

A novel approach to modelling the bond characteristics between CFRP fabrics and steel plate joints under quasi-static tensile loads

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Research Article

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Abstract

Carbon fiber reinforced polymer (CFRP) materials have been effectively used as externally bonded sheets to repair damaged steel structures such as airplanes and ships. In this study, a series of double strap joints with different bonding lengths are considered and examined to experimentally and theoretically assess the effective bond length. Various models exist in the literature which are used to predict the strength of steel and CFRP joints under various loading conditions. Non-linear Lagrange stress method (NLS) which is a novel stress-based method for predicting the failure load values is presented for the first time. This approach is based on 2D and 3D linear elastic finite element analysis. Relying only on two experimental tests, the new approach proposed here can quickly and easily predict the failure load in steel/CFRP samples. In this methodology, it is assumed that the adhesive joint will fail as the normal stress along the adhesive mid-line reaches a predetermined value at a critical distance. In addition, experimental data on steel/CFRP joints gathered from the literature are compared to predictions using the NLS method. It was found that results from the theoretical predictions (NLS) were in good agreement with experimental tests conducted on double strap joints. It was also revealed that the average accuracy of the NLS method is superior to other methods such as cohesive zone model and Hart-Smith. The results revealed that under the best conditions, the NLS model is 5 times more accurate than existing models.

1.0 Introduction

Polymeric composite materials (PMC) show superior properties such as high specific strength and stiffness making them versatile materials with various applications in military, civilian aircrafts, space, and automobile industries. PMCs can also be used to improve the life expectancy and load carrying capacities of compromised steel components. For example, carbon fiber reinforced polymer (CFRP) laminates can be applied in retrofitting of steel structures instead of conventional mechanical fasteners due to their light weight and low-cost advantages. Recent studies have shown that the use of CFRP offers an excellent alternative method for repairing deteriorated structures [1–7]. The main application of CFRP laminates is to make structures lighter while maintaining their integrity and strength [8]. For instance, 43% of metal structures in the Boeing 787 aircraft have been replaced by CFRP laminates [9]. This highlights the importance of a suitable failure predictive approach to design steel/composite bonded joints [10, 11]. The discontinuity of reinforcing fibers at joint interfaces can only be overcome by either using adhesive joints or rivet joints to form structural joints in many applications [12–16]. Traditional mechanical fastening approaches cannot be used with PMCs because of the introduction of local stress concentrations in and around the joints [17].

The failure mechanisms in steel/CFRP adhesive bonded joints directly affects the operational life of the components. There is a hand full of various criteria in the literature used in predicting the failure load of adhesively bonded joints. The majority of the available failure criteria are based on stress, strain or energy condition in the bond layer [18–25]. Typically, shear stress and normal stress (also known as peel stress) values are considered as key parameters in failure assessment of steel/CFRP adhesive bonded

Critical energy as a common failure approach was utilized for double strap joints (DSJ) by some researchers [26, 27]. Hashin's variational method has been improved by Chalkley and Rose [28] and Barroso et al. [29] went even further by including the stress singularity effects to evaluate 4 failures in double lap joints. In a similar study, Lee et al. used an experimental approach to obtain the joint strength and the failure modes in steel and GFRP bonded DSJs [30]. A number of experimentally driven research has focused on investigating the strength of CFRP and steel DSJs. The influence of several parameters on the joint strength has been studied in these experimentally driven approaches (e.g. [31–39]). The bond strength and fatigue crack propagation between steel/FRP sheets has been studied by Zhao et al. [40]. Mohee et al. [41] have recently reviewed various aspects of CFRP joints including strength, failure modes, performance and design parameters. The strength of the bond between steel coupons and CFRP strips was studied using a series of DSJs by Fawzia et al. [42]. The researchers modified the Hart-smith model and predicted the failure load in steel/CFRP DSJs and obtained reasonable results when compared to experimental data.

Until now, many approaches have been proposed to analyze the behavior of steel components strengthened with CFRP patches [43–47], including nonlinear theory [48], digital image correlation (DIC) [49–52], extended finite element method (XFEM) [53, 54] and cohesive zone model (CZM) [55–58].

Adhesive joining of composite-to-metal structures is faced with numerous challenges including the potential for debonding at the adhesive/substrate interface, difficulty in processing, such as achieving a uniform bond line thickness as well as complex surface preparation and tooling, and susceptibility of the joint to creep [52]. As a result, several studies have been performed to investigate the failure behavior of adhesive joints [53–58]. Extensive research has been conducted on the bond behavior between patching systems and steel under static loading [59–62], fatigue loading [63], large deformation cyclic loading [64] and environmental effect [65, 66].

Many steel structures deteriorate over time due to environmental corrosion, creep, and changes in their use and hence, need to be strengthened to resist the new loads to which they are subjected to. Static tensile loading is the most common type of load on structures such as steel bridges and buildings [60]. The bond between the CFRP composite and steel is a key issue in CFRP strengthening of aging steel structures; the use of an appropriate type of adhesive results in an acceptable bond.

The Lagrange method of optimization is a well-established mathematical solution algorithm technique that is used for solving constrained optimization problems. Most optimization problems consist of a nonlinear objective function and one or more linear or nonlinear constraint equations. In this method, the constraints as multiples of a Lagrange multiplier, are subtracted from the objective function until the optimal objective function is obtained [75].

In this paper the bond characteristics between CFRP laminate and steel members under quasi-static loading are investigated experimentally and theoretically in a series of DSJs. The presented criterion is based on normal (peel) stress along the midplane of the adhesive layer. In order to validate the presented Loading [MathJax]/jax/output/CommonHTML/fonts/TeX/fontdata.js with six series of experimental data reported in

the literature [67, 68] on DSJs. The aim of this research paper is to predict the failure loads of experimentally tested joints using a newly proposed failure predictive model, namely the non-linear Lagrange stress (NLS) method.

2.0 Materials And Experimental Procedure

A schematic representation of the geometry of the test samples used in this research are presented in Fig. 1. DSJs made from grade 300 steel sheets and a single ply of unidirectional normal modulus CFRP sheet on each side of the joints were fabricated with different bonding lengths (Fig. 2). The CFRP sheets were placed with the fibers aligned in the direction of the tensile load. The normal modulus of CFRP was used in this study. Two different size of steel sections (120mmx30mm and 160mmx40mm) were used to investigate the effect of steel section size on the bond properties. UHU Endfest 300 was used as adhesive to bond the CFRP to the steel members. The selection of the adhesive was done based on recommendations from industrial practitioners. The mechanical properties of the CFRPs required in the finite element simulations were obtained from the manufacturer's data sheet. UHU® Plus Endfest 300 adhesive (UHU, Buehl, Germany) with a mixing ratio of binder to hardener equal to 2:1 was used for bonding the components [69]. The length, width and thickness of the steel sheets were $L_{sub} = 160$ mm, $W = 40$ mm, $t_{sub} = 3$ mm, and $L_{sub} = 120$ mm, $W = 40$ mm, $t_{sub} = 3$ mm, respectively. Each CFRP layer was 0.176 mm in thickness, while the adhesive was 0.5 mm thick. Separate tensile tests were conducted on the steel plates (ASTM E8) [71] and adhesive material (ASTM D638) [70, 72] in order to obtain the corresponding mechanical properties under tensile loading (displayed in Table 1). Bulk specimens were prepared from adhesives, and then the stress–strain ($\sigma - \epsilon$) behaviors of the adhesive was determined from bulk dumb-bell shaped specimens tested under the conditions specified. The bonding length L_1 was always kept less than L_2 ($L_2 = L_1 + 30$ mm) to ensure that the failure only occurred on the L_1 side of the steel sheets (Fig. 1).

The steel sheets were grit blasted to remove any traces of dust, paint, oil and any other contaminants in the bond region ensuring a reliable contact between the adhesive and steel. The bond region was then wiped with acetone before the adhesive was applied to ensure a chemically active surface was obtained. The DSJs were cured for 45 min at 70°C after the application of the adhesive glue. The coupons were then post-cured for 7 days at room temperature (at 25°C).

Table 1
Mechanical properties of materials used for experiments.

Properties	Steel plates	Adhesive	CFRP sheets *
Tensile modulus (GPa)	203	2.1	200
Poisson's ratio	0.25	0.35	0.28
Ultimate strength (MPa)	520	39	1900
*Manufacturer's data			

Different bonding lengths of $L_1=10, 20, 30, 40, 50, 60$ mm were considered for experiments. At least three test coupons were tested for each mentioned bond length. A universal tensile testing machine (Instron ElectroPuls™ E10000, Norwood, MA, United States) was used for the tensile experiments under quasi-static loading. The load–displacement curves were obtained with a constant strain rate of 2 mm/min. The average value of the three tests was used in all of the numerical calculations.

Two steel sections were used as A=120x40 mm and B=160x40 mm to investigate the effect of steel section size on the bond properties. The steel/CFRP DSJs with various bonding lengths before and after the tensile tests are shown in Figure 2.

