

Experimental Study of Solar Energy Based Water Purifier (SEBWP) of Single Slope Type by Incorporating N Similar Evacuated Tubular Collectors (ETCs) having Series Connection

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Research Article

Keywords: evacuated tubular collector, solar still, experimental study, exergo-enviro-economic analysis, energy metrics

Posted Date: June 11th, 2021

DOI: <https://doi.org/10.21203/rs.3.rs-558075/v1>

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1 **Experimental study of solar energy based water purifier (SEBWP) of single slope type by**
2 **incorporating N similar evacuated tubular collectors (ETCs) having series connection**

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12 **Abstract**

13 This research paper deals with the experimental investigation of solar energy based water
14 purifier (SEBWP) of single slope type by incorporating N similar evacuated tubular collectors
15 (ETCs) having series connection. Experimental investigation has been done for a year from
16 August 2018 to July 2019. MATLAB has been used for evaluating performance parameters of
17 the system followed by the validation of these results with their experimental values. A fair
18 agreement has been found between theoretical and experimental values. Values of correlation
19 coefficients for condensing glass temperature, water temperature and water yield have been
20 found to be 0.9932, 0.9928 and 0.9951 respectively. Further, energy metrics, productivity, cost
21 of producing one kg of fresh water, exergoeconomic and enviroeconomic parameters have been
22 evaluated. Values of energy payback time, per kg cost of producing fresh water and exergy loss
23 per unit Rs. have been evaluated to be 1.72 years, Rs. 0.95/kg and 0.128 kWh/Rs. respectively.

24 **Key words:** evacuated tubular collector; solar still; experimental study, exergo-enviro-economic
25 analysis, energy metrics

26 **1. Introduction:**

27 The design, analysis, installation and experimental study of solar energy based water purifier
28 (SEBWP) of single slope type by incorporating N alike evacuated tubular collectors (ETCs) is a
29 pressing need at a time when the world is grappling with the current problem of fresh water
30 scarcity. The purification of dirty water using solar energy is one of the best solutions for
31 providing the fresh water as it is environment friendly and does not need much technical
32 knowledge for its maintenance. The experimental study of solar energy based single slope water
33 purifier (SEBWP) by incorporating flat plate collector was presented by Rai and Tiwari (1983).
34 Since then, a lot of modifications in the design of SEBWP operating in active mode have been
35 reported. Tripathi and Tiwari (2006) have studied SEBWP in active mode for different water
36 depth using solar fraction and concluded that the internal convective heat transfer coefficient
37 decreases with rising water depth due to increases in the sensible heat content of water mass.
38 Dimri et al. (2008) investigated performance of SEBWP in active mode by incorporating
39 material of cover and it was concluded that the production of fresh water (yield) of reported
40 system was higher with copper due to higher thermal conductivity of copper as compared to
41 glass and plastic.

42 The main drawback of SEBWP in active mode reported by Rai and Tiwari (1983), Tripathi and
43 Tiwari (2006), Dimri et al. (2008) was that SEBWP systems were not self sustainable as the
44 pump was running using conventional source of energy i.e. electrical energy from grid. These
45 systems could be made self sustainable by integrating photovoltaic panel with collector. Based

46 on this concept, Kumar and Tiwari (2009) reported hybrid active SEBWP and compared results
47 with SEBWP in passive mode. They revealed that the lesser production cost of fresh water from
48 passive SEBWP was obtained due to less materials required for the production of passive
49 SEBWP; however, the volume of fresh water production from passive SEBWP was very low and
50 hence could not be commercialized. Further, Dev and Tiwari (2010) reported the thermal
51 modeling of PVT integrated SEBWP in active mode and concluded that the nonlinear
52 characteristic equation is better suited for performance analysis. Singh et al. (2011) studied PVT
53 integrated SEBWP of double slope type experimentally and it was reported that the yield was 1.4
54 times higher than the similar single slope set up due to better distribution of solar radiation
55 throughout the day in the case of double slope type. El-Sebaai et al. (2011) investigated SEBWP
56 by incorporating shallow solar pond and reported that the fresh wateryield was 68.12% more than
57 the simple SEBWP due to addition of heat by solar pond. Esfahani et al. (2011) have investigated
58 experimentally a special type portable SEBWP consisting of solar collector, thermoelectric
59 cooling device for enhancing condensation, black wool covered wall and concluded that the
60 output was comparable with other types of SEBWP. Arslan (2012) investigated experimentally
61 the different designs of SEBWP in active mode and concluded that the circular box type SEBWP
62 in active mode is most efficient and the highest daily efficiency was obtained as 68.1% due to
63 improvement in the design.

64 The fresh water yielding of SEBWP in active mode can be improved by replacing flat plate
65 collectors by evacuated tubes as vacuum is present in such tubes which prevent heat loss by
66 convection. Singh et al. (2013) and Kumar et al. (2014) developed thermal model for SEBWP of
67 single slope type by incorporating evacuated tubes in which end points of all pipe were slotted in
68 the basin of solar still in natural as well as forced modes of flow respectively. Sampatkumar et al.

69 (2013) investigated SEBWP of single slope type integrated with evacuated tubes and it was
70 concluded that the yield of SEBWP was increased by 129% after integrating with evacuated
71 tubes due to additional heat addition by evacuated tubes to basin of SEBWP.

72 In another study, Hamadou and Abdellatif (2014) investigated SEBWP in active mode of
73 operation for sea water production under optimized condition and concluded that the fresh water
74 production is not the proportion of heat transfer fluid rate. Feilizadeh et al. (2015) investigated
75 experimentally multistage SEBWP in active mode by incorporating solar collectors and
76 concluded that the percentage increase in fresh water production decrease as the collector to
77 basin area ratio is increased because heated water is further heated. Taghvaei et al. (2015)
78 investigated SEBWP in active mode experimentally for five days continuously and concluded
79 that the overall fresh water production and efficiency decreased with the increases in brine depth
80 due to sensible heat absorbed by brine mass at increased brine depth. Sandeep et al. (2015)
81 studied SEBWP of single slope type in which extra condensing surface was provided and
82 concluded that the fresh water production in the improved design was 14.5% higher than the
83 conventional SEBWP of single slope type due to improvement in the condensation as the extra
84 condensation surface was provided. Singh et al (2016) investigated PVT integrated SEBWP of
85 single slope type and concluded that there was a fair conformity between values of theoretical
86 and experimental analyses with coefficient of correlation varying between 0.97 and 0.99.

87 Issa and Chang (2017) studied SEBWP by integrating with evacuated tube in mixed mode
88 condition and it was concluded that the yield was better than the conventional SEBWP because
89 of heat addition by evacuated tubes in mixed mode connection to basin. Sahota and Tiwari
90 (2017) developed characteristic equation and reported an improvement in fresh water production
91 by 32% with CuO nanofluid over base fluid (water) due to increased absorptivity of nanofluid.

92 Joshi and Tiwari (2018) investigated SEBWP in active mode of operation by incorporating heat
93 exchanger and reported that the fresh water yield cost was lowest for partially covered flat plate
94 collectors (FPC) with PVT; whereas, SEBWP integrated with fully covered PVT-FPC performed
95 best for electricity generation. Singh and Tiwari (2017), Singh (2018, 2019) and Singh and Al-
96 Helal (2018) performed the analytical study of basin type SEBWP by incorporating N alike
97 evacuated tubular collectors from energy, exergy, cost and energy metrics viewpoints and
98 concluded that SEBWP of double slope type performed better than SEBWP of single slope type
99 due to better distribution of solar energy throughout the day in the case of double slope type.

