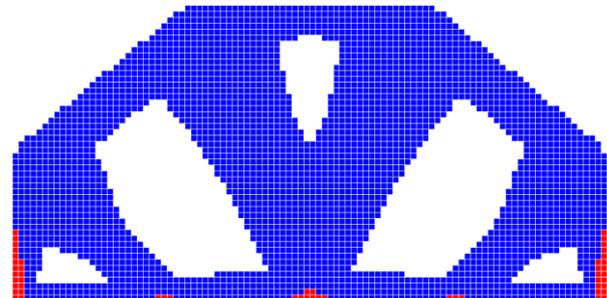
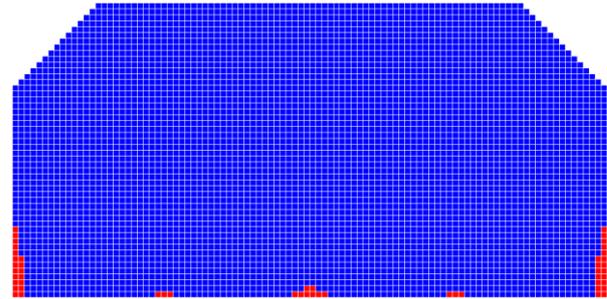
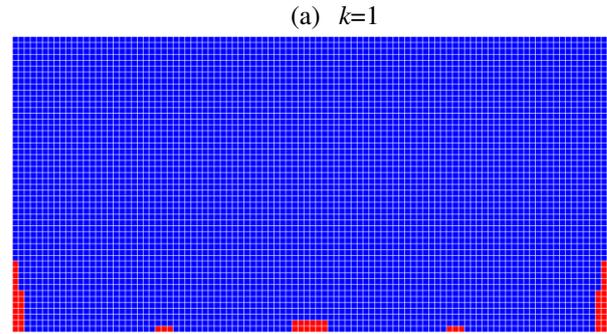
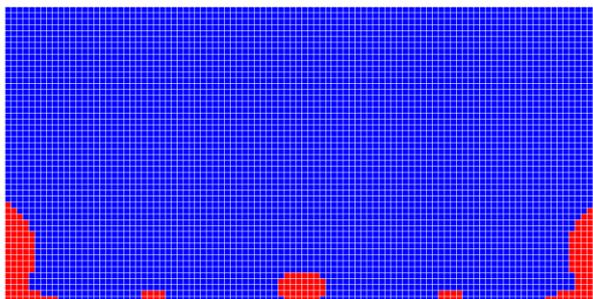
Figure 2 Optimization domain of bridge structure

Table 1 Parameters corresponding to bridge structure

Test Case	Candidate materials m	Penalization factor p	Elastic Modulus E_j	Density ρ_j	Mass Fraction f_0
Bridge #1	μ_1, μ_2, μ_3	3	[2,1,e-9]	[2,1,e-9]	0.2
Bridge #2	μ_1, μ_2, μ_3	3	[2,1,e-9]	[2,1,e-9]	0.25
Bridge #3	μ_1, μ_2, μ_3	3	[2,1,e-9]	[2,1,e-9]	0.3
Bridge #4	μ_1, μ_2, μ_3	3.5	[2,1,e-9]	[2,1,e-9]	0.25
Bridge #5	μ_1, μ_2, μ_3	4	[2,1,e-9]	[2,1,e-9]	0.25
Bridge #6	$\mu_1, \mu_2, \mu_3, \mu_4$	3	[4,2,1,e-9]	[4,2,1,e-9]	0.1
Bridge #7	$\mu_1, \mu_2, \mu_3, \mu_4$	3	[9,3,1,e-9]	[9,3,1,e-9]	0.05

6.1.1 Multi-material topology optimization of bridge structure

Figure 3 shows the evolution of multi-material topology optimization for bridge structure #1 with three kinds of materials including void material. It can be seen that most of the elements are converted from material μ_1 to material μ_2 in the first iteration. As the number of optimization increases, the amount of material μ_1 is getting less and less and the amount of material μ_2 is getting more and more. In the 11th iteration, some elements are changed from material μ_2 to material μ_3 (void material). After the 16th iteration, the amount of material μ_3 is getting more and more and the main body of optimal topology is forming. Until multi-material topology optimization is completed, an optimal topology with 3 materials is obtained, as shown in Figure 3(f). The main body of optimal topology is composed of material μ_2 and material μ_3 . Material μ_1 only appears in areas with high stress.



(f) $k=36$ (optimal topology)
 μ_1 μ_2 μ_3

Figure 3 Evolution of topology optimization for bridge structure #1

6.1.2 The effect of mass fraction f on optimization results

Figure 4 shows the optimization curves of structural compliance C for bridge structure #1, #2 and #3 when mass fraction f is 0.2, 0.25, and 0.3, respectively. As the number of optimization increases, three curves of structural compliance increase rapidly to a maximum value and then fall smooth. The optimal topologies are obtained after 36, 33, and 19 iterations, respectively, as shown in Fig.4. We find a law that the structural compliance and number of iterations when the optimal topology is obtained gradually becomes larger, as the mass fraction gets smaller. But the optimal topologies are similar, which can prove that the proposed method is effective and robust.