The details of each experiment (i.e., the dimensions of the DSJs and the corresponding failure loads) is presented in Table 2. As is shown in Table 2, three experimental tests, namely P_1 , P_2 and P_3 are carried out for each bonding length to have better accuracy. The average failure load for each set of tests is reported as P_{avg} .

Table 2: Details of experimental failure loads for the tested double strap joints (DSJs).

Specimen label	L_1 (mm)	L_2 (mm)	P_1 (kN)	P_2 (kN)	P_3 (kN)	P_{avg} (kN)
A	10	40	10.5	9.7	9.9	10.03
A	20	50	14.1	14.3	14.7	14.36
A	30	60	14.8	14.6	13.8	14.4
A	40	70	14.5	14.3	13.7	14.16
A	50	80	13.9	14.7	15.1	14.56
A	60	90	14.6	14.9	15.2	14.9
B	10	40	12.2	12.9	13.2	12.8
B	20	50	17.8	18.1	18.9	18.3
B	30	60	20.1	20.5	21.2	20.6
B	40	70	21.8	20.9	20.5	21.1
B	50	80	22.0	21.2	20.9	21.4
B	60	90	20.5	21.8	20.4	20.9

3.0 Finite Element Analysis

This section is dedicated to evaluating the failure of the DSJs using the elastic behavior of the joints. Finite element analysis is performed on the DSJs in order to predict the failure loads and to obtain the stress distribution across the adhesive mid-plane in the 2D model of the specimens. Only half of each specimen was modeled due to the symmetry of the joints, (see Fig. 4). The finite element program Dassault System ABAQUS-CAE-6.13 (NTNU, Trondheim, Norway) with the CPE8R element type was utilized to enable faster calculations. This element type is an eight-node, biquadratic plane strain quadrilateral element with reduced integration for 2D simulations. 3D finite element analysis was carried out for comparing results with an 8-node three-dimensional cohesive element (COH3D8) [67]. This 3D finite element analysis is strongly recommended by the ABAQUS user manual in modelling two bodies connected by adhesive [73]. The boundary conditions and the applied load in the finite element models are shown in Fig. 4. The same element type was used for finite element modelling of the adhesive joints. A mesh convergence study was undertaken in order to ensure that the proper size of elements was used for analyses. Smaller elements were used in the adhesive layer to improve the accuracy of the output

results. The mesh pattern used in this research for modelling DSJs is shown in Fig. 5. Also, as seen in Fig. 3, the load–displacement curve is linear up to the final fracture which takes place suddenly with no effective plastic deformation in the adhesive layer. Therefore, the linear elastic assumption is reasonable for failure load prediction in the DSJs. An assumption of linear elastic behavior is typical for most of the structural adhesives which behave predominantly in a linear manner until the final failure [74].

4.0 Failure Load Prediction

The effective bonding length of the DSJs should be calculated prior to failure load predictions. According to the data related to adhesive joints in the literature, increasing the bonding length results in a higher load bearing capacity. However, for the adhesive joints with the bonding lengths greater than a specific value, the failure load would remain the same. This critical bonding length is called the “effective bonding length”, L_{eff} .

A simple method of obtaining L_{eff} using 2D and 3D linear elastic finite element analysis is proposed here. One major advantage of the proposed methodology is that only the Poisson’s ratio and elastic modulus are required to calculate the effective bonding length. Therefore, finite element models of DSJs with different bonding lengths are analyzed under a constant statically applied stress. The longitudinal stress variation along the adhesive mid-plane should be exported for each bonding length (Figs. 6 and 7), and then the value of the first peak in the longitudinal stress curve should be recorded as the critical stress σ_{cr} . The corresponding critical longitudinal stress values for the rest of the joints is then obtained in a similar manner. Comparing the critical longitudinal stress values for different bonding lengths reveals the effective bonding length (Fig. 8) as the value where the critical longitudinal stress remains constant with further increase in bonding length (Fig. 8). In the next subsection, the NLS method is described in order to directly determine the value of failure strength and the effective bond length before performing any other experiments.

4.1 Calculating the Effective Bond Length using the NLS Method

The joint with 10 mm bonding length was selected as the reference joint in this study (first experimental test). The applied axial tensile stress can be any value. This load is applied to the joint and the variation of longitudinal stress along the adhesive layer is obtained. 1200 MPa was chosen as the axial tensile load in this study. The first positive peak stress from the bond line edge was considered as the critical longitudinal stress σ_{cr} and it was equal to 10.61 MPa for the reference joint. Selecting a key parameter in a failure method must be done in a way that it is consistent with the detected failure mechanism in the tested joints. Longitudinal stress along the adhesive mid-plane was considered as the key parameter in the failure analysis based on the CFRP rupture and delamination failure mechanisms due to longitudinal stresses at the junction of steel components.

A similar procedure was followed for the rest of the joints ($L_1 = 20, 30, 40, 50$ and 60 mm). The value of critical longitudinal stresses (i.e., first peak value in the longitudinal stress variation along the bond length) for all joints were obtained from elastic analyses. The longitudinal stress distribution along the path defined in the mid-plane of the adhesive layer for the other joints of the present experimental series are shown in Figs. 6 and 7. The variation of critical longitudinal stresses for different bonding lengths are illustrated in Fig. 8. As can be seen from Fig. 8, the bonding length of $L_1 = 30$ mm is the effective bonding length and for higher bonding lengths the longitudinal stress remains constant.

4.2 Obtaining Theoretical Failure Loads using the NLS Method

Equation (1) is proposed to obtain the failure loads of the joints. This equation includes a nonlinear piece for bonding lengths smaller than L_{eff} and a constant piece for DSJs with bonding lengths greater than L_{eff}

$$P_{Theor} = \begin{cases} Lagrange(x, Pointx, Pointy) & 0 < L_1 < L_{eff} \\ F^* & L_{eff} \leq L_1 \end{cases} \quad (1)$$

where F^* is the experimental failure load of adhesive joints with the effective bonding length. The Lagrange function in Equation (1) is based on MATLAB's© internal Lagrange interpolation function. According to this approach, the failure loads of DSJs with different bonding lengths can be predicted by only testing two joints. Considering any of the tested DSJs with bonding lengths greater than L_{eff} as the reference joints for failure load prediction, results in the theoretical predictions presented in Table 3 and Figure 9. Good agreement is seen between the NLS predictions and the experimental results.

Table 3: Dimensions of the adhesive joints and details of experimental and theoretical failure loads for the tested DSJs.

Series	Bonding Length, L1	Bonding Length, L2	Experimental failure load, $P_{Exp.}$ (kN)	Theoretical failure load, $P_{Theor.}$ (kN)	$P_{Theor.}/P_{Exp.}$
A	10	40	10.03	10.03	1
A	20	50	14.36	12.2	0.85
A	30	60	14.4	14.4	1
A	40	70	14.16	14.4	1.01
A	50	80	14.56	14.4	0.98
A	60	90	14.9	14.4	0.96
B	10	40	12.76	12.76	1
B	20	50	18.26	16.68	0.91
B	30	60	20.6	20.6	1
B	40	70	21.06	20.6	0.97
B	50	80	21.38	20.6	0.96
B	60	90	20.9	20.6	0.98

For further validation, the proposed method was used to predict the failure loads of six series of DSJs which have been reported in previous experimental studies by Al-Zubaidy et al. [68] in 2D simulation and Al-Mosawe et al. [67] in 3D simulation.

Al-Zubaidy et al. [68] conducted experiments on DSJs of width 50 mm and bonding lengths of $L_1 = 10, 20, 30, 40, 50, 60, 70, 80, 90,$ and 100 mm. For the experimental data series C, one layer of CFRP sheet was bonded on each side of the joints. For the experimental data series D, three layers of CFRP sheets were bonded on each side of the adhesive joints. MBrace saturant epoxy adhesive was used for joining the CFRP sheets together and also joining them to the steel plates. Table 4 presents the mechanical properties of the tested joints [68]. In order to simplify the numerical modeling, the three layers of CFRP and two layers of adhesive between them in series D were considered as a single part having an equivalent tensile modulus calculated as follows [71]:

$$E_{eq} = \frac{E_{adh} * t_{adh} + E_{CFRP} * t_{CFRP}}{t_{adh} * t_{CFRP}} \quad (2)$$

where E_{eq} is the equivalent modulus of the CFRP/adhesive layer, E_{adh} and E_{CFRP} are the tensile modulus of adhesive layer and CFRP sheets, respectively. The terms t_{adh} and t_{CFRP} signify the total bondline thickness and the thickness of the CFRP layers. Table 5 summarizes the values of the failure load, obtained experimentally from the tensile quasi-static tests on the DSJ specimens and numerically by means of the 2D finite element analyses based on the NLS method. The discrepancies between the results are also included in Table 5.

Figure 10 shows the critical longitudinal stress as a function of bonding length for the joints in series C and series D. According to the plateaus in Figure 10, the effective bonding lengths in series C and D are 30 and 40 mm, respectively. Using these effective bonding lengths and considering one of the joints ($L > L_{eff}$) as the reference joint, the failure loads of the remaining joints were estimated using the NLS method. A comparison between the NLS predictions and the experimental results is illustrated in Figure 11. A good correlation is seen between the experimental data and NLS estimates for failure loads in the tested DSJs.