100 Kumar et al. (2018) investigated SEBWP operating in active mode experimentally and
101 concluded that the fresh water production from SEBWP operated in active mode was six times
102 more than the SEBWP in passive mode due to addition of more heat by collectors and increased
103 temperature difference between water surface and condensing cover as the condensing cover was
104 cooled. Gupta et al. (2018) developed the distinctive equation for SEBWP of single slope type by
105 incorporating CPC which was fully covered with PV for the same packing factor as that of
106 partially covered and it was reported that the instantaneous efficiency of the system containing
107 CPC with full coverage of PV was better than SEBWP consisting of partially covered PVT-CPC
108 due decreased top loss. Elsheikh et al. (2019) reported the application of artificial neural network
109 for different solar energy devices for optimization and prediction of performance parameters and
110 reviewed the work on solar energy devices. Elbar et al. (2019) investigated SEBWP of single
111 slope type by integrating PV and it was reported that the yield obtained was higher by 31.48% for
112 the PV integrated SEBWP over conventional SEBWP because PV acted as reflector which
113 allows more solar energy into the SEBWP. Feilizadeh et al. (2019) studied thermosyphon
114 SEBWP in active mode with improved condenser and concluded that the increase in the

115 production of fresh water was 46% higher with improved condenser due to the difference of
116 partial vapor pressure between the water surface and condenser surface. Bait (2019) reported the
117 experimental study of SEBWP of double slope type by incorporating tubular solar collector and
118 concluded that the annual production was 35.73% more over the conventional SEBWP due to
119 compact design of collector. Sharshir et al. (2019) conducted an experimental analysis of a
120 pyramid-type SEBWP incorporating evacuated tubes and filled with nanofluid and concluded
121 that the modified system produced 64.5 % more fresh water than the traditional SEBWP due to
122 improved fluid properties.

123 Essa et al. (2020) studied SEBWP operating in active mode using artificial neural network and
124 reported that Hawks Optimizer – artificial neural network was the most suitable for forecasting
125 the production of fresh water by active mode operated SEBWP. Parsa et al. (2020) reported the
126 effect of variation of water depth on SEBWP powered by photovoltaic and concluded that the
127 fresh water yield was 42.5% more if the depth was raised to 70.23% (from 3871 to 13005) due to
128 increased radiation, decreased atmospheric pressure and ambient air temperature. Hassan (2020)
129 investigated SEBWP experimentally by incorporating parabolic trough collector and concluded
130 that maximum freshwater production increases by about 6% in case of using double slope type
131 than the similar single slope set up. Tiwari et al. (2020) have reported the outcome of condensing
132 cover effect on PVT-CPC integrated conical solar still performance and it was found that the
133 production of fresh water (yield) of active conical SEBWP is higher than conventional SEBWP
134 due to increased condensing cover surface area. Shoeibi et al. (2020) studied SEBWP of double
135 slope type by incorporating thermoelectric cooling and heating and concluded that due to the
136 increased water temperature in the modified solar still, the yield of the modified solar still was
137 76.4 percent higher than the traditional SEBWP of double slope type. Kumar et al. (2020)

138 reported the effect of variation of number of collectors on the environmental parameter of
139 SEBWP of single slope type and concluded that due to an increase in the amount of heat added
140 to the SEBWP basin, the value of carbon credit increased with the number of collectors. Further,
141 Singh et al. (2020) examined the impact of mass flow rate variation on the life cycle conversion
142 efficiency of a single slope SEBWP and concluded that as the mass flow rate decreased, the
143 system's life cycle conversion efficiency improved because fluid flowing through collector tubes
144 had more time to consume solar energy. Shehata et al. (2020) investigated ultrasonic humidifier
145 augmented SEBWP with evacuated collector experimentally and concluded that concluded that
146 the ultrasonic humidifier improved the productivity by 44% due to circulation of water between
147 solar still and evacuated collector.

148 From the extant research study, it has been found that the theoretical study of SEBWP of single
149 slope type coupled with N alike ETCs has been carried out by incorporating different parameters
150 like exergoeconomic, enviroeconomic, energy metrics, productivity and efficacies. However, no
151 researcher in the world has worked on an experimental study of SEBWP integrating ETCs. The
152 system under study is different from the system reported by Singh et al. (2013), Kumar et al.
153 (2014) Sampatkumar et al. (2013) and Issa and Chang (2017) in the sense that they used
154 evacuated tubes; whereas, the ETC in the present study consists of U shaped copper tubes
155 inserted in evacuated tubes. Further, experimental study is a must of any renewable system as it
156 helps in realization of particular technology/system. Hence, experimental study of SEBWP
157 integrated with evacuated tubular collectors has been carried out and reported in this research
158 paper. The main objectives can be stated as follows:

- 159 i. Experimental validation of theoretical results with experimental values for SEBWP of
160 single slope type integrated with N alike ETCs for N = 13.

- 161 ii. Cost estimation of producing unit kg of fresh water, productivity and exergoeconomic
162 parameter for SEBWP of single slope type integrated with ETCs on the basis of
163 experimental data for $N = 13$.
- 164 iii. Evaluation of energy metrics and enviroeconomic parameter of SEBWP integrated
165 with ETCs taking experimentally collected data as basis.



166
167

168 Fig. 1: Experimental setup of SEBWP of single slope type integrated with evacuated tubular
169 collectors

170 **2. Experimental setup of SEBWP of single slope type integrated with N alike ETCs**

171 The specification of SEBWP of single slope type integrated with N alike ETCs has been revealed
172 as Table 1. Fig. 1 represents the experimental setup of SEBWP of single slope type integrated

173 with evacuated tubular collectors. It consists of series connected evacuated tubular collectors,
174 pump and single slope type SEBWP. The experimental setup incorporates series connected
175 evacuated tubular collectors (13 in number) to SEBWP of single slope type with the help of
176 pump. Pump gets its power from grid for its working. Collectors are connected in series with the
177 help of insulated pipe, the output of last collector is connected to basin through insulated pipe
178 and input to the first collector has also been taken through insulated pipe from pump which takes
179 water from basin through insulated pipe. One collector has a surface area of 0.0864 m^2 hence the
180 total surface area of the sequence of evacuated tubular collectors is 1.1232 m^2 . The evacuated
181 tubular collector consists of two concentric cylinders made up of glass and vacuum is provided
182 between these two concentric glass cylinders which prevents heat loss by convection. So, heat
183 loss is lower in this collector in comparison to other collectors like flat plate collector and
184 compound parabolic concentrator collector where heat loss takes place by convection also. The
185 inner glass cover's inner surface is painted black to serve as an absorber. A copper U-tube has
186 been inserted inside the inner glass cylinder. Copper tube has been taken due to its high thermal
187 conductivity property.

188 The evacuated tubular collectors are connected in series to a single slope type SEBWP basin
189 with $2 \text{ m} \times 1 \text{ m}$ (2 m^2) basin area. It was fabricated using galvanized iron (GI) sheet. The inside
190 surface of GI sheet was painted black to absorb solar radiation. The outer surface was covered
191 with glass wool and thermocol. The top surface of single slope type SEBWP was covered with
192 glass having angle of inclination as 15° as the setup was designed for summer season viewpoint.
193 The glass was fixed with help of iron clamp and rubber placed in between iron frame and glass.
194 The sealing was done using window-putty with an aim to avoid seepage of vapor.

195

196 Table 1: Specifications of solar energy based water purifier (SEBWP) of single slope type by
 197 incorporating N number of series connected evacuated tubular collectors (ETCs)

SEBWP of single slope type			
Component	Specification	Component	Specification
Length	2 m	Cover material	Glass
Width	1 m	Orientation	South
Inclination of glass cover	15°	Thickness of glass cover	0.004 m
Height of smaller side	0.14 m	K_g	0.816 W/m-K
Material of body	GI Sheet	Thickness of insulation	0.1 m
Material of stand	GI	K_i	0.166 W/m-K
ETC			
Component	Specification	Component	Specification
Type and no. of collectors	ETC , 13	α_p	0.8
DC motor rating	12 V, 24 W	F'	0.968
Radius of inner copper tube	0.0125 m	τ_g	0.95
Thickness of copper tube	0.0005 m	$K_g (Wm^{-1}K^{-1})$	1.09
Outer radius of outer glass tube of evacuated coaxial glass tube	0.024 m	Angle of ETC with horizontal	30°
Inner radius of inner glass tube of evacuated coaxial glass tube	0.0165 m	Length of each copper tube	1.8 m
Thickness of outer/inner glass tube of evacuated coaxial glass tube	0.002 m		

198

199 The short wavelength solar radiation reaches the water surface after passing through the
 200 condensing cover where a part of energy is reflected by water and the remainder is transmitted to
 201 the basin liner after being absorbed by water. The basin liner transmits the absorbed energy to
 202 the water as it is insulated from outside, and loss of heat is not possible to outside. The

203 temperature of the water increases and within the solar still the heat transfer from the water
204 surface to condensing cover takes place via convection, radiation and evaporation. Water vapor
205 condenses at the inside surface of the cover after losing latent heat of condensation, and film
206 wise condensation is ensured by careful cleaning of the surface so that condensate can be
207 collected as it will trickle down due to the component of gravity force. Drop wise condensation
208 has negative effect on the performance of solar still as it will not allow the solar radiation to pass
209 through it i.e., it will act as opaque surface to incoming solar radiation. The heat accumulated at
210 the condensing glass surface is dissipated to the surrounding by means of convection and
211 radiation and it strongly depends on the wind speed or water flow rate if additional arrangement
212 is made to dissipate it in the form of water flow over condensing cover over a certain time
213 period. A bottom opening has also been created to allow the sediments to be flushed out after a
214 period of time. Digital thermocouples were used to measure the different temperatures.