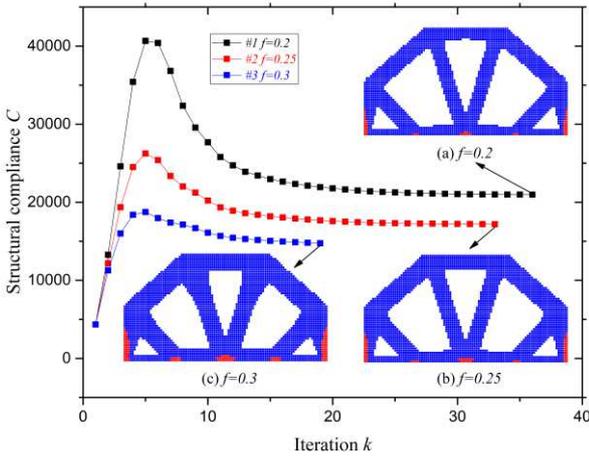


Figure 4 Optimization curves of structural compliance when mass fraction f is 0.2, 0.25, and 0.3

6.1.3 The effect of penalization factor p on optimization results

Figure 5 shows the optimization curves of structural compliance C for bridge structure #2, #4 and #5 when penalization factor p is 3, 3.5, and 4, respectively. The optimal topologies are obtained after 33, 18, and 15 iterations, respectively, as shown in Fig.5. We find the second law that the number of iterations gradually decrease as the penalty factor p increases. This means that increasing the penalty factor can get a lower number of iterations. Similar optimal topologies can be obtained when the penalty factors are different, which again proves that the method proposed in this paper is effective and moderate.

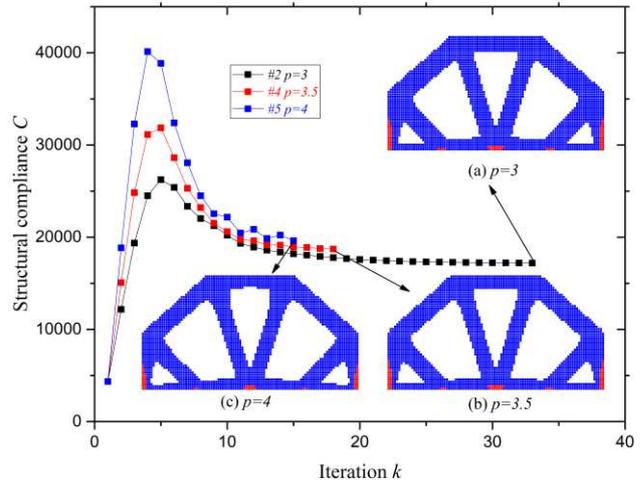
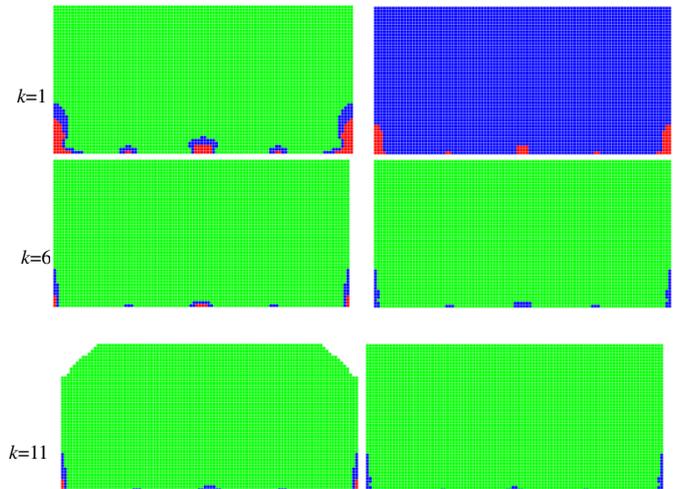


Figure 5 Optimization curves of structural compliance when penalization factor p is 3, 3.5, and 4

6.1.4 The effect of the number of candidate materials m on optimization results

Figure 6 shows the evolution of topology optimization for bridge structure #6 and #7 with four kinds of materials including void material. The main body of optimal topology with four kinds of materials is similar with the main body of optimal topology with three kinds of materials. The optimal topology for bridge structure #6 is composed of material μ_3 and material μ_4 . Materials μ_1 and μ_2 appears in areas with high stress. The optimal topology for bridge structure #7 also is composed of material μ_3 and material μ_4 . But material μ_2 only appears in areas with high stress and material μ_1 disappears. Because material μ_1 has a higher density, material μ_1 has been fully converted to material μ_2 under the condition of smaller mass fraction f .



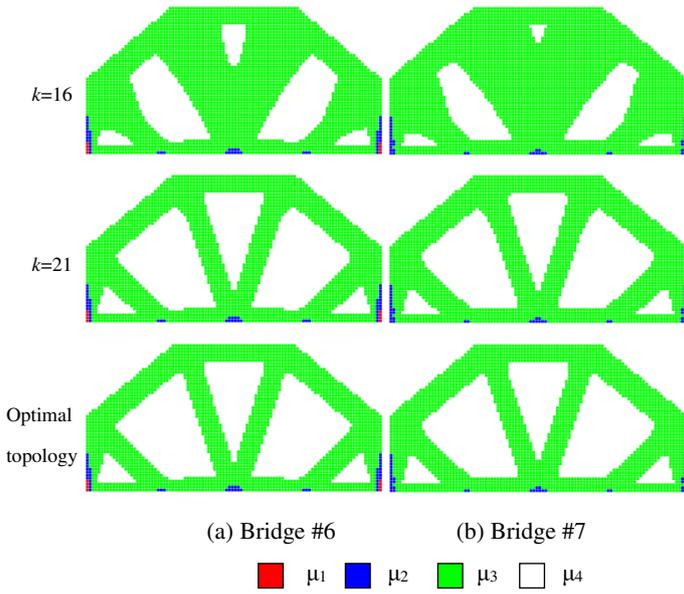


Figure 6 Evolution of topology optimization for bridge structure #6 and #7

6.2 Cantilever beam structure under the condition of volume constraint or mass constraint

The short cantilever beam structure is a rectangle with a length of 200 and a width of 100, as shown in Fig.7. The left boundary of design domain is fixed. A vertical load of $F_3 = 10\text{ N}$ is applied to the centre point B on the right boundary of the design domain. The short cantilever beam structure is divided into 200×100 finite element meshes. The other necessary parameters corresponding to these test cases are listed in Table 2.