Table 4
Mechanical properties of materials used for joints series C and D Al-Zubaidy et al. [68].

Material property	Series C	Series D
Steel length (mm)	210	210
Steel cross-section(mm)	50*5	50*5
Steel modulus (GPa)	203	203
Steel poisson's ratio	0.25	0.25
CFRP cross-section (mm)	50*0.176	50*1.588
CFRP modulus (GPa)	205	52.778
CFRP poisson's ratio	0.28	0.28
Adhesive modulus (GPa)	2.229	2.229
Adhesive cross-section (mm)	50*0.53	50*0.53
Adhesive poisson's ratio	0.35	0.35
Bond length range (mm)	10–100	10–100

Table 5: Details of experimental results with theoretical failure load predictions for the joints series C and D [68]

Series	Bonding Length, L1	Bonding Length, L2	Experimental Failure Load, $P_{Exp.}$ (kN)	Theoretical Failure Load, $P_{Theor.}$ (kN)	$P_{Theor.}/P_{Exp.}$
C	10	80	19.3	19.3	1
C	20	80	32.6	30.45	0.93
C	30	80	41.6	41.6	1
C	40	80	40.6	41.6	1.02
C	50	80	41.0	41.6	1.01
C	60	80	40.2	41.6	1.03
C	70	100	41.1	41.6	1.01
C	80	100	39.5	41.6	1.05
C	90	115	39.8	41.6	1.04
C	100	115	40.3	41.6	1.03
D	10	80	27.3	27.3	1
D	20	80	49.3	44.5	0.90
D	30	80	67.5	61.7	0.91
D	40	80	78.9	78.9	1
D	50	80	76.9	78.9	1.02
D	60	80	74.9	78.9	1.05
D	70	100	75.5	78.9	1.04
D	80	100	76.4	78.9	1.03
D	90	115	74.8	78.9	1.05
D	100	115	76.1	78.9	1.03

Al-Mosave et al. [67] have also reported the effects of CFRP properties and sections on the steel/CFRP laminate bonds under quasi-static loading. In their research Araldite 420 adhesive was used to join the CFRP sheets together and also to the steel plates. Table 6 presents the mechanical properties of the tested joints from [67].

Table 7 summarizes the values of the failure load, obtained experimentally from the tensile quasi-static tests on the DSJ specimens and numerically by means of the 3D finite element analysis based on the NLS method, including the discrepancies. The choice of 3-D modelling was necessary to enable good comparisons between the actual failure loads and finite element analysis prediction in the cohesive zone model and NLS model.

Figure 12 summarizes the critical longitudinal stress as a function of bonding length for the joints in series E, F, G and H from [67]. According to the plateaus in Fig. 12, the effective bonding length of the joints in series E, F, G and H are 80, 70, 90 and 70 mm, respectively. Using these effective bonding lengths and considering one of the joints as the reference joint, the failure loads of the remaining joints were estimated using the NLS method. A comparison between the NLS predictions and the experimental results is illustrated in Fig. 13. A very good correlation is seen between the experimental data and the NLS estimates for the failure loads in the tested DSJs.

Table 6
Mechanical properties of materials used for joints series E, F, G and H. Al-Mosawe et al. [67].

Material property	Series E	Series F	Series G	Series H
Steel length (mm)	200	200	200	200
Steel cross-section(mm)	40 * 10	40 * 10	40 * 10	40 * 10
Steel modulus (GPa)	203	203	203	203
Steel poisson's ratio	0.25	0.25	0.25	0.25
CFRP cross-section (mm)	20 * 1.4	10 * 1.4	20 * 1.4	20 * 1.2
CFRP type	Low-modulus	Low-modulus	Normal-modulus	Ultra-high modulus
CFRP modulus (GPa)	159.4	159.4	203.0	450.0
CFRP poisson's ratio	0.28	0.28	0.28	0.28
Adhesive modulus (GPa)	1.9	1.9	1.9	1.9
Adhesive cross-section (mm)	20 * 1.4	10 * 1.4	20 * 1.4	20 * 1.2
Adhesive poisson's ratio	0.21	0.21	0.21	0.21
Bond length range (mm)	30–130	30–130	30–130	30–130

Table 7

Details of experimental results, theoretical failure load predictions from NLS and CZM and PS models along with their corresponding error percentage for the joints series E through H.

Series	Bonding length, L1	Bonding length, L2	Experimental failure load, P_{Exp} (kN)	Theoretical failure load: NLS model (kN)	Theoretical failure load: CZM model (kN)	%Error: NLS	%Error: CZM
E	30	100	41.0	41	30.1	0	26.58
E	40	100	50.7	50.16	40.1	1.06	20.90
E	50	100	60.0	59.2	51.2	1.33	14.66
E	60	100	69.1	68.48	59.2	0.89	14.32
E	70	110	76.5	77.64	68.4	1.49	10.58
E	80	120	86.8	86.8	79.2	0	8.75
E	90	130	93.6	86.8	88.1	7.26	5.87
E	100	140	100.3	86.8	94	13.45	6.28
E	110	150	108.0	86.8	100.4	19.62	7.03
E	120	160	108.7	86.8	100.8	20.14	7.26
E	130	170	109.1	86.8	101	20.43	7.42
Average:						7.78	11.78
F	30	100	15.52	15.52	15.4	0	0.77
F	40	100	20.4	19.79	20.7	3.10	1.32
F	50	100	25.70	24.07	26.3	6.34	2.33
F	60	100	27.63	28.34	32.1	2.56	16.17
F	70	110	32.62	32.62	34	0	4.23
F	80	120	30.43	32.6	39.1	7.13	28.49
F	90	130	31.10	32.6	42.4	4.82	36.33
F	100	140	33.48	32.6	42.8	2.62	27.83
F	110	150	34.95	32.6	51.2	6.72	46.49
F	120	160	31.97	32.6	51.6	1.97	61.40
F	130	170	35.03	32.6	51.7	6.93	47.58
Average:						3.84	24.81
Loading [MathJax]/jax/output/CommonHTML/fonts/TeX/fontdata.js					30.8	0	26.49

Series	Bonding length, L1	Bonding length, L2	Experimental failure load, P _{Exp.} (kN)	Theoretical failure load: NLS model (kN)	Theoretical failure load: CZM model (kN)	%Error: NLS	%Error: CZM
G	40	100	51.7	50.61	40.1	02.38	22.43
G	50	100	60.7	59.33	50.4	2.10	16.96
G	60	100	69.8	68.05	59.4	2.25	14.89
G	70	110	75	76.76	67.2	2.50	10.4
G	80	120	87.4	85.48	80	2.34	8.46
G	90	130	94.2	94.2	89.2	0	5.30
G	100	140	101.3	94.2	96.3	7	4.93
G	110	150	108.3	94.2	101.6	13.01	6.18
G	120	160	108.5	94.2	100.8	13.17	7.09
G	130	170	109.2	94.2	101.2	13.71	7.32
Average:						4.61	11.85
H	30	100	31.76	31.76	30.1	0	5.22
H	40	100	43.3	42.12	40.4	2.72	6.69
H	50	100	54.4	52.48	53.4	3.52	1.83
H	60	100	64.1	62.84	63.7	1.96	0.62
H	70	110	73.2	73.2	71.6	0	2.18
H	80	120	73.2	73.2	71.9	0	1.77
H	90	130	73.2	73.2	71.4	0	2.45
H	100	140	73.2	73.2	72.6	0	0.81
H	110	150	73.2	73.2	72.4	0	1.09
H	120	160	73.2	73.2	72.1	0	1.50
H	130	170	73.2	73.2	72.3	0	1.22
Average:						0.74	2.3

5.0 Limitations And Advantages Of The Nls Method

According to the experimental results, the load–displacement curves were linear up to the final fracture which took place suddenly with no effective plastic deformation in the adhesive layer. This indicates that

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the adhesive and the adherents remained in linear elastic state. Therefore, the linear elastic fracture mechanics (LEFM) assumption is reasonable for failure load predictions. There are some limitations and advantages in using the NLS method for DSJs as a failure load prediction model which are briefly discussed in this section. Generally, the term “failure” in this theoretical prediction model for DSJs, describes the state of total failure modes such as adhesive layer failure (cohesive failure), de-bonding between the CFRP and the adhesive layer or between the steel plates and the adhesive layer (adhesive failure), CFRP rupture, CFRP delamination, or a combination of modes.

In order to predict the failure load of the DSJs with different bonding lengths using the NLS model, only two DSJ samples corresponding to the effective bonding length should be tested, which is a major advantage of this methodology. It is very important to note that although the NLS method for DSJs has two experimental parts, the value of the effective bond length can be estimated conveniently and rapidly by applying the NLS method proposed in this research before conducting any experiments. However, in some experimental research (see for instance Refs. [74]) complicated numerical finite element analysis by considering the material nonlinear behavior should be conducted to find the effective bond length. In fact, the simple calculation in predicting the effective bond length using a nonlinear Lagrange approach is the substantial advantage in the NLS method for DSJs.