215

216 **3. Instrumentation**

217 Measuring instruments were used for the measurement of different parameters. The velocity of
218 air blowing was measured using the digital anemometer model of LUTRON AM-4201. Solar
219 radiation was assessed on an hourly basis with the aid of a Solarimeter with a minimum count of
220 20 W/m^2 . The various temperatures were measured using digital thermocouple. The calibrated
221 mercury thermometer was used for the measurement of atmospheric temperature. The
222 measurement of distilled water was done using measuring flask.

223

224

225 Table 2: Variation of different parameters of SEBWP of single slope type integrated with
 226 evacuated tubular collectors at 0.09 m water depth for 29 April 2019

Time (h)	I_c (W/m ²)	I_s (W/m ²)	V_a (m/s)	T_a (°C)	T_w (°C)	T_{gi} (°C)	Yield (kg/h)
8:00	420	440	3.40	30.0	34.0	32.0	0.000
9:00	600	640	1.50	32.0	39.3	33.5	0.085
10:00	780	820	1.20	33.0	45.6	34.7	0.098
11:00	900	920	2.20	34.6	52.2	40.4	0.380
0:00	960	980	1.70	36.0	60.5	46.5	0.687
13:00	980	1020	0.90	37.5	69.2	56.4	0.982
14:00	960	1000	1.00	39.5	78.2	65.5	1.125
15:00	880	920	1.00	39.0	83.4	73.3	1.458
16:00	760	780	2.10	39.4	89.7	78.5	1.615
17:00	600	600	1.20	37.0	91.4	81.3	1.760
18:00	320	340	1.60	36.8	85.2	77.2	1.268
19:00	0	0	1.30	35.3	80.4	70.3	0.988
20:00	0	0	1.10	35.0	76.7	63.4	0.790
21:00	0	0	0.00	34.0	70.3	58.4	0.578
22:00	0	0	0.90	34.2	68.2	55.5	0.415
23:00	0	0	0.30	28.5	65.4	50.4	0.400
24:00	0	0	0.00	27.2	61.2	48.5	0.320
1:00	0	0	0.80	27.4	60.4	47.2	0.225
2:00	0	0	0.90	26.3	59.4	44.5	0.215
3:00	0	0	1.00	25.4	54.9	42.5	0.200
4:00	0	0	1.20	26.2	52.5	40.3	0.200
5:00	0	0	1.30	26.5	51.6	39.8	0.200
6:00	0	0	2.20	25.6	50.3	37.5	0.115
7:00	0	0	1.80	28.4	48.5	36.5	0.100

227

228 **4. Methodology**

229 The experiment was carried out on the roof of Galgotias College of Engineering and
 230 Technology's Mechanical Block in Greater Noida, Uttar Pradesh, India. The data for the typical
 231 day (April 29, 2019) have been presented as Table 2. The basin of single slope type SEBWP was
 232 filled with underground water 24 h prior to the starting of the experimentation for establishing a
 233 steady state condition prior to the start of the experiment. The experiment began at 8 A.M. local

234 time and lasted until 7 A.M. the next day. Solarimeter was used to measure the solar intensity on
235 the surface of the SEBWP and the evacuated solar collector. The data for the different
236 parameters were collected for 24 h. Different parameters for which data were collected are as
237 follows:

- 238 i. Basin water temperature
- 239 ii. Inner surface glass temperature
- 240 iii. Global radiation falling on the surface of SEBWP and collector
- 241 iv. Temperature of blowing air
- 242 v. Distillate output on per hour basis

243 The observation on the hourly basis has been presented as Table 2.

244 **5. Thermal modeling**

245 The thermal modeling of SEBWP of single slope types by incorporating N alike ETCs involves
246 the writing of equation taking energy balancing as the base for all parts of the system followed
247 by simplification. The objective of simplification of equations obtained from balancing energy is
248 to express the unidentified parameters in terms of known parameters like solar intensity,
249 atmospheric temperature and constants. The water temperature, inner condensing glass cover
250 temperature and fresh water yield on hourly basis are developed as a function of solar intensity,
251 ambient temperature and heat transfer coefficients. When writing energy balance equations, the
252 following assumptions are made to simplify the complex situation:

- 253 i. The vapor leakage in SEBWP is neglected.
- 254 ii. Solar distiller unit's water depth is constant. The change in distilled water yield is
255 very small when the water depth changes thus change in depth can be neglected.

- 256 iii. The brackish water held in the basin does not develop layers.
- 257 iv. The heat capacity of the bottom and side insulating material along with condensing
- 258 glass cover is neglected.
- 259 v. The condensation with film type characteristic occurs at inside plane of condensing
- 260 cover. Careful cleaning of the inner surface of the glass ensures film-wise
- 261 condensation and by providing small angle to the condensing cover favors it. The
- 262 component of gravity force along the condensing cover will allow the condensate to
- 263 trickle down along the surface and finally collected in measuring jar.
- 264 vi. All evacuated collectors are identical.

265 Following Singh et al. (2017), development of expression for the temperature from the last

266 collector and thermal energy addition to the basin water is done by energy balancing for receiver

267 surface and water flowing in the copper tubes.

268 **5.1 For evacuated tubular collectors**

269 **5.1.1 For the absorber surface**

$$270 \quad \alpha\tau^2 I(t) 2R dx = [F' h_{pf}(T_p - T_f) + U_{tpa}(T_p - T_a)] 2R dx \quad (1)$$

271 Where F' denotes collector efficiency factor.

272 **5.1.2 For fluid flowing through tube**

$$273 \quad \dot{m}_f C_f \frac{dT_f}{dx} dx = F' h_{pf}(T_p - T_f) 2\pi r dx \quad (2)$$

274 Where r = Radius of copper tube.

275 Using equations (1) and (2), the water temperature at the first collector's outlet can be expressed

276 as

$$T_{fo1} = \frac{(A F_R(\alpha\tau))_1}{\dot{m}_f C_f} I(t) + \frac{(A F_R U_L)_1}{\dot{m}_f C_f} T_a + K_k^N T_{fi} \quad (3)$$

Where, the value of T_{fi} is equal to T_w .

The temperature at the first collector's outlet will be the same as the temperature at the second collector's inlet, the temperature at the second collector's outlet will be the same as the temperature at the third collector's inlet, and so on. Using this condition, the fluid temperature at the Nth collector's outlet can be calculated as follows:

$$T_{foN} = \frac{(A F_R(\alpha\tau))_1 (1-K_k^N)}{\dot{m}_f C_f (1-K_k)} I(t) + \frac{(A F_R U_L)_1 (1-K_k^N)}{\dot{m}_f C_f (1-K_k)} T_a + K_k^N T_{fi} \quad (4)$$

The heated fluid (water) available at the outlet of Nth collectors allowed to basin of SEBWP of single slope type and hence, $T_{wo} = T_{foN}$. After getting the fluid temperature at the outlet of Nth collector, one can obtain the expression for useful heat gain as

$$\dot{Q}_{uN} = \dot{m}_f C_f (T_{foN} - T_{fi}) = \frac{(1-K_k^N)}{(1-K_k)} (A F_R(\alpha\tau))_1 I(t) + \frac{(1-K_k^N)}{(1-K_k)} (A F_R U_L)_1 (T_{fi} - T_a) \quad (5)$$

5.2 For SEBWP of single slope type

5.2.1 For inside surface of condensing glass cover

$$\alpha_g I_S(t) A_g + h_{1w} (T_w - T_{gi}) A_b = \frac{K_g}{L_g} (T_{gi} - T_{go}) A_g \quad (6)$$

Here, $\alpha_g = (1 - R_g) \alpha_g$ denotes the effective absorptivity of glass cover and $h_{1w} = h_{rwg} + h_{cwg} + h_{ewg}$ represents the rate of net heat transfer coefficient between water surface and inner surface of the glass cover.