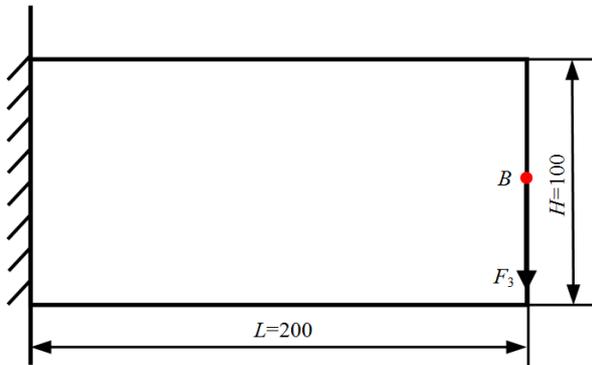


Figure 7 Optimization domain of cantilever beam structure

Table 2 Parameters corresponding to cantilever beam structure

Test Case	Candidate materials m	Penalization factor p	Elastic Modulus E_j	Density ρ_j	Mass Fraction f_0
Cantilever #8	μ_1, μ_2, μ_3	3	$[2, 1, e-9]$	$[2, 1, e-9]$	0.3
Cantilever #9	μ_1, μ_2, μ_3	3	$[2, 1, e-9]$	$[1, 1, 1]$	0.3

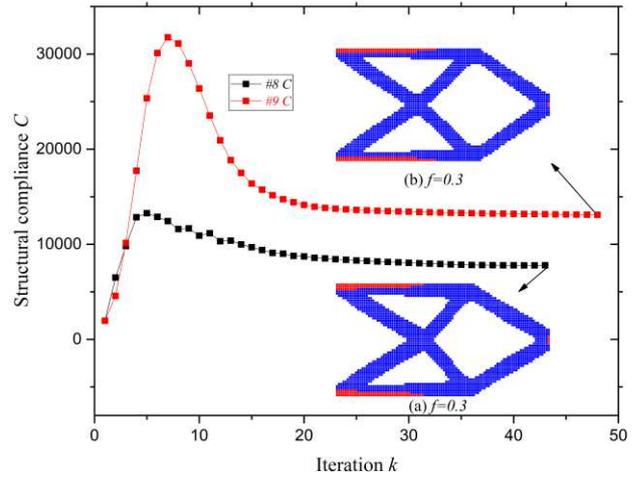


Figure 8 Process of topology optimization for cantilever beam

Figure 8 is the process of topology optimization for cantilever beam #8 and #9 with three kinds of materials including void material under mass or volume constraint. The optimal topologies are obtained after 43 and 48 iterations, respectively. We find that the main body of optimal topology under volume constraint is similar with optimal topology under mass constraint. Because the effect of density is not considered under volume constraint, the structural compliance under the condition of volume constraint is much greater than those under mass constraint.

6.3 Short MBB beam with real material

The short MBB beam structure is a rectangle with $200\text{mm} \times 100\text{mm}$, as shown in Fig.9. The left lower endpoint of the design domain is full constrained. In the vertical direction of the design domain, the right lower endpoint is constrained. The vertical load F_4 is 1000 N, located on the centre point D of lower boundary. The short MBB beam is divided into 200×100 finite element meshes.

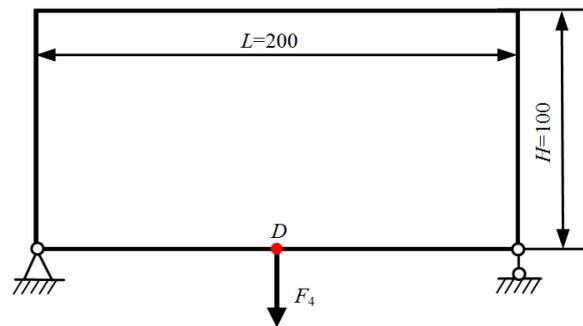


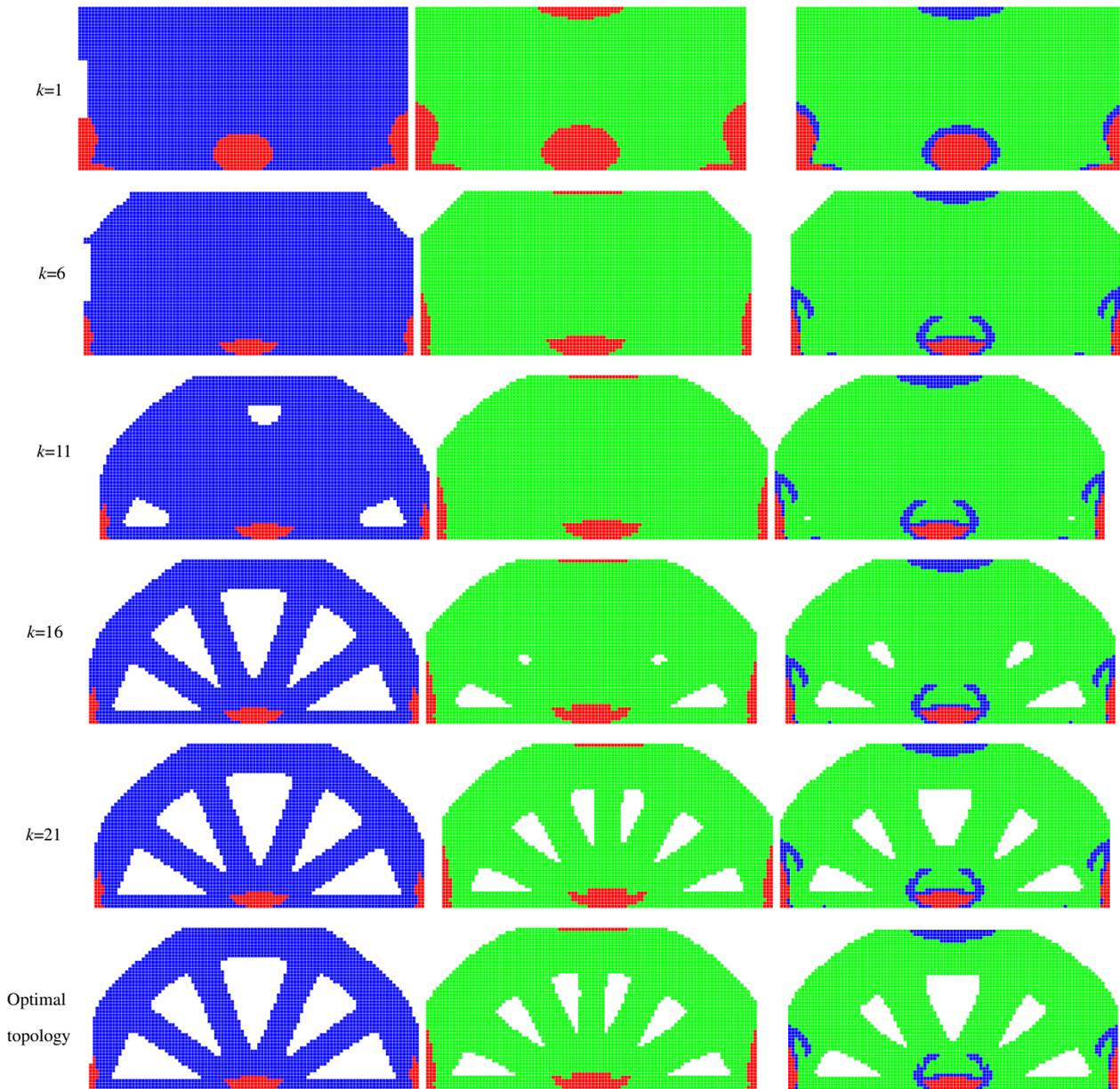
Figure 9 Optimization domain of short MBB beam structure