Various approaches to predict the failure behavior of adhesively bonded joints based on strain, stress, and energy have been suggested in the literature. Almost all of these methods require consideration of the nonlinear behavior of materials in the finite element simulation and need additional material properties to estimate the failure loads of the adhesive joints. However, in the NLS method, failure can be predicted by conducting 2D-linear and 3D-linear elastic analysis. Comparing the current prediction results with previously published results available in the literature reveals that the average accuracy of the non-linear Lagrange stress is in the same order or even better of the other failure models such as the cohesive zone model and the Hart–Smith model. Due to the advantages noted above, one may recommend the use of the NLS method for predicting the failure load in steel/CFRP DSJs.

6.0 Conclusions

The use of CFRP as externally bonded sheets is an effective approach to repair and improve the strength of damaged steel structures. In this paper, a new method, namely the non-linear Lagrange stress method, was presented for failure load prediction in steel/CFRP adhesively bonded DSJs based on longitudinal stress along the adhesive mid-plane. According to this method, after calculating the effective bonding length, the failure of DSJs with different bonding lengths can be predicted. Two sets of experimental tests were conducted on DSJs and the results of theoretical predictions of the NLS model were compared with the experimental results. The accuracy of the NLS method, including 2D and 3D simulations, was also verified by using six set of DSJ test results from the literature. It was revealed that the average accuracy of the NLS method was superior to the CZM method and this approach could estimate the experimental failure loads very well. The results showed that under the best conditions, the NLS model is

3times more accurate than the CZM model. The main reason for the increased accuracy was attributed to the underlying nonlinear nature of the NLS model.

Declarations

Authors' contributions H. Hamedi carried out the numerical analysis and respective data analysis, performed the validation study and sketched the paper. A. Kamyabi-Gol built and revised the paper until final form.

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Data availability The raw/processed data required to reproduce these findings cannot be shared at this time due to technical or time limitations.

Compliance with ethical standards

Conflict of interest The authors declare that they have no conflict of interest.

Code availability Not applicable.

Compliance with Ethical Standards

- Disclosure of potential conflicts of interest Not applicable
- Research involving Human Participants and/or Animal Not applicable
- Informed consent Not applicable

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Figures

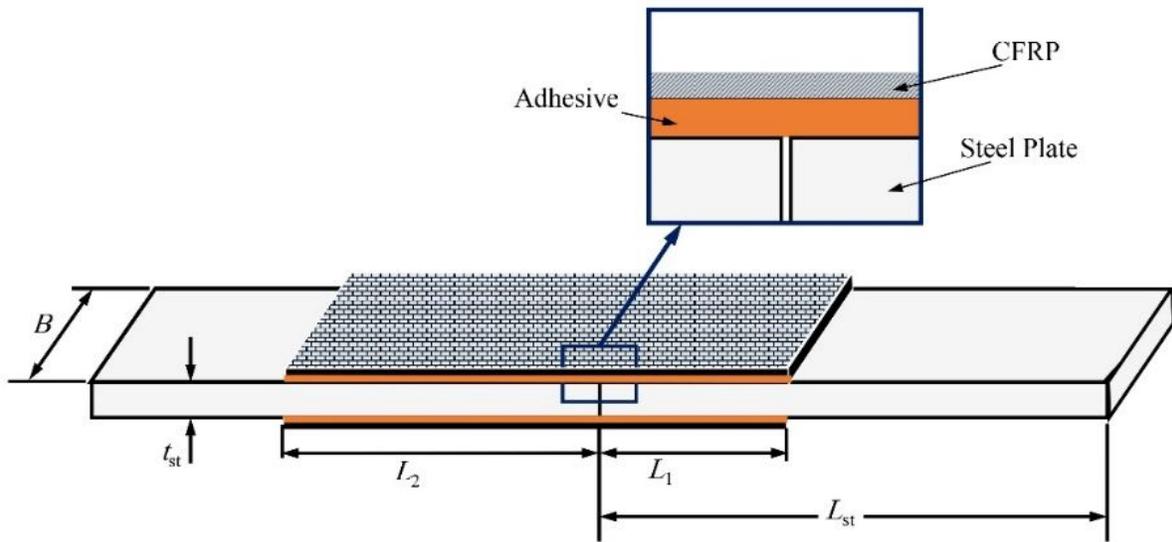
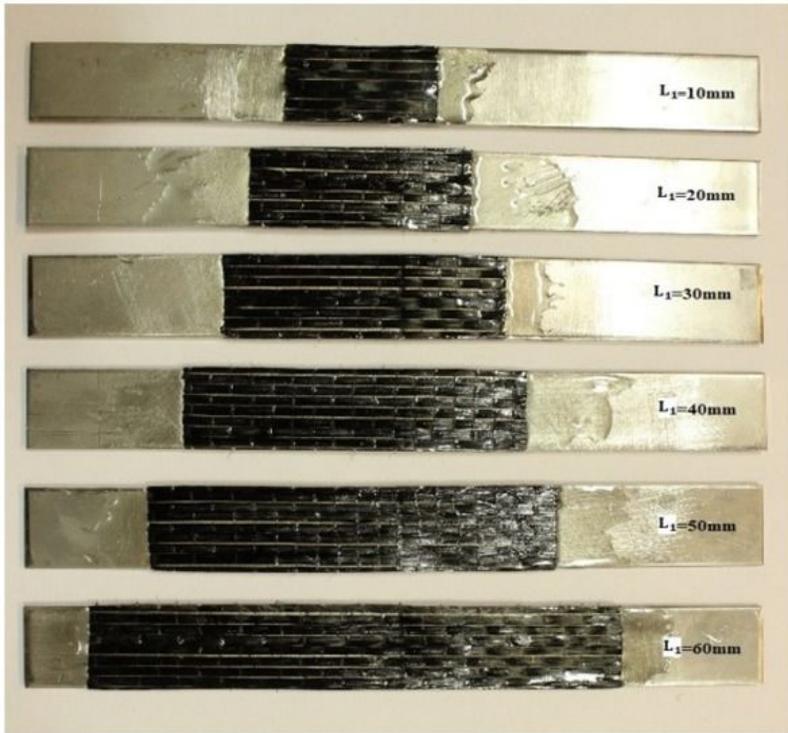
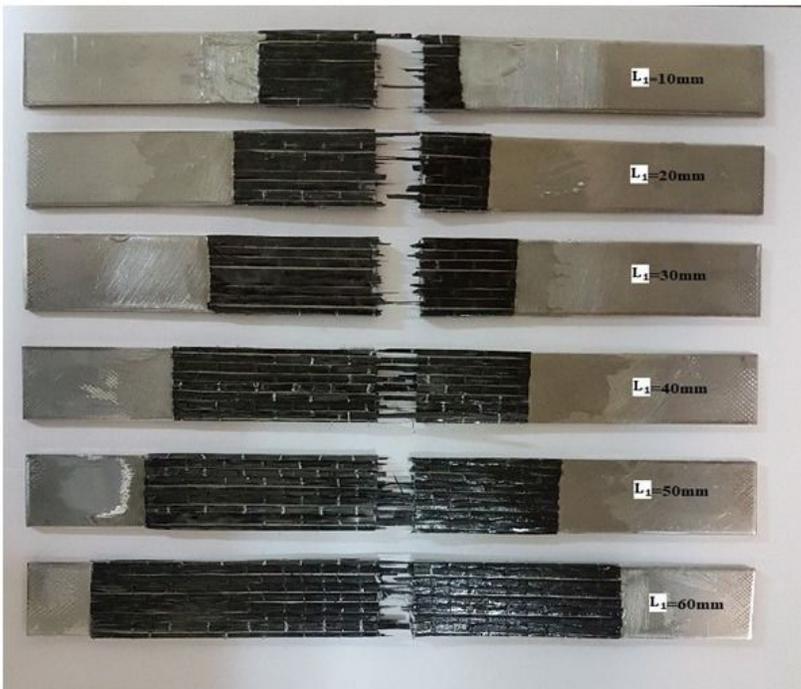


Figure 1

A schematic of steel/CFRP double strap joint.