Outer surface of condensing glass cover:

$$\frac{K_g}{L_g} (T_{gi} - T_{go}) A_g = h_{1g} (T_{go} - T_a) A_g \quad (7)$$

Where, $h_{1g} = h_{rg} + h_{cg}$ or $h_{1g} = 5.7 + 3.8V$

297 **Water mass in basin:**

$$298 \quad \dot{Q}_{uN} + \alpha'_w I_S(t) A_b + h_{bw}(T_b - T_w) A_b = h_{1w}(T_w - T_{gi}) A_b + M_w C_w \frac{dT_w}{dt} \quad (8)$$

299 Where, $\alpha'_w = (1 - R_g)(1 - \alpha_g)(1 - R_w)\alpha_w$ which denotes the effective absorptivity of water
300 mass and \dot{Q}_{uN} denotes useful heat gain per hour basis from N same evacuated tubular collectors
301 connected in series.

302 **Basin liner:**

$$303 \quad \alpha'_b I_S(t) A_b = h_{bw}(T_b - T_w) A_b + h_{ba}(T_b - T_a) A_b \quad (9)$$

304 Where, $\alpha'_b = (1 - R_g)(1 - \alpha_g)(1 - R_w)(1 - \alpha_w)\alpha_b$ = The fraction of solar flux absorbed by
305 basin liner.

306 Appendix-A contains the expressions for the different unknown terms used in equations (3) to
307 (6). The first order differential equation of water temperature (T_w) for N-ETC-SS can be obtained
308 using equation (1) and equations (3) to (6) as mentioned:

$$309 \quad \frac{dT_w}{dt} + a_1 T_w = f_1(t) \quad (10)$$

310 Appendix-A contains the expressions for for a_1 and $f_1(t)$ used in equation (7). The solution
311 to differential equation (7) is written as

$$312 \quad T_w = \frac{\bar{f}_1(t)}{a_1} (1 - e^{-a_1 t}) + T_{w0} e^{-a_1 t} \quad (11)$$

313 Where, T_{w0} is the temperature of water at $t = 0$ and during the time interval 0-t, the average
314 value of $f(t)$ can be expressed as $\bar{f}(t)$. After computing the value of T_w with the help of equation
315 (8), one can evaluate values of glass temperature (T_{gi} and T_{go}) using equations (3) and (4) as
316 follows.

$$317 \quad T_{gi} = \frac{\alpha'_g I_S(t) A_g + h_{1w} T_w A_b + U_{c,ga} T_a A_g}{U_{c,ga} A_g + h_{1w} A_b} \quad (12)$$

318
$$T_{go} = \frac{\frac{K_g T_{gi} + h_{1g} T_a}{L_g}}{\frac{K_g}{L_g} + h_{1g}} \quad (13)$$

319 After estimating parameters namely water temperature (T_w) and glass temperature, the hourly
 320 yield (\dot{m}_{ew}) can be estimated as:

321
$$\dot{m}_{ew} = \frac{h_{ewg} A_b (T_w - T_{gi})}{L} \times 3600 \quad (14)$$

322 The value of L can be estimated using the relationship provided by Fernandez and Chargoy
 323 (1990) and Toyama (1972).

324 6. Analysis

325 6.1 Statistical analysis

326 The rapport between values based on theoretical as well as experimental analyses of different
 327 parameters (T_w , T_g and potable water production) can be determined by calculating the coefficient
 328 of correlation (r_1) and the root mean square percent deviation (e). The value of KARL
 329 PEARSON'S coefficient of correlation (r_1) can be estimated as:

330
$$r_1 = \frac{\sum(x_i - \bar{x})(y_i - \bar{y})}{\sqrt{\sum(x_i - \bar{x})^2 \sum(y_i - \bar{y})^2}} \quad (15)$$

331 The value of e can be estimated as:

332
$$e = \sqrt{\frac{\sum \left[\frac{(x_i - y_i)}{x_i} \right]^2}{N_0}} \quad (16)$$

333 The coefficient of determination can be evaluated as the square of correlation coefficient (r_1^2). It
 334 measures how well the model replicates experiential values (Chapra and Canale 1989, Nakara
 335 Chaudhary 2004). Furthermore, the experiment's internal uncertainty has been estimated.

336 Measurement uncertainty is influenced by both fixed and random errors. The value of standard
 337 uncertainty can be computed as (Bell 1999):

$$338 \quad U_I = \sqrt{\sum_{t=1}^{N_s} \sqrt{\frac{\sigma_t^2}{N_o - 1}}} \quad (17)$$

339 Here, σ denotes the standard deviation and it can be computed as

$$340 \quad \sigma = \sqrt{\sum_{t=1}^{N_o} (X_t - \bar{X})^2} \quad (18)$$

341 N_s denotes the number of sets and N_o denotes the total number of observations. The value of
 342 percentage uncertainty can be computed as

$$343 \quad \text{Percentage uncertainty} = \frac{U_I}{\text{Average of total number of observations}} \times 100 \quad (19)$$

344 **6.2 Uniform end of year annual cost (UEOYAC), cost of potable water (COPW) and** 345 **productivity analyses:**

346 **6.2.1 UEOYAC analysis**

347 The value of UEOYAC for SEBWP of single slope type integrated with N alike ETCs can be
 348 estimated as (Tiwari 2002):

$$349 \quad UEOYAC = PC \times CRF + MC \times CRF - SV \times SFF \quad (20)$$

350 Where PC, SV, CRF, SFF and MC stand for present cost, salvage value, capital recovery factor,
 351 sinking fund factor and maintenance cost in that order. The value of MC may be estimated as the
 352 multiplication of PC with maintenance cost factor that is normally considered as 0.1. The value
 353 of CRF which is used for converting PC into UEOYAC and can be expressed as:

354
$$CRF = \frac{i \times (1+i)^n}{(1+i)^n - 1} \quad (21)$$

355 and SFF can be written as

356
$$SFF = \frac{i}{(1+i)^n - 1} \quad (22)$$

357 SFF is applied for converting SV into UEOYAC. In this case, i and n stand for the rate of interest
358 and system life, respectively.

359 The value of PC for a SEBWP of single slope type integrated with ETCs with a 30-year life span
360 can be calculated as

361
$$PC = PI + P_u + \frac{P_u}{(1+i)^{10}} + \frac{P_u}{(1+i)^{20}} \quad (23)$$

362 Where, $PI = \text{Cost of solar still} + \text{Cost of ETCs} + \text{Fabrication cost}$ (24)

363 The cost of fabrication also involves piping and labor. Values of UEOYAC have been evaluated
364 using equation (20) and they have been presented in Table 5. The required capital investment has
365 been presented as Table 4.

366 **6.2.2 COPW analysis**

367 The cost of obtaining per kg of fresh water from SEBWP of single slope type integrated with
368 ETCs can be written as

369
$$COPW = \frac{UEOYAC}{\text{Annual yield}} \quad (25)$$

370 Values of COPW have been estimated using equation (25) and they have been presented in Table
371 5.

372 **.6.2.3 Productivity analysis**

373 Productivity gives the relation between output and input and it is different from efficiency in the
374 sense that the value of productivity should always be more than 100% whereas the value of

375 efficiency should be less than 100%. Higher the productivity better will be the living standard of
 376 persons because higher productivity means more products are available for use. It is also
 377 expressed as the ratio of effectiveness and efficiency. The value of annual productivity for
 378 SEBWP of single slope type integrated with ETCs can be estimated as (Ashcroft 1950, Benson
 379 1952, Cox 1951, International Labor Office 1979):

$$380 \text{ Productivity} = \frac{\text{Output from SEBWP integrated with ETCs}}{\text{Input provided to SEBWP integrated with ETCs}} \times 100 \quad (26)$$

381 Here, output from SEBWP of single slope type integrated with ETCs represents the annual fresh
 382 water produced from the system. This output can be expressed in terms of rupees by multiplying
 383 the amount of annual fresh water in kg with unit cost (Rs./kg) of fresh water sold in the market.
 384 Hence, output from SEBWP of single slope type integrated with ETCs in terms of Rs. can be
 385 written as

$$386 \text{ Output from SEBWP integrated with ETCs} = (\text{Annual yield}) \times (\text{Selling price}) \quad (27)$$

387 Input provided to SEBWP of single slope type integrated with ETCs will be UEOYAC and it can
 388 be estimated using equation (20). The productivity has been evaluated using equation (26) and
 389 has been presented in Table 5.