The topology optimization problem of short MBB beam

with mass constraint $f=0.3$ is illustrated here. We use four real materials including steel, aluminium, magnesium and void material. The elastic modulus of steel, aluminium, magnesium and void material are $E_S=210\text{GPa}$, $E_A=70\text{GPa}$, $E_M=40\text{GPa}$ and $E_V=10^{-6}E_S$, respectively. The density of steel, aluminium, magnesium and void material are $\rho_S=7890\text{kg/m}^3$, $\rho_A=2630\text{kg/m}^3$, $\rho_M=1740\text{kg/m}^3$ and $\rho_V=10^{-6}\rho_S$, respectively. Poisson's ratio $\nu=0.3$.

Figure 10 shows the evolution of multi-material topology optimization for MBB structure with 4 real materials including void material. The three different material combinations are: (A) Steel, aluminium and void; (B) Steel, magnesium and void; (C) Steel, aluminium, magnesium

and void. Through a certain number of optimization calculations, we obtain the optimal topologies of the three material combinations. The main body of the three optimal topologies are similar. Steel is mainly distributed in high stress areas. Low stress area is occupied by aluminium and magnesium. But the details of the three optimal topologies are different. Comparison with the optimal topology of combination A, steel appears in the upper middle region of the optimal topology of combination B. In the optimal topology of combination C aluminium replaces magnesium in the upper middle region. At the same time, aluminium also appeared around the steel.



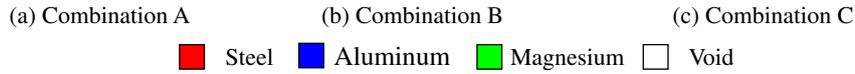


Figure 10 Evolution of multi-material topology optimization for MBB beam

When the topology optimization converges, the values of the mass fraction are different. Due to the different mass fractions, the minimum compliance problems of three material combinations cannot be reasonably compared.

The product of structural compliance C and mass fraction f when the convergence conditions are met are used as the unified performance index PI . It can be formulated as

$$PI = C \cdot f \quad (50)$$

The dimension of the performance index PI is the same as the dimension of the objective function. The performance index PI can be called the objective function including constraint conditions. In this article it is referred to herein as structural compliance including mass fraction. The smaller the performance index PI , the smaller the objective function including constraint conditions. Therefore, it is of practical significance to use the performance index PI to evaluate the optimal topology of the three material combinations.

When the optimal topology is obtained, the parameters of the three material combinations are shown in the Table 3. After 39 iterations, the combination C obtained the optimal topology. At this time, the structural compliance is 137.80mm and the mass fraction f is 19.69%. The performance index PI is 27.13N.mm, which is the smallest of the three material combinations. So it can be considered that the optimal topology of combination C is the optimal topology of multi-material topology optimization.

Table 3 Parameters when the optimal topology is obtained

Material combinations	k	$C(N.mm)$	$f(\%)$	$PI(N.mm)$
Combination A	33	132.93	22.15	29.45
Combination B	28	139.01	20.78	28.88
Combination C	39	137.80	19.69	27.13

7 Conclusions

The innovation of this paper is to transform the multi-material topology optimization problem into multiple single-material topology optimization problems. The material conversion scheme is based on the elastic modulus of the candidate material after interpolation. Compared with the single material SIMP method, the proposed method also has the advantages of fast convergence and less iteration.

This paper is the first time that puts forward to solve the

problem of multi-material topology optimization based on multiple SIMP of variable density method. It can provide a new thought for researchers to study multi-material topology optimization in the future. At present, the proposed method can only achieve one-way conversion of candidate materials in descending order of elastic modulus. The next step is to incorporate the idea of BESO method into my research. The purpose is to achieve bidirectional transformation of candidate materials.

8 Declaration

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Availability of data and materials

The datasets supporting the conclusions of this article are included within the article.

Authors' contributions

The author' contributions are as follows: Hong-yu Jiao wrote the manuscript; Ying Li wrote the program using ANSYS APDL language and was responsible for proofreading.

Competing interests

The authors declare no competing financial interests.

Consent for publication

Not applicable

Ethics approval and consent to participate

Not applicable

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Appendix

Appendix and supplement both mean material added at the end of a book. An appendix gives useful additional information, but even without it the rest of the book is complete: In the appendix are forty detailed charts. A supplement, bound in the book or published separately, is given for comparison, as an enhancement, to provide corrections, to present later information, and the like: A yearly supplement is issue.