(a)



(b)

Figure 2

Double strap joints (DSJs) prepared for experiments (a) before the test; (b) after the test

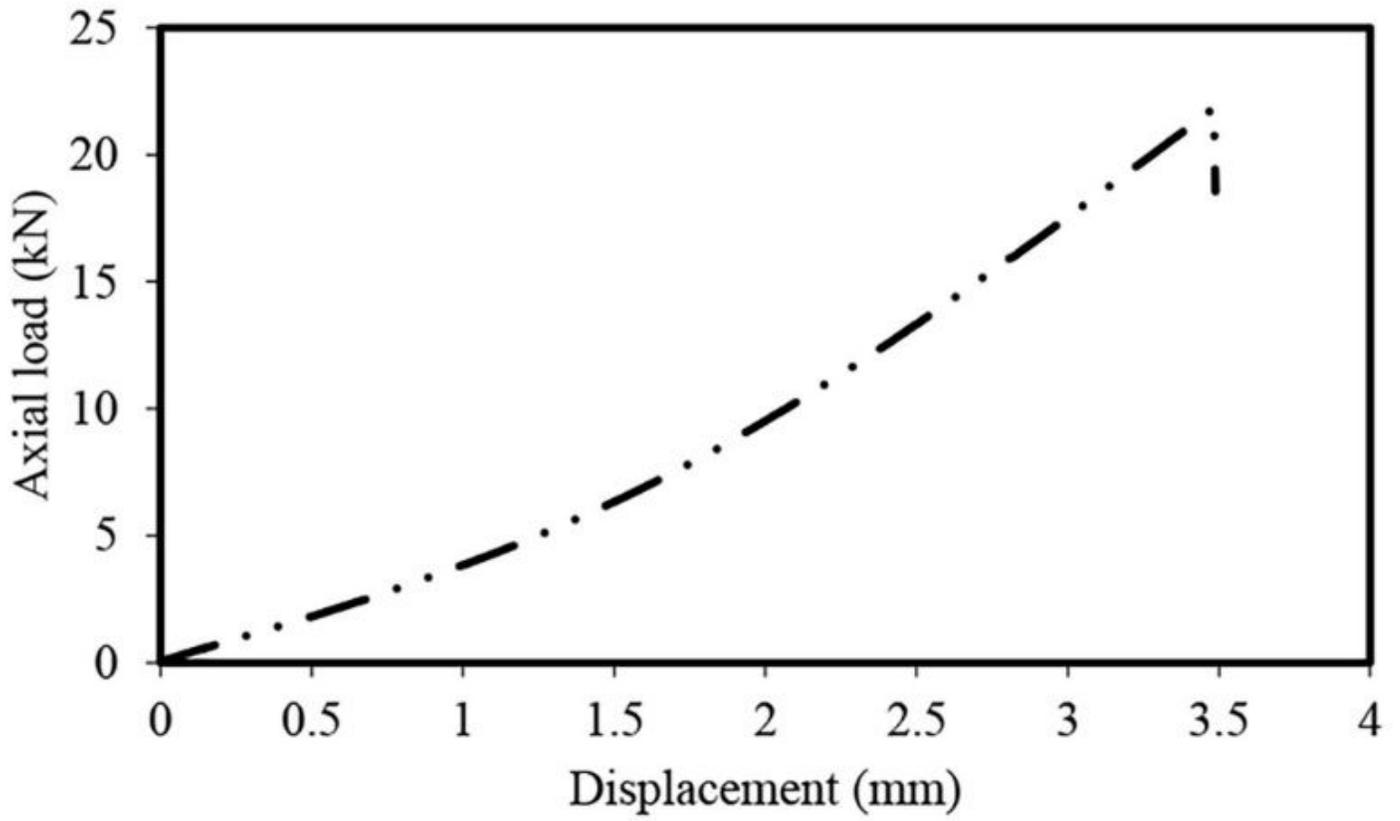
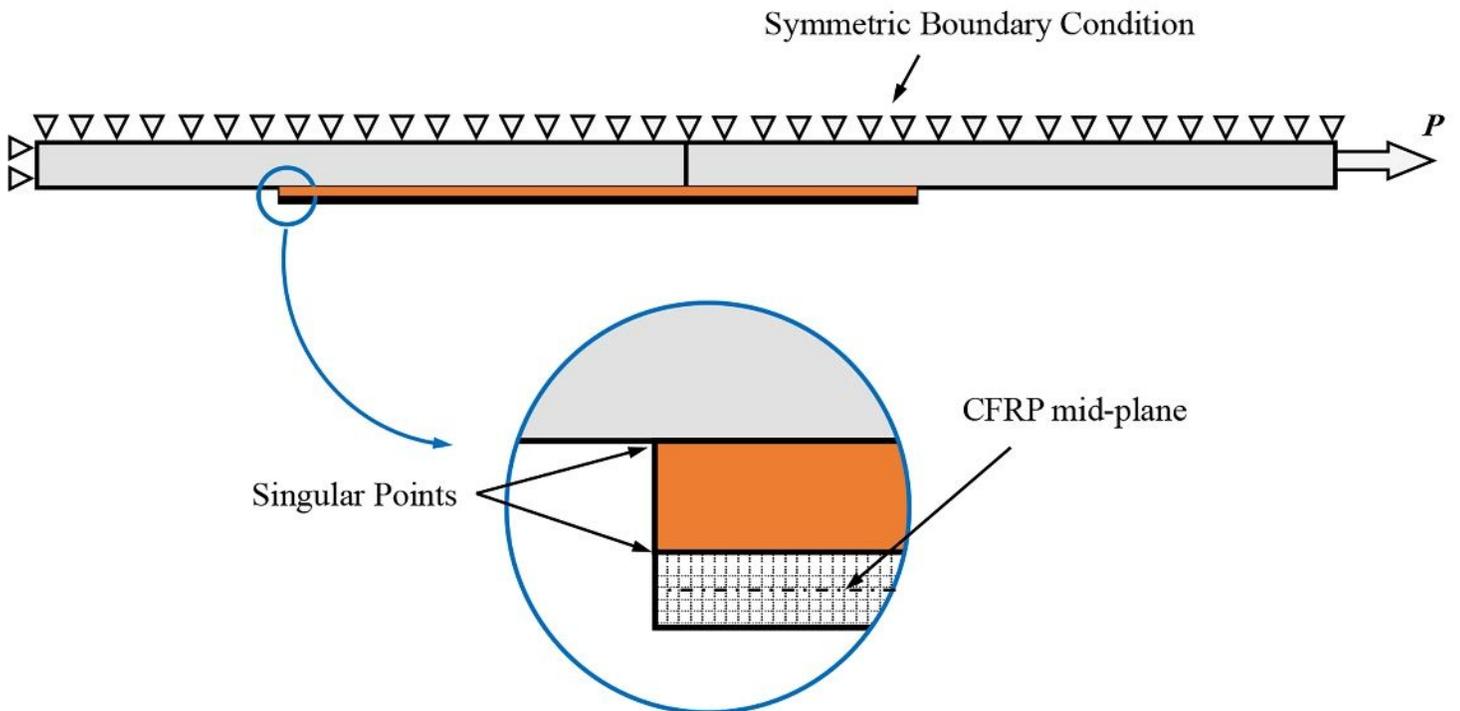


Figure 3

An example of a load–displacement curve for the steel/CFRP DSJs under tensile quasi-static loading



Applied boundary and loading conditions in the finite element model. (P: applied load)

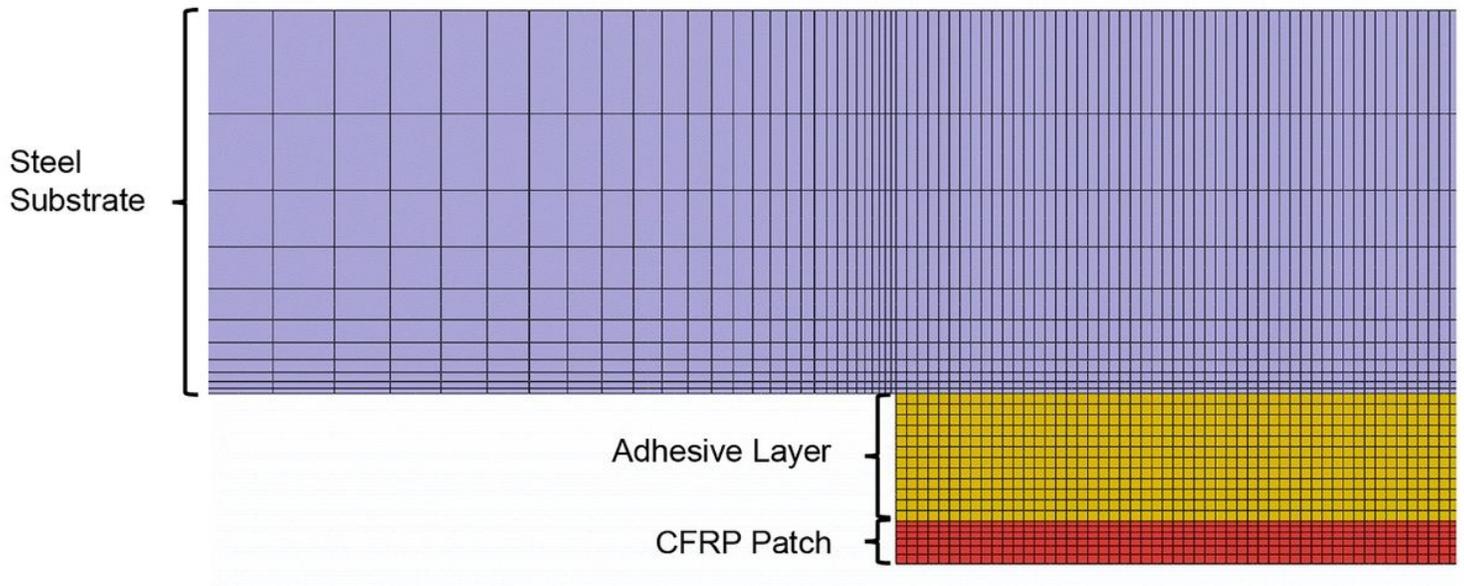


Figure 5

A typical mesh pattern used for DSJ models in this research.