390 **6.3 Energy metrics analysis**

391 The energy metrics analysis is an essential part of solar energy technology because the
 392 application of solar energy technology is not justifiable if energy produced by solar energy based
 393 system during the entire life span is less than the value of embodied energy of the solar system. It
 394 involves the calculation of energy payback time (T_{EPB}), life cycle conversion efficiency (η_{LCC}) &
 395 energy production factor (F_{EP}). Energy metrics offers the performance of the system over a

396 longer period of time. The embodied energy encompasses both energy as well exergy; however,
 397 the exergy part is much higher than the energy part. Embodied energy is one of the most
 398 important parameters for the calculation of T_{EPB} . Economics of renewable energy system
 399 involves the selection of low embodied energy for the selected system. It is essential to focus on
 400 the energy densities of all the materials involved in the fabrication of the renewable energy
 401 assisted system to calculate the total embodied energy. Embodied energy of the different
 402 components has been determined by the product of mass of the component with the energy
 403 density of that material. Total embodied energy is calculated by sum of embodied energy for
 404 individual components.

405 **6.3.1 T_{EPB} analysis**

406 The term T_{EPB} for SEBWP of single slope type integrated with ETCs is the time span required to
 407 recover embodied energy which is known as the energy needed for the production of SEBWP of
 408 single slope type integrated with ETCs. The value of T_{EPB} can be calculated taking energy or
 409 exergy as the basis; however, the value of T_{EPB} on the basis of energy is far lower than the
 410 exergy based T_{EPB} because exergy means quality of energy and the amount of exergy produced
 411 by the system is much lower than the energy obtained from the system. A comparatively poorer
 412 value of T_{EPB} is expected as lower value of T_{EPB} means the energy or exergy based breakeven
 413 point will be obtained in lesser time and higher amount of energy will be produced which results
 414 in higher amount of carbon credit. T_{EPB} for SEBWP of single slope type integrated with ETCs on
 415 the basis of energy as well as exergy can be expressed as:

$$416 \quad T_{EPB,energy} = \frac{\text{Embodied energy of SEBWP integrated with ETCs}(E_{in})}{\text{Annual energy output obtained from SEBWP integrated with ETCs}(E_{out})} \quad (28)$$

$$417 \quad T_{EPB,exergy} = \frac{\text{Embodied energy of SEBWP integrated with ETCs}(E_{in})}{\text{Annual exergy output obtained from SEBWP integrated with ETCs}(E_{out})} \quad (29)$$

418 The value of hourly exergy rate can be estimated as follows:

$$419 \quad \text{Hourly exgy} = h_{ewg} \times A \times \left[(T_w - T_{gi}) + (T_a + 273) \ln \left\{ \frac{(T_w + 273)}{(T_{gi} + 273)} \right\} \right] \quad (30)$$

420 **6.3.2 F_{EP} analysis**

421 The term F_{EP} for SEBWP of single slope type integrated with ETCs is defined as the ratio of
 422 Annual energy output obtained from SEBWP of single slope type integrated with ETCs to
 423 energy needed for the production of SEBWP of single slope type integrated with ETCs. Thus,
 424 F_{EP} is the reciprocal of term T_{EPB} that represents the overall performance of the system. The
 425 ideal value on annual basis is 1. Values of EPFF $_{EP}$ for EBWP integrated with ETCs on the basis
 426 of energy as well as exergy can be estimated as:

$$427 \quad F_{EP,energy} = \frac{\text{Annual energy output obtained from SEBWP integrated with ETCs}(E_{out})}{\text{Embodied energy of SEBWP integrated with ETCs}(E_{in})} \quad (31)$$

$$428 \quad F_{EP,exergy} = \frac{\text{Annual energy output obtained from SEBWP integrated with ETCs}(E_{out})}{\text{Embodied exergy of SEBWP integrated with ETCs}(E_{in})} \quad (32)$$

429 **6.3.3 η_{LCC} analysis**

430 The term η_{LCC} gives an idea about net output of SEBWP of single slope type integrated with
 431 ETCs with regard to solar energy impinging the surface of the system for the whole life span of
 432 the system. The ideal value of LCCE for SEBWP of single slope type integrated with ETCs is
 433 one. The system is considered performing better in the value of η_{LCC} is higher. Value of η_{LCC} for
 434 SEBWP of single slope type integrated with ETCs on the basis of energy and exergy can be
 435 estimated as

436
$$\eta_{LCC,exergy} = \frac{(Annual\ energy \times n) - E_{in}}{(Annual\ solar\ exergy) \times n} \quad (33)$$

437
$$\eta_{LCC,exergy} = \frac{(Annual\ exergy \times n) - E_{in}}{(Annual\ solar\ exergy) \times n} \quad (34)$$

438 The variation of monthly solar energy falling on the surface of system has been presented in Fig.
 439 6. By adding monthly exergy for twelve months, value annual solar energy impinging on the
 440 surface can be estimated. Values of η_{LCC} , have been estimated using equations (32) and (33) and
 441 they have been presented in Table 6.

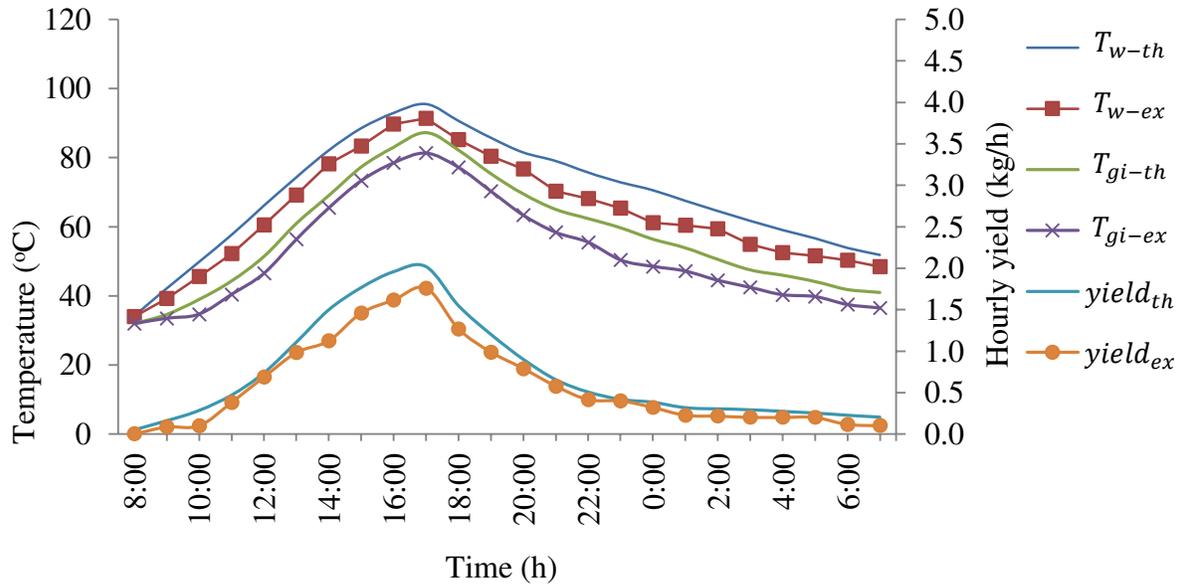
442 **6.4 Exergoeconomic and enviroeconomic analyses of SEBWP of single slope type integrated**
 443 **with ETCs:**

444 **6.4.1 Exergoeconomic analysis of SEBWP of single slope type integrated with ETCs:**