Figures

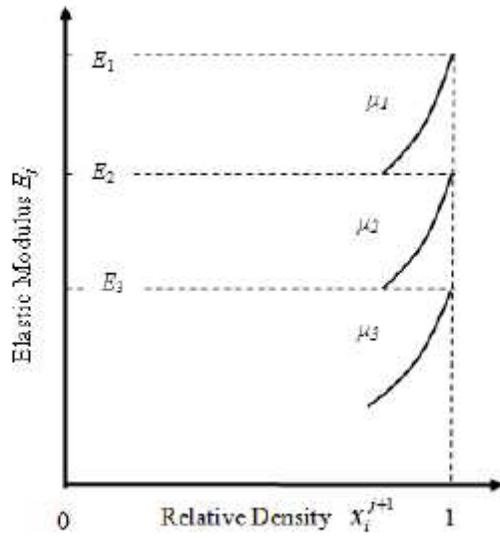


Figure 1

Elastic modulus in multiple SIMP of variable density method

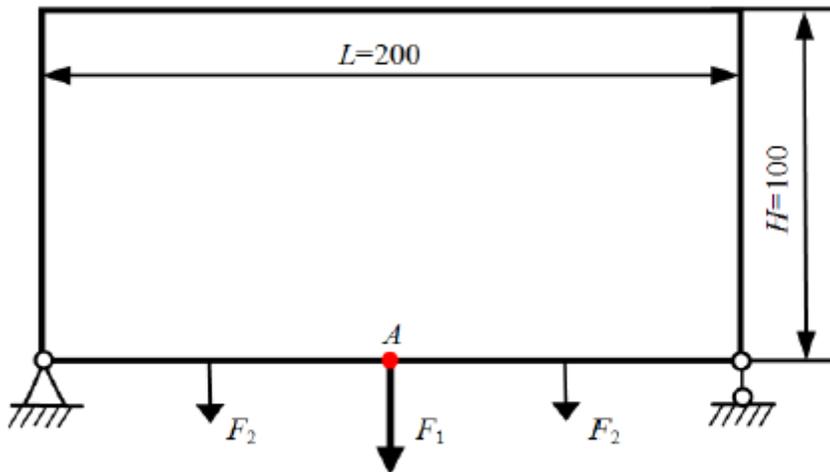
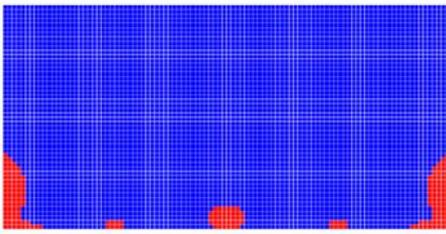
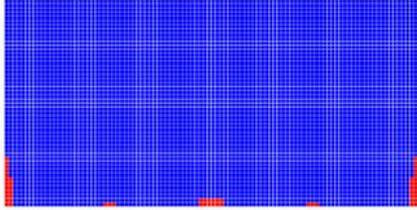


Figure 2

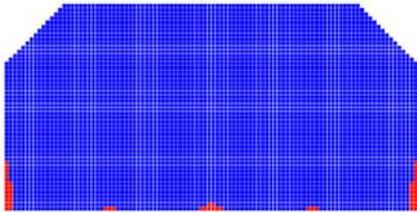
Optimization domain of bridge structure



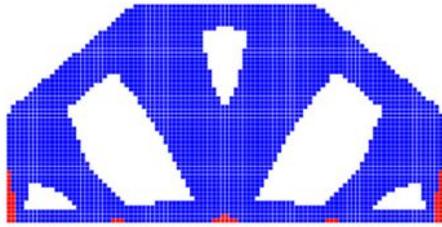
(a) $k=1$



(b) $k=6$



(c) $k=11$



(d) $k=16$



(e) $k=21$



(f) $k=36$ (optimal topology)