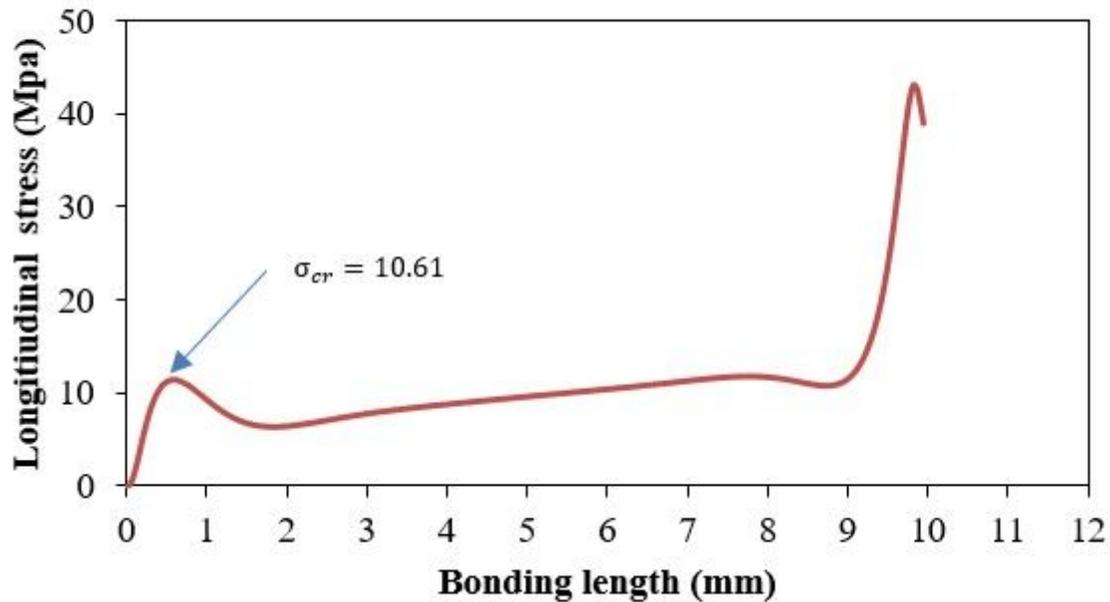
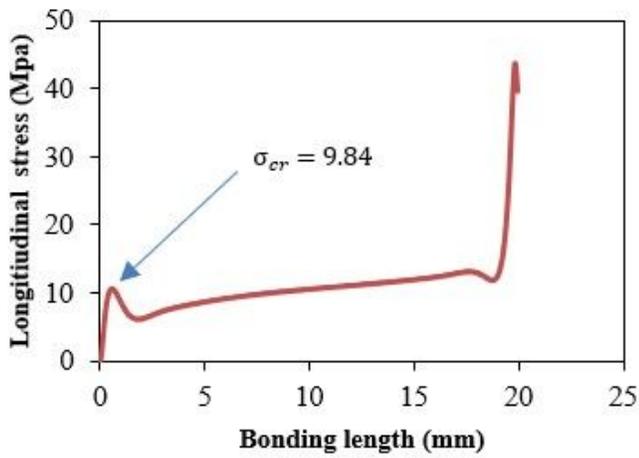
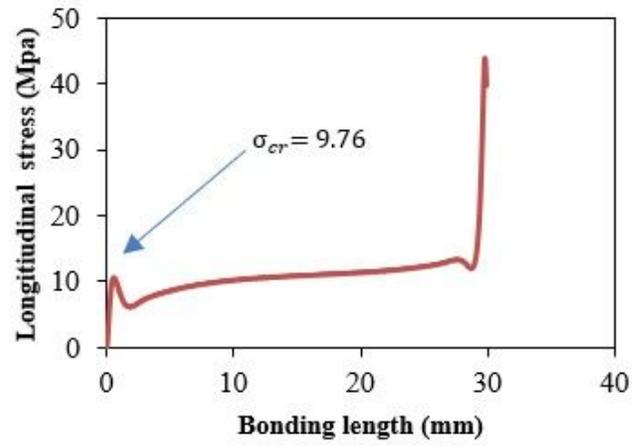


Figure 6

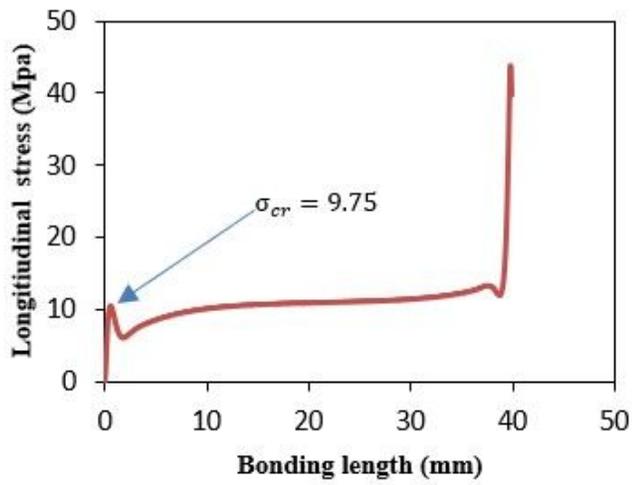
Longitudinal stress distribution along the path defined in the mid-plane of the adhesive layer for the present experimental series A-L1 = 10.



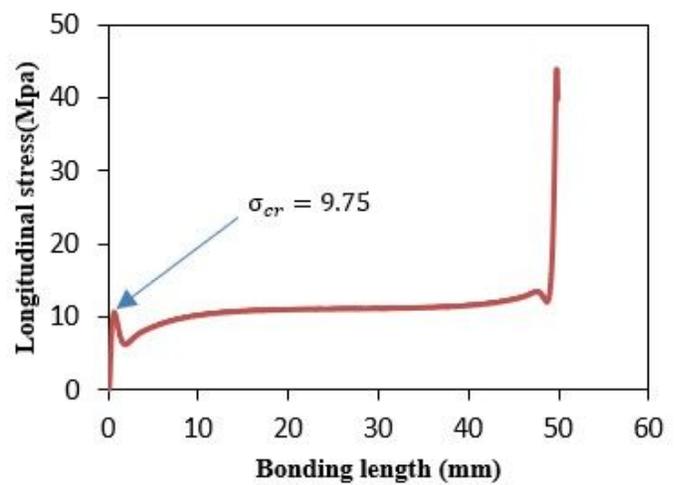
(a)



(b)



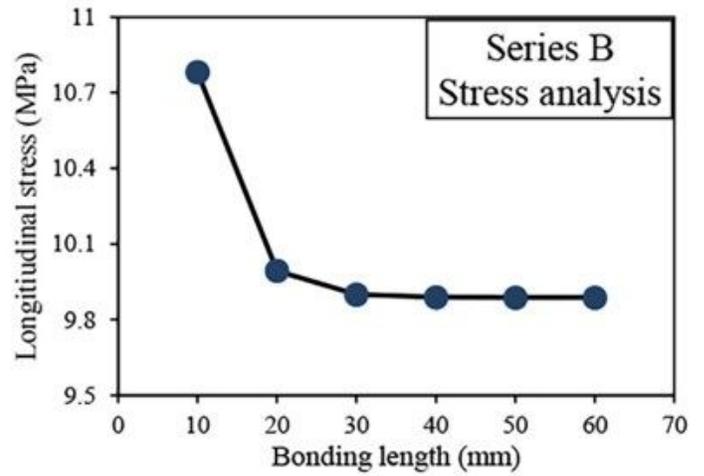
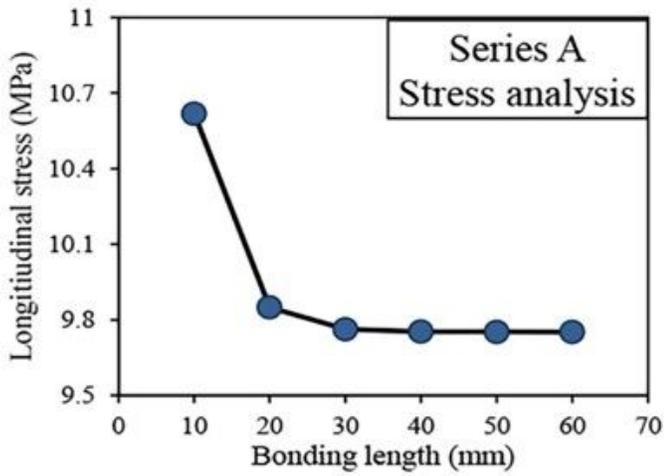
(c)



(d)

Figure 7

Longitudinal stress distribution along the path defined in the mid-plane of the adhesive layer for the present experimental series—(a) $L_1 = 20$ mm; (b) $L_1 = 30$ mm; (c) $L_1 = 40$ mm; (d) $L_1 = 50$ mm (applied stress for all the specimens: 1200 MPa).