445 The value of exergoeconomic parameter has been estimated using first and second laws of
 446 thermodynamics. This relationship means that the system is constructed in such a way that it
 447 achieves an overall optimum design by efficiently balancing the exergy and economic
 448 parameters. The exergoeconomic parameter relates either exergy loss or exergy gain with
 449 UEYOYAC. In the case of exergy gain, the objective is maximization type, whereas, in the case of
 450 exergy loss, the objective is minimization type. The parameter exergoeconomic can be estimated
 451 as:

452
$$Exergoeconomic\ parameter = \frac{Annual\ exergy\ loss\ for\ SEBWP\ integarted\ with\ ETCs\ (L_{ex,annual})}{UEYOYAC} \quad (35)$$

453 Here, rate of exergy loss for SEBWP can be estimated as



473

474 Fig. 2: Validation of values of T_w , T_{gi} and hourly yield on April 29, 2019 for SEBWP of single
 475 slope type integrated with evacuated tubular collectors

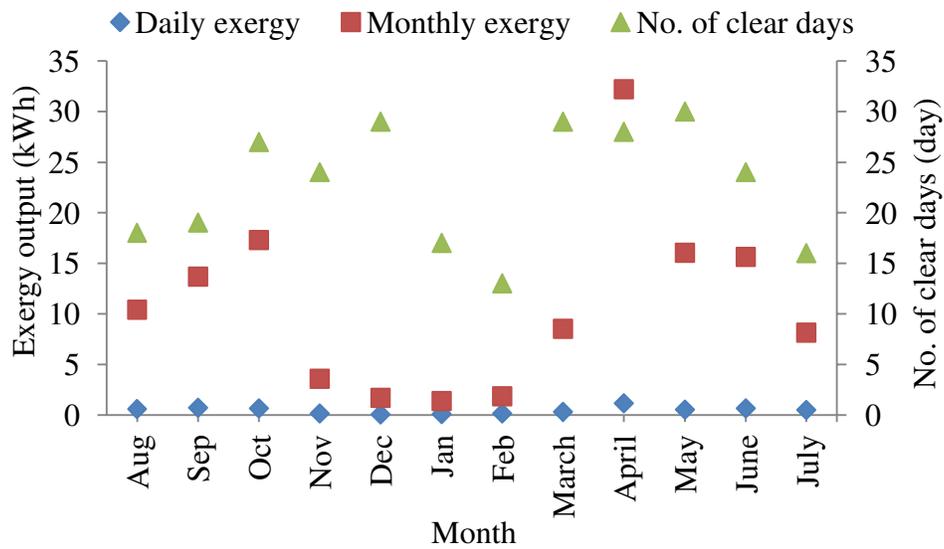
476

477 Table 3: Calculation for annual production of potable water for SEBWP of single slope type
 478 integrated with evacuated tubular collectors

Typical day of month	Number of clear days (day)	Daily yield (Kg)	Monthly yield (Kg)
02-08-18	18	8.95	161.14
28-09-18	19	8.90	169.19
11-10-18	27	9.70	261.98
07-11-18	24	3.54	84.94
16-12-18	29	1.79	51.92
03-01-19	17	2.40	40.80
03-02-19	13	3.15	40.91
19-03-19	29	5.53	160.37
29-04-19	28	14.20	397.71
19-05-19	30	10.16	304.89
08-06-19	24	11.15	267.55
02-07-19	16	10.07	161.12
Annual yield (Kg)			2102.53

479

480 Fig. 2 represents the validation of values of T_w, T_{gi} and hourly yield on April 29, 2019 for
 481 SEBWP of single slope type integrated with ETCs. Values of r and e for T_w, T_{gi} and hourly yield
 482 have been estimated using equations (15) and (16) respectively. It has been found that values of r
 483 varies from 0.9928 to 0.9951 and that of e varies from 8.2 % to 28.53% which show that there is
 484 a fair agreement between theoretically calculated vales and experimentally collected values for
 485 T_w, T_{gi} and hourly yield. Table 3 represents the evaluation of annual yield for SEBWP of single
 486 slope type integrated with ETCs based on experimentally collected values for typical day of each
 487 month.



488
 489 Fig. 3: Variation of monthly exergy output for SEBWP of single slope type integrated with
 490 evacuated tubular collectors

491 Fig. 3 represents the variation of monthly exergy output for SEBWP of single slope type
 492 integrated with evacuated tubular collectors. The hourly exergy output has been estimated using
 493 equation (30) followed by the estimation of daily exergy by summing hourly exergy for 24 h.
 494 The monthly exergy has been estimated by multiplying daily exergy with number of clear days
 495 in that month. It has been found that monthly exergy is maximum for April because of better

496 solar intensity received in the month of April. Further, the value of monthly exergy depends on
 497 daily exergy and number of clear days.

498 Table 4: Capital investment for SEBWP of single slope type integrated with evacuated tubular
 499 collectors

S.N.	Parameter	cost
1	Solar still	12000
2	Copper tube @ 280 per meter	14924
3	Evacuated tube @500each	6500
4	Aluminum stand	3000
5	Iron stand for solar still	1000
6	Motor and pump	2000
7	Fabrication cost	5000
8	Salvage value of the system after 30 years taking inflation rate is 4%	13755.46

500

501 Table 5: Calculation of UEOYAC production cost and productivity for SEBWP of single slope
 502 type integrated with evacuated tubular collectors

Evaluation of UEOYAC								
S.N.	n (Year)	i (%)	PC Rs.	M Rs.	SV Rs.	$F_{CR,i,n}$ (Fraction)	$F_{SR,i,n}$ (Fraction)	UEOYAC Rs.
1	30	2	47410.64	4741.06	13755.46	0.045	0.025	2002.94
2	30	5	46405.61	4640.56	13755.46	0.065	0.015	3111.67
3	30	10	45492.37	4549.24	13755.46	0.106	0.006	5221.88

Evaluation of COPW and Productivity								
S.N.	n (Year)	i (%)	UEOYAC Rs.	AY (kg)	COPW (Rs./kg)	SP Rs.	RE Rs.	Productivity (%)
1	30	2	2002.94	2102.53	0.95	5	10512.65	524.86
2	30	5	3111.67	2102.53	1.48	5	10512.65	337.85
3	30	10	5221.88	2102.53	2.48	5	10512.65	201.32

503

504 The investment in installing SEBWP of single slope type integrated with ETCs has been
 505 presented in Table 4. The cost of different components is the price of products as per local
 506 market. Also, the salvage value has been estimated as per the local market price. UEOYAC for
 507 SEBWP of single slope type has been estimated using equation (20) and they have been

508 presented in Table 5. The life span of SEBWP integrated with ETCs has been taken as 30 years
509 except motor and pump. The life of pump with motor has been taken as 10 years and it has been
510 assumed that the inflation after 10 years can be adjusted with its salvage value. The rate of
511 interest has been considered as 2%, 5% and 10%. The value of UEOYAC is minimum for 2%
512 rate of interest as 2% rate of interest is minimum. Values of COPW and annual productivity for
513 SEBWP of single slope type integrated with ETCs has been estimated using equations (25) and
514 26 respectively and they have been presented in Table 5. It is found that the value of COPW is
515 minimum for 2% rate of interest because UEOYAC is minimum for 2% rate of interest. Further,
516 the value of productivity is maximum for 2% rate of interest as UEOYAC value is minimum for
517 2% rate of interest. Also, productivity is inversely proportional to UEOYAC as evident from
518 equation (26). It has also been observed that the value of productivity is more than 100% for all
519 interest rates under consideration. It means that the system is feasible.

520 Table 6 presents the calculation of embodied energy (E_{in}), energy payback time (T_{EPB}), energy
521 production factor (F_{EP}) and η_{LCC} for SEBWP of single slope type integrated with ETCs. Fig. 4
522 presents the variation of monthly solar energy falling on the surface of SEBWP of single slope
523 type integrated with ETCs. The value of embodied energy has been estimated as the product of
524 energy density (kWh/kg) and mass (kg). The mass of different components has been calculated
525 as the product of density (kg/m^3) and volume (m^3). Values of $T_{EPB,energy}$ and $T_{EPB,exergy}$ has
526 been found to be 1.72 year and 25.9 year respectively. The value of $T_{EPB,energy}$ is lower than
527 $T_{EPB,exergy}$ because exergy represents the quality of energy (high grade energy) and hence lower
528 value of exergy is obtained from SEBWP of single slope type integrated with ETCs. Values of
529 $F_{EP,energy}$ and $F_{EP,exergy}$ have been found to be 0.58 and 0.039. The value of $F_{EP,energy}$ is
530 higher as F_{EP} is the reciprocal of T_{EPB} as evident from equations (31) to (34). Values of

531 $\eta_{LCC,energy}$ is higher than $\eta_{LCC,exergy}$ because exergy is lower than energy as exergy represents
 532 the quality of energy.