μ_1  μ_2  μ_3 

Figure 3

Evolution of topology optimization for bridge structure #1

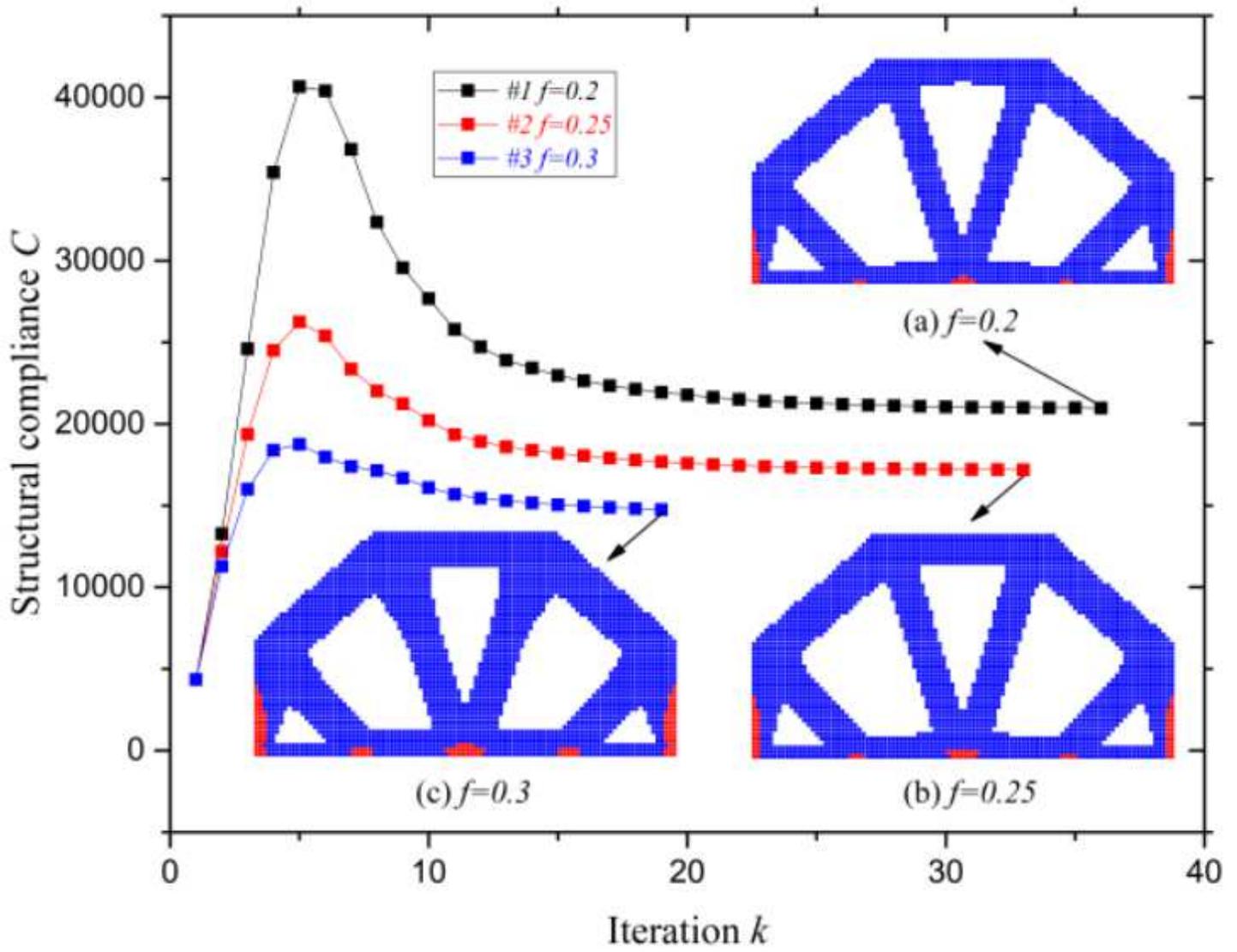


Figure 4

Optimization curves of structural compliance when mass fraction f is 0.2, 0.25, and 0.3

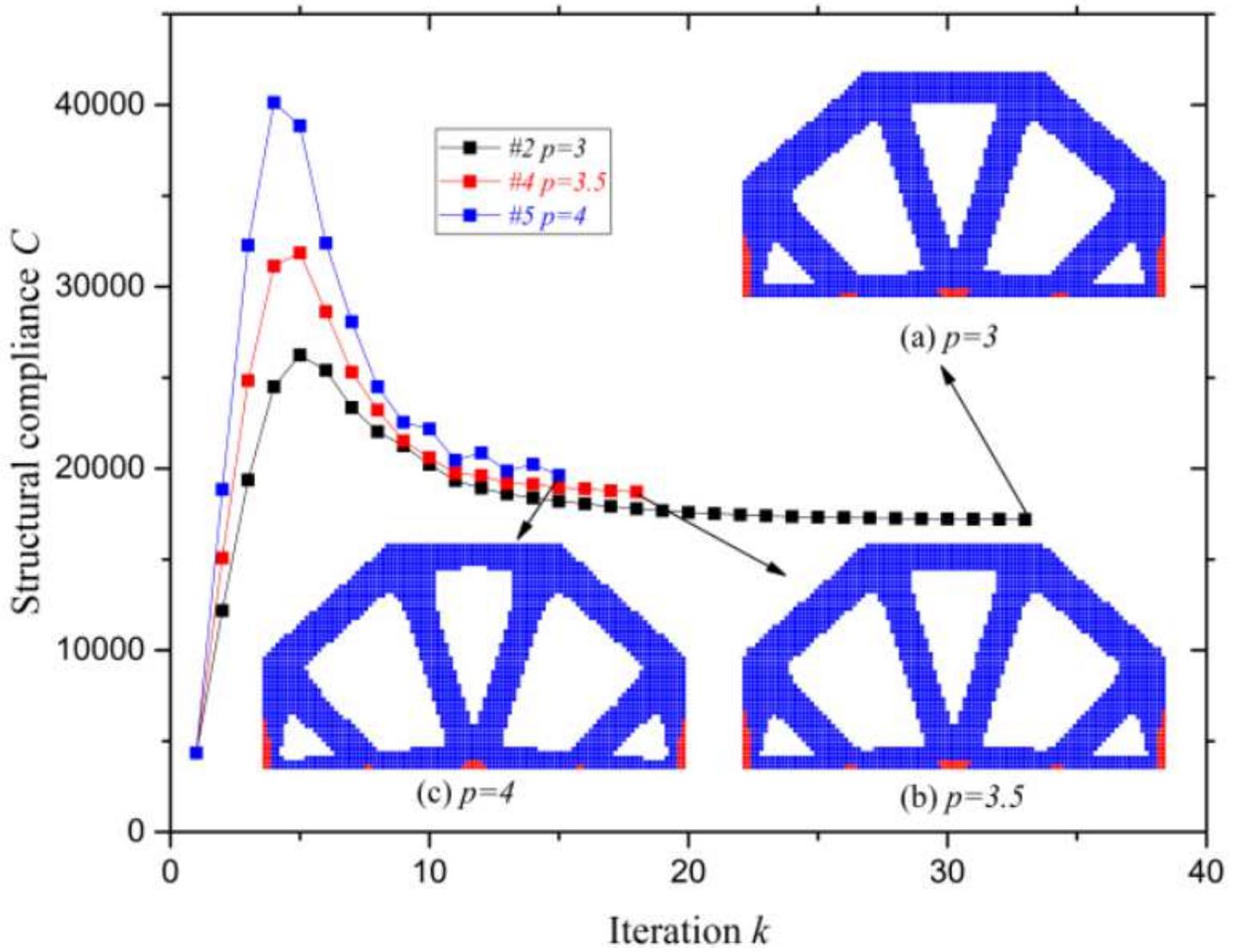


Figure 5

Optimization curves of structural compliance when penalization factor p is 3, 3.5, and 4