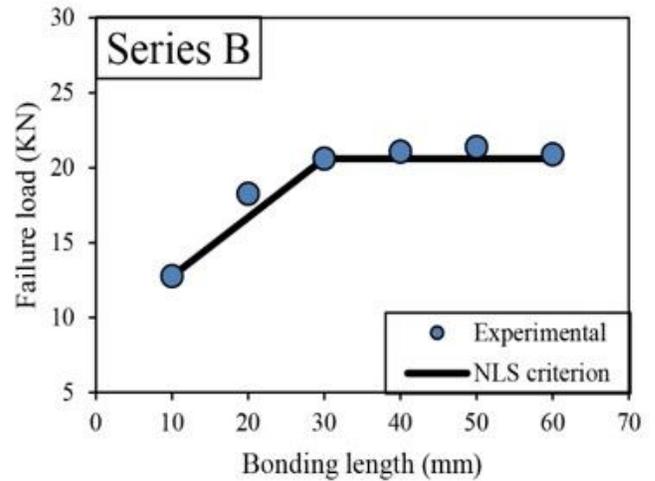
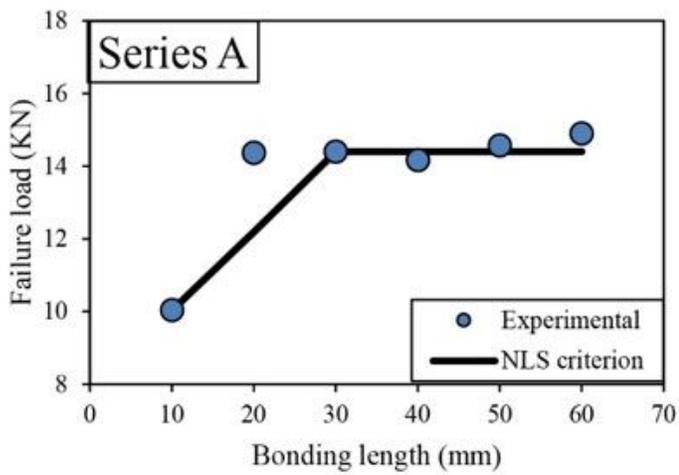


(a)

(b)

Figure 8

The variation of critical longitudinal stress for different bonding lengths (applied stress for all the specimens: 1200 MPa).



(a)

(b)

Figure 9

Comparison between the experimental failure loads of DSJs and the theoretical predictions obtained by means of point stress (NLS) method; (a) series A; (b) series B.

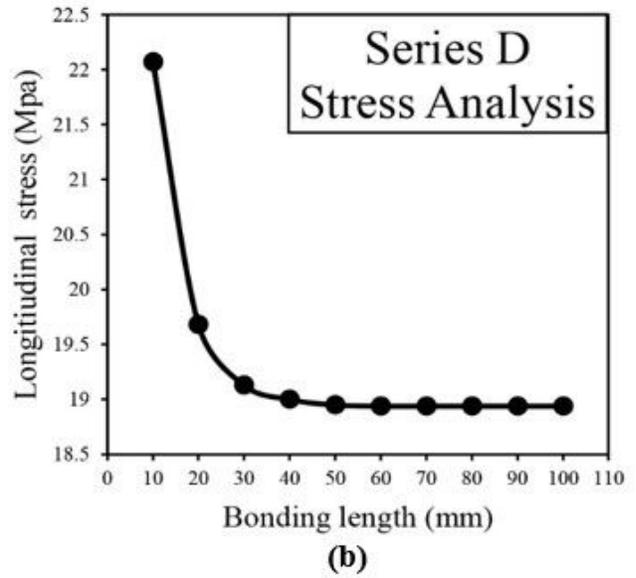
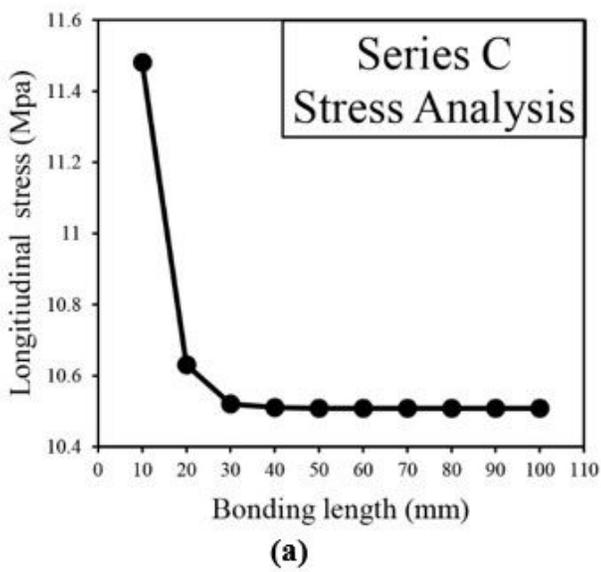


Figure 10

The variation of critical longitudinal stress for different bonding lengths (applied stress for all the specimens: 1200 MPa); (a) series C; (b) series D.

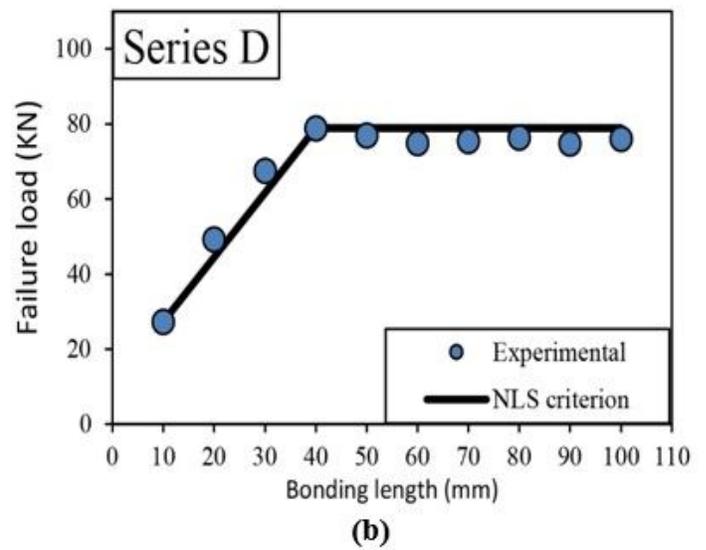
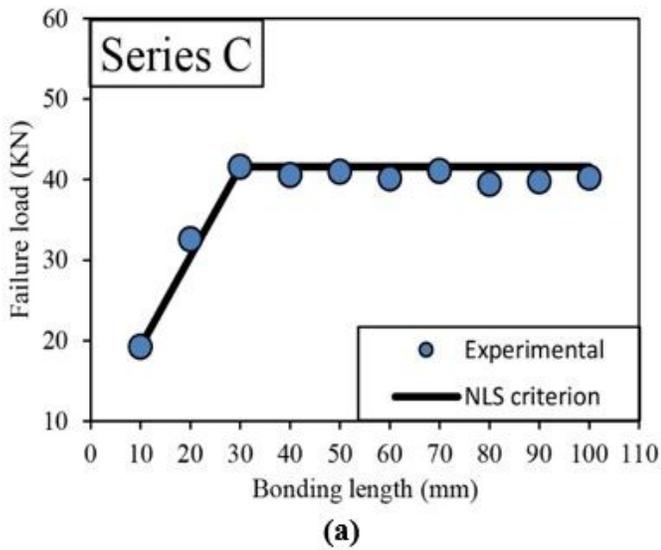
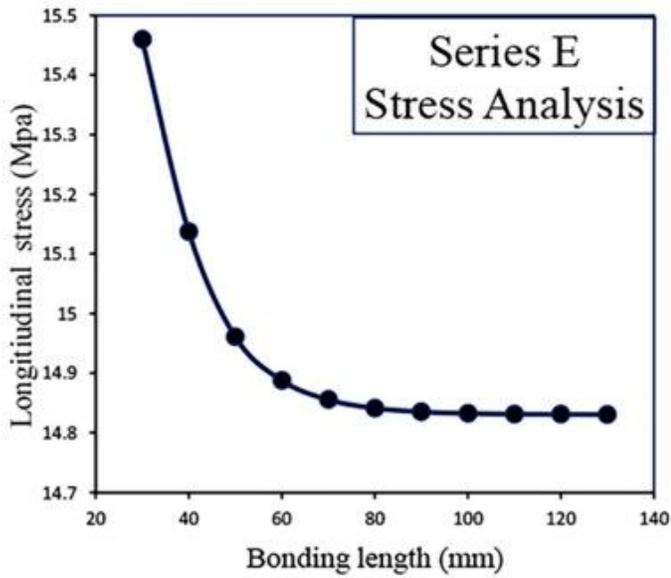
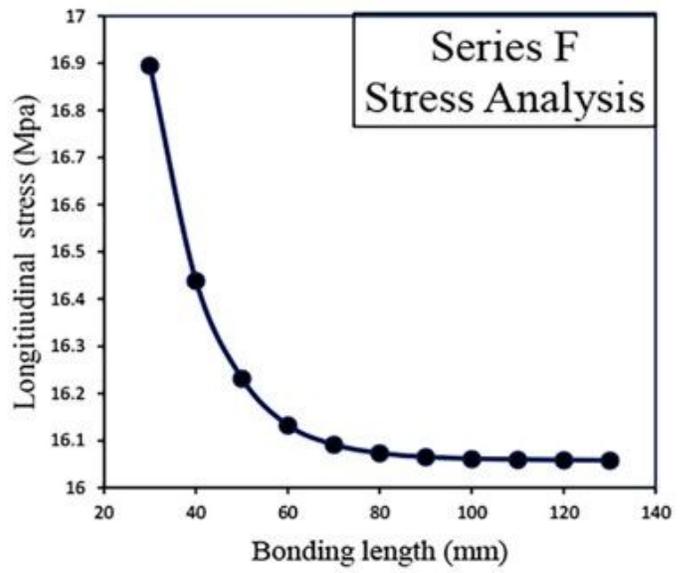