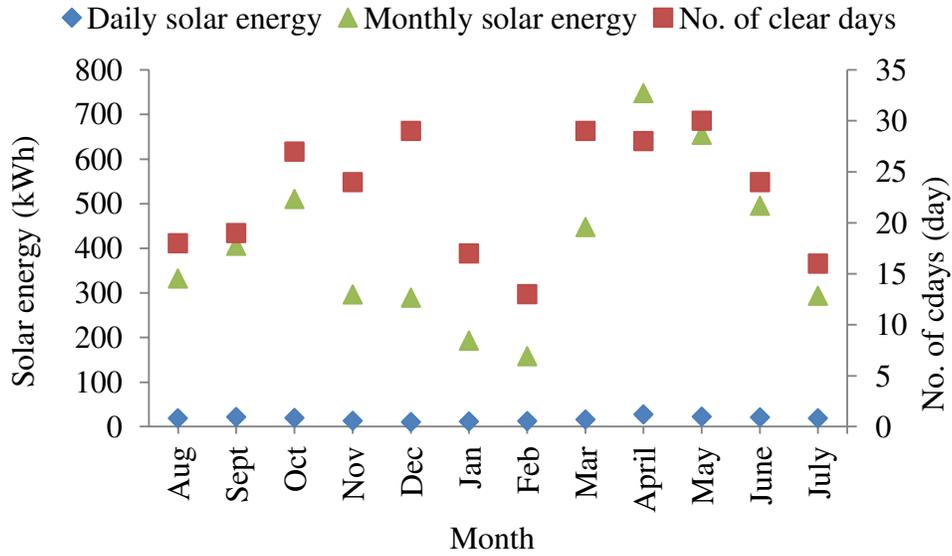
533

534 Table 6: Calculation of embodied energy (E_{in}), energy payback time (T_{EPB}), energy production
 535 factor (F_{EP}) and life cycle conversion efficiency (η_{LCC}) for SEBWP of single slope type
 536 integrated with evacuated tubular collectors

Name of component	Solar energy based water purifier of single slope type integrated with ETCs
	Embodied energy (kWh)
Solar still	706.99
ETC (N=13)	1287.43
Others	20
Single slope PVT-FPC active solar distillation system	
Annual yield = 2102.53 kg	
Total embodied energy = 2014.42 kWh	
Net annual energy available from SEBWP of single slope type integrated with ETCs = 1170.95 kWh	
Net annual exergy available from SEBWP of single slope type integrated with ETCs = 77.76 kWh	
Life of the system(Year) = 30	
The value of T_{EPB} for SEBWP of single slope type integrated with ETCs based on energy (year) = 1.72	
The value of T_{EPB} for SEBWP of single slope type integrated with ETCs based on exergy (year) = 25.90	
The value of F_{EP} for SEBWP of single slope type integrated with ETCs based on energy (per year) = 0.58	
The value of F_{EP} for SEBWP of single slope type integrated with ETCs based on exergy (per year) = 0.039	
Solar energy for life time (E_{sol}) in kWh= 144770.23	
Solar exergy for life time (E_{sol}) in kWh= 134636.31	
The value of η_{LCC} for SEBWP of single slope type integrated with ETCs based on energy (fraction) = 0.23	
The value of η_{LCC} for SEBWP of single slope type integrated with ETCs based on exergy (fraction) = 0.0024	

537

538



539

540 Fig. 4: Variation of monthly solar energy falling on the surface of SEBWP of single slope type
541 integrated with evacuated tubular collectors

542 Table 7: Evaluation of exergoeconomic parameter for SEBWP of single slope type integrated
543 with evacuated tubular collectors

S.N.	n (Year)	i (%)	UEOYAC (Rs.)	Annual Exergy loss (kWh)	Exergoeconomic parameter (kWh/Rs.)
1	30	2	2002.94	669.1278	0.334
2	30	5	3111.669	669.1278	0.215
3	30	10	5221.878	669.1278	0.1281

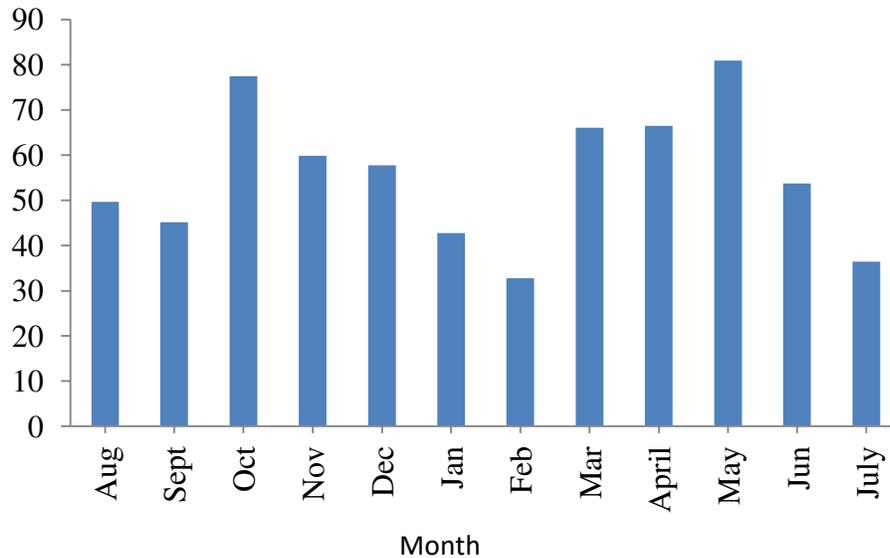
544

545 Table 8: Evaluation of enviroeconomic parameter for SEBWP of single slope type integrated
546 with evacuated tubular collectors

Single slope active solar still	
Life (year)	30
Embodied energy (kWh)	2014.42
Net annual energy available (kWh)	1170.95
Net energy available (kWh) for life time	35128.50
CO ₂ credit (t)	66.22
Environmental cost (Enviroeconomic parameter) (\$)	960.31

547

548 Table 7 presents the estimation of exergoeconomic parameter for SEBWP of single slope type
 549 integrated with ETCs and Fig. 5 presents the variation of monthly exergy loss for SEBWP of
 550 single slope type integrated with ETCs. The value of exergoeconomic parameter has been found
 551 to be minimum for 10% rate of interest because UEOYAC is highest for this interest rate. Table
 552 8 presents the evaluation of enviroeconomic parameter for SEBWP of single slope type
 553 integrated with ETCs. The carbon credit has been estimated as 66.22 t and the corresponding
 554 enviroeconomic parameter has been found to be 960.31\$.



555
 556 Fig. 5: Variation of monthly exergy loss for SEBWP of single slope type integrated with
 557 evacuated tubular collectors

558 **8. Conclusions**

559 The experimental study of SEBWP of single slope type integrated with ETCs has been carried
 560 out and based on the findings of this research; the following conclusions have been drawn:

- 561 i. A fair agreement has been found between experimental and theoretical values of T_w ,
 562 T_{gi} and yield with correlation coefficient varying between 0.9928 and 0.9951.

- 563 ii. COPW values have been found to range from Rs. 0.95 to Rs. 2.48 as interest rates
564 range from 2% to 10%. Values of productivity have been found to be more than 100%
565 which represent that the system is feasible.
- 566 iii. Values of $T_{EPB,energy}$ and $T_{EPB,exergy}$ have been found to be 1.72 years and 25.90
567 years respectively; values of $F_{EP,energy}$ and $F_{EP,exergy}$ have been found to be 0.58
568 per year and 0.039 per year respectively; whereas, values of $\eta_{LCC,energy}$ and
569 $\eta_{LCC,exergy}$ have been found to be 0.23 and 0.0024 respectively.
- 570 iv. The value of exergy loss (kWh) per unit UEYOYAC (Rs.) has been found to vary
571 between 0.13 and 0.33.
- 572 v. The value of enviroeconomic parameter has been found to be 960.31 \$ for the system.