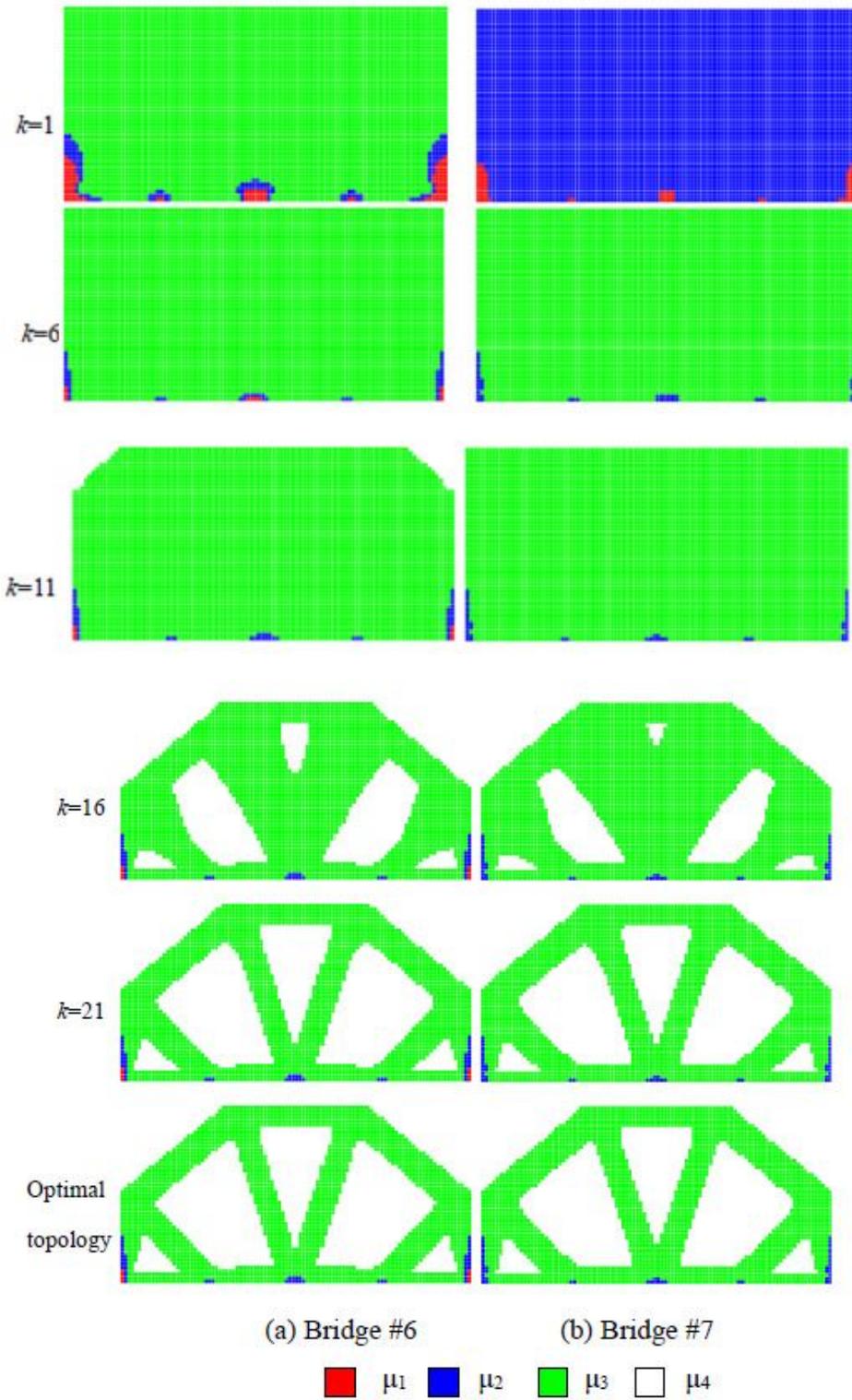


Figure 6

Evolution of topology optimization for bridge structure #6 and #7

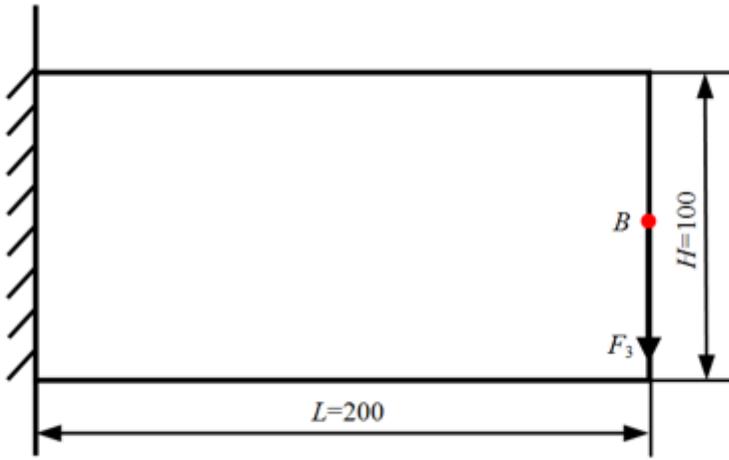


Figure 7

Optimization domain of cantilever beam structure

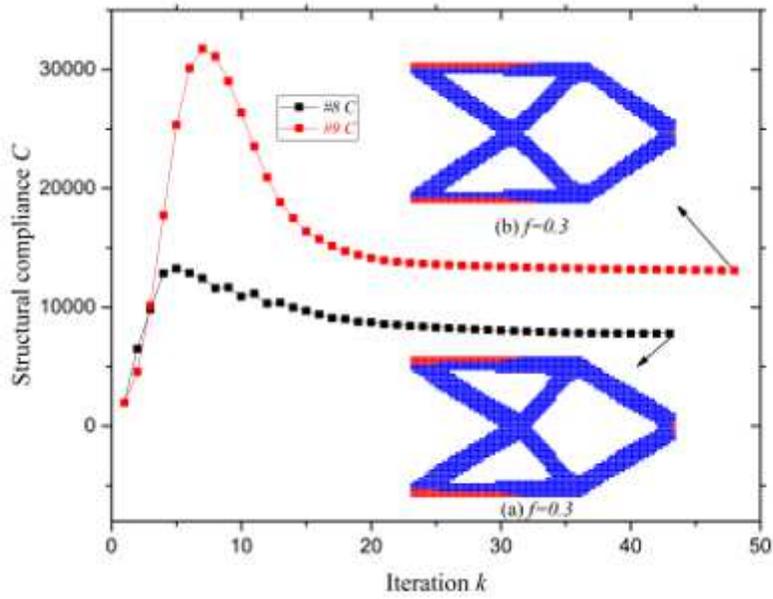


Figure 8