Figure 11

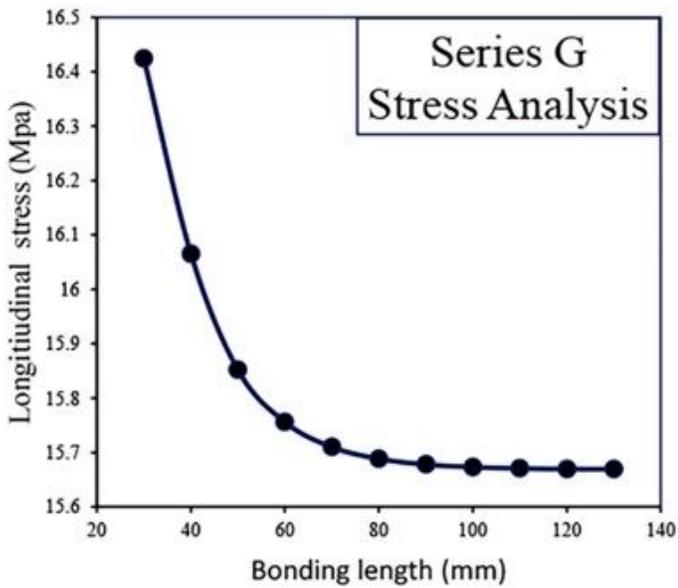
Comparison between the experimental failure loads of Double Strap Joints and the theoretical predictions obtained by means of NLS method; (a) series C; (b) series D.



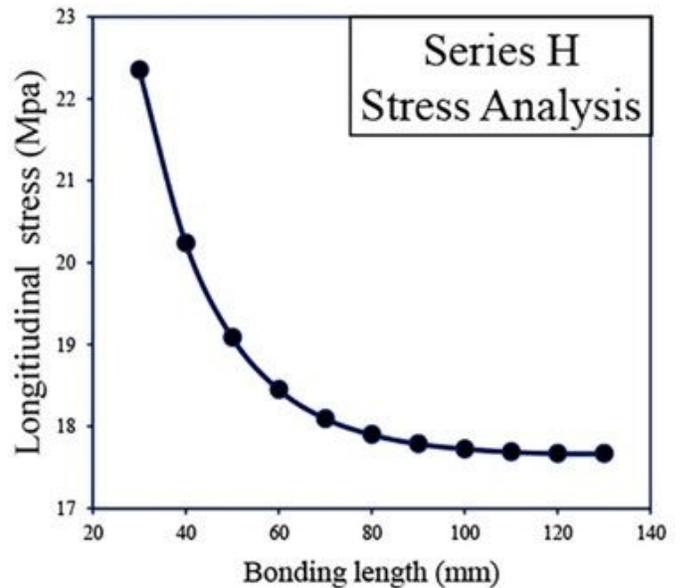
(a)



(b)



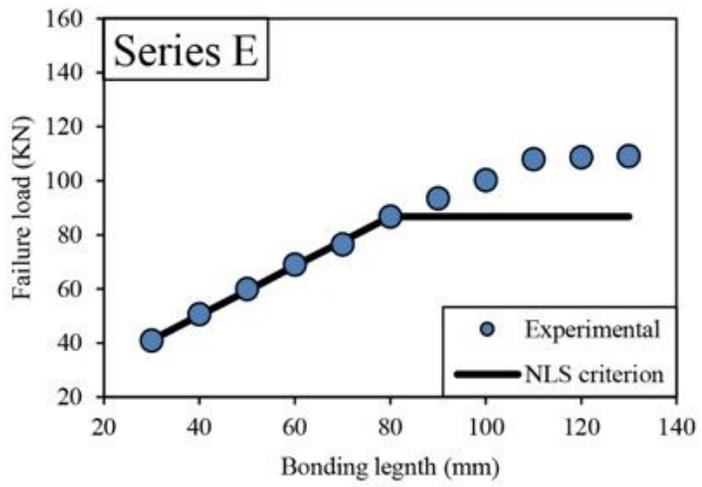
(c)



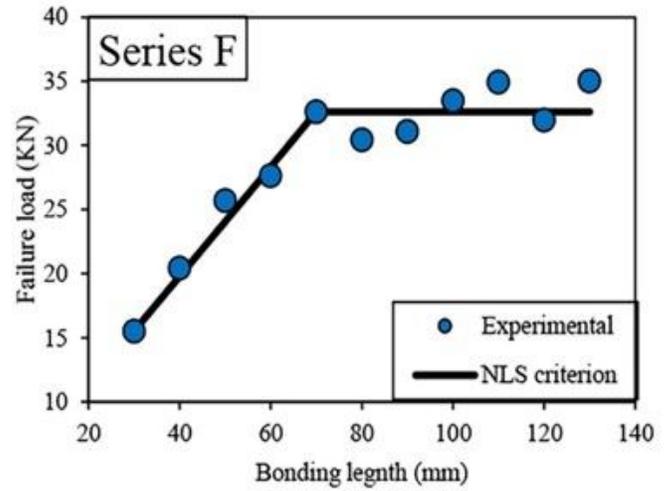
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Figure 12

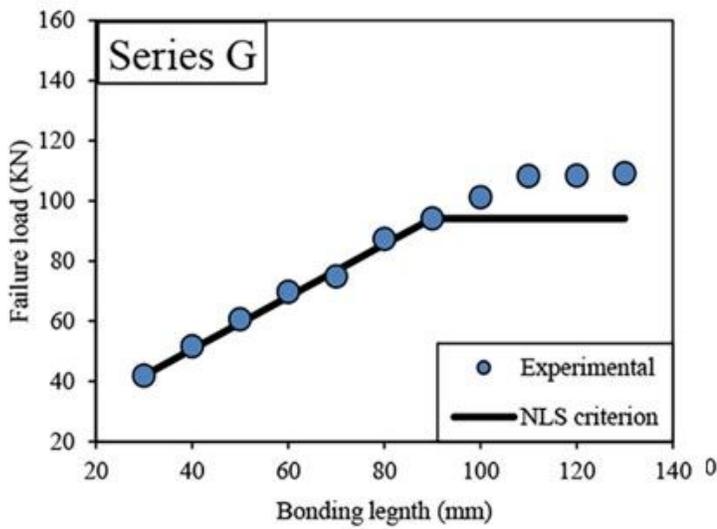
The variation of critical longitudinal stress for different bonding lengths (applied stress for all the specimens: 1200 MPa); (a) series E; (b) series F; (c) series G; (d) series H.



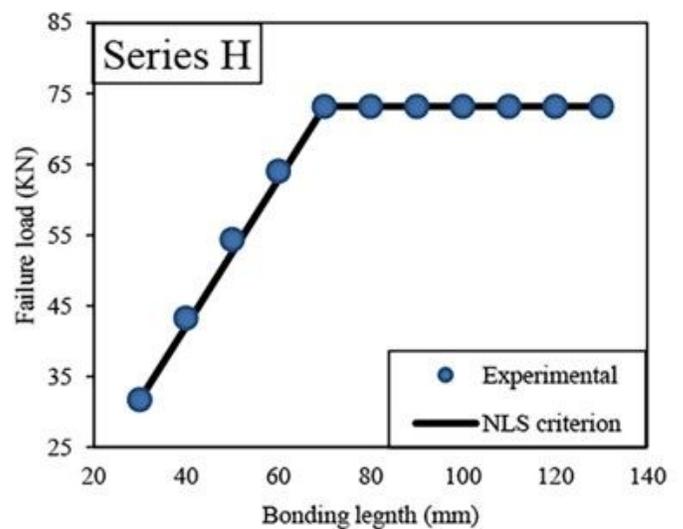
(a)



(b)



(c)



(d)

Figure 13

Comparison between the experimental failure loads of Double Strap Joints and the theoretical predictions obtained by means of NLS method; (a) series E; (b) series F; (c) series G; (d) series H.