573 Appendix

$$574 (AF_R(\alpha\tau))_1 = PF_1 \alpha \tau^2 A_R F_R; \quad (A F_R U_L)_1 = (1 - K_k) \dot{m}_f C_f;$$

$$575 PF_1 = \frac{h_{pf}}{F' h_{pf} + U_{tpa}} ; \quad U_L = \frac{U_{t,pa} h_{pf}}{F' h_{pf} + U_{t,pa}} ;$$

$$576 F_R = \frac{\dot{m}_f C_f}{U_L A_R} \left[1 - \exp\left(-\frac{2\pi r' L' U_L}{\dot{m}_f C_f}\right) \right] ;$$

$$577 K_k = \left(1 - \frac{A_R F_R U_L}{\dot{m}_f C_f} \right)$$

$$578 h_{pf} = 100 W m^2 K^{-1}$$

$$579 U_{t,pa} = \left[\frac{R_{O2}}{R_{O1} h_i} + \frac{R_{O2} \ln\left(\frac{R_{i2}}{R_{i1}}\right)}{K_g} + \frac{1}{C_{ev}} + \frac{R_{O2} \ln\left(\frac{R_{O2}}{R_{O1}}\right)}{K_g} + \frac{1}{h_o} \right]^{-1}$$

$$580 a_1 = \frac{1}{M_w C_w} \left[\dot{m}_f C_f (1 - K_k^N) + U_s A_b \right] ;$$

$$581 \bar{f}_1(t) = \frac{1}{M_w C_w} \left[\alpha'_{eff} A_b \bar{I}_s(t) + \frac{(1 - K_k^N)}{(1 - K_k)} (AF_R(\alpha\tau))_1 \bar{I}_c(t) + \left(\frac{(1 - K_k^N)}{(1 - K_k)} (AF_R U_L)_1 + U_s A_b \right) \bar{T}_a \right] ;$$

$$582 \alpha'_{eff} = \alpha'_w + h_1 \alpha'_b + h'_1 \alpha'_g ; h_1 = \frac{h_{bw}}{h_{bw} + h_{ba}} ;$$

$$583 \quad h'_1 = \frac{h_{1w}A_g}{U_{c,ga}A_g + h_{1w}A_b} ; h_{1w} = h_{rwg} + h_{cwg} + h_{ewg} ;$$

$$584 \quad h_{ewg} = 16.273 \times 10^{-3} h_{cwg} \left[\frac{P_w - P_{gi}}{T_w - T_{gi}} \right] ;$$

$$585 \quad h_{cwg} = 0.884 \left[(T_w - T_{gi}) + \frac{(P_w - P_{gi})(T_w + 273)}{268.9 \times 10^3 - P_w} \right]^{\frac{1}{3}} ;$$

$$586 \quad P_w = \exp \left[25.317 - \frac{5144}{T_w + 273} \right] ; P_{gi} = \exp \left[25.317 - \frac{5144}{T_{gi} + 273} \right] ;$$

$$587 \quad h_{rwg} = (0.82 \times 5.67 \times 10^{-8}) [(T_w + 273)^2 + (T_{gi} + 273)^2] [T_w + T_{gi} + 546] ;$$

$$588 \quad U_s = U_t + U_b ; U_b = \frac{h_{ba}h_{bw}}{h_{bw} + h_{ba}} ; U_t = \frac{h_{1w}U_{c,ga}A_g}{U_{c,ga}A_g + h_{1w}A_b} ;$$

$$589 \quad U_{c,ga} = \frac{\frac{K_g}{l_g} h_{1g}}{\frac{K_g}{l_g} + h_{1g}} ; h_{ba} = \left[\frac{L_i}{K_i} + \frac{1}{h_{cb} + h_{rb}} \right]^{-1} ;$$

$$590 \quad h_{cb} + h_{rb} = 5.7 \text{ Wm}^{-2}\text{K}^{-1} , \quad h_{bw} = 250 \text{ Wm}^{-2}\text{K}^{-1} ;$$

591

592

593 **Nomenclatures**

594	SEBWP	solar energy based water purifier
595	ETCs	evacuated tubular collectors
596	N	number of evacuated tubular collectors
597	GI	galvanized iron
598	I(t)	solar intensity falling on the surface of collector, W/m ²
599	I _s (t)	solar intensity falling on the surface of SEBWP, W/m ²
600	R	outer radius of glass tube, m
601	F'	collector efficiency factor, fraction
602	h _{pf}	heat transfer coefficient from plate to fluid, W/m ² -K
603	T _p	temperature of absorber plate, °C
604	T _f	temperature of fluid/water, °C

605	T_a	atmospheric temperature, °C
606	U_{tpa}	overall heat transfer coefficient from plate to environment, W/m ² -K
607	\dot{m}_f	mass flow rate, kg/s
608	C_f	specific heat capacity of fluid/water, kJ/kg-K
609	r	radius of copper tube, m
610	T_{fo1}	temperature of fluid at the outlet of first collector, °C
611	T_{fi}	temperature of fluid at the inlet of first collector, °C
612	T_{foN}	temperature of fluid at the outlet of Nth collector, °C
613	\dot{Q}_{uN}	rate of useful heat gain, kWh
614	A_g	area of glass cover, m ²
615	h_{1w}	total heat transfer coefficient from water surface to glass cover, W/m ² -K
616	T_w	temperature of water, °C
617	T_{gi}	temperature at inside surface of glass, °C
618	T_{go}	temperature at outside surface of glass, °C
619	A_b	area of basin liner, m ²
620	K_g	thermal conductivity of glass, W/m-K
621	L_g	thickness of glass cover, m
622	h_{1g}	total heat transfer coefficient from glass surface to ambient, W/m ² -K
623	h_{bw}	heat transfer coefficient from basin liner to water, W/m ² -K
624	T_b	temperature of basin liner, °C
625	M_w	mass of water in basin, kg
626	h_{ba}	heat transfer coefficient between basin liner and ambient, W/m ² -K
627	\dot{m}_{ew}	hourly water yield, kg/h
628	r_1	coefficient of correlation, fraction

629	e	root mean square percent deviation, %
630	r_1^2	coefficient of determination, fraction
631	U_I	standard uncertainty
632	σ	standard deviation
633	UEOYAC	uniform end of year annual cost, Rs.
634	COPW	cost of potable water, Rs./kg
635	PC	present cost, Rs.
636	CRF	capital recovery factor, fraction
637	MC	maintenance cost, Rs.
638	SFF	sinking fund factor, fraction
639	i	interest rate, %
640	n	life of system, year
641	SV	salvage value, Rs.
642	P_u	cost of pump, Rs.
643	T_{EPB}	energy payback time, Year
644	F_{EP}	energy production factor, per year
645	η_{LCC}	life cycle conversion efficiency, fraction
646	E_{in}	embodied energy, kWh
647	h_{cwc}	convective heat transfer coefficients from water surface to inside surface of
648		condensing cover, W/m ² -K
649	h_{rwc}	radiative heat transfer coefficients from water surface to inside surface of
650		condensing cover, W/m ² -K
651	T_{wf}	final temperature of water, °C
652	T_{wi}	initial temperature of water, °C
653		

654	CRP	carbon dioxide reduction price, \$
655	R_{i1}	inner radius of inner cylindrical glass tube, m
656	R_{i2}	outer radius of inner cylindrical glass tube, m
657	R_{o1}	inner radius of outer cylindrical glass tube, m
658	R_{o2}	outer radius of outer cylindrical glass tube, m
659	α	absorptivity
660	τ	transmissivity
661	α'_g	effective absorptivity of glass
662	α'_w	effective absorptivity of water
663	α'_b	effective absorptivity of basin liner

664

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790

791 **Declaration of competing Interest**

792 The authors declare that there is no competing interest

793 **Availability of data and materials**

794 All data are given in the manuscript

795 **Ethics approval and consent to participate**

796 Not applicable

797 **Consent for publication**

798 Not applicable

799 **Funding**

800 There is no funding received for the research work carried out

801 **Authors Contribution**

802 Sanjeev Kumar Sharma - Writing-review & editing

803 Ashis Mallick – Writing, Formal