Process of topology optimization for cantilever beam

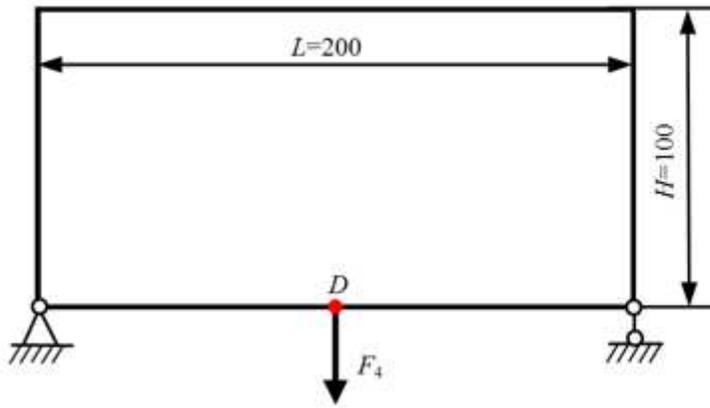


Figure 9

Optimization domain of short MBB beam structure

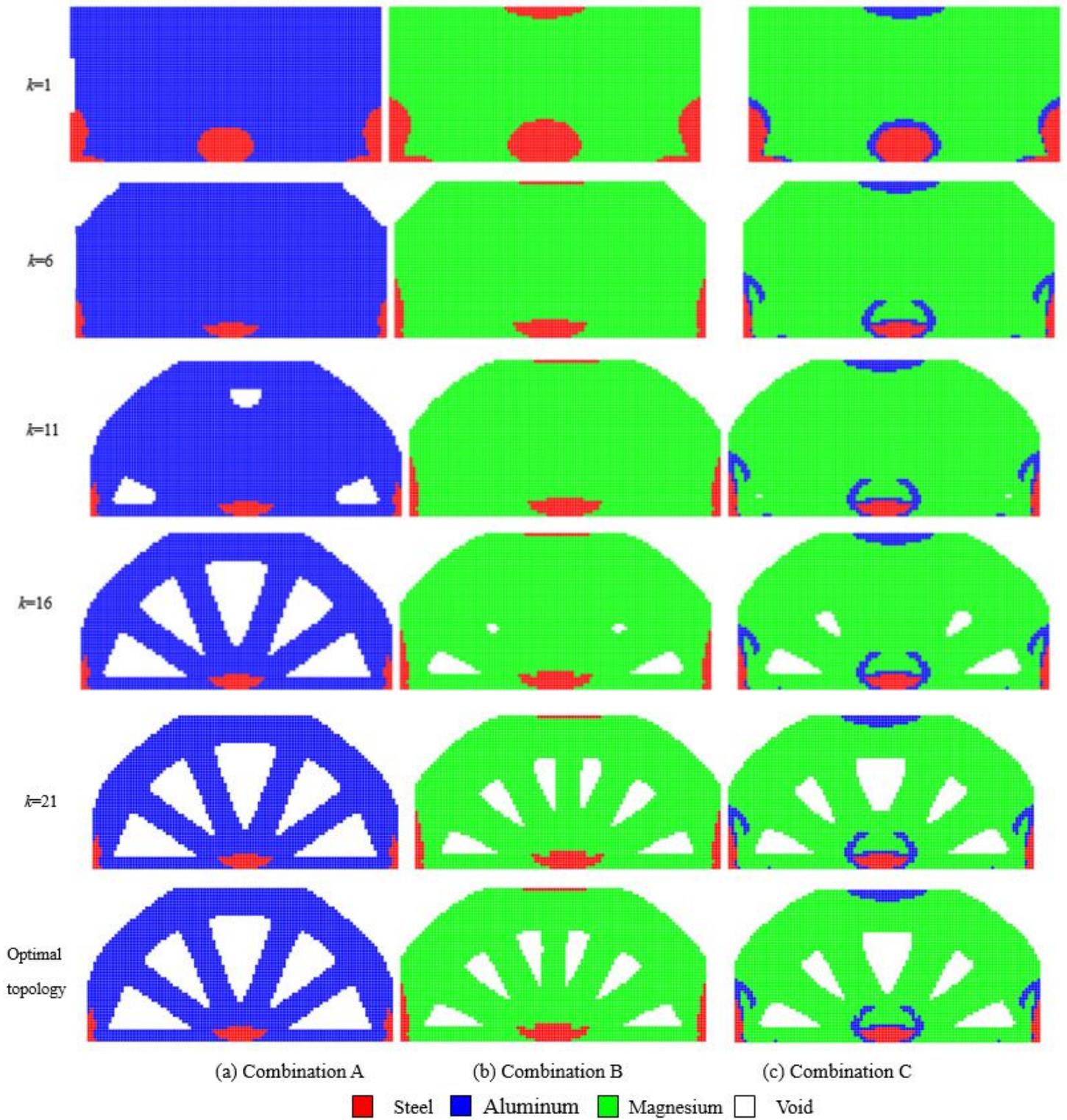


Figure 10

Evolution of multi-material topology optimization for MBB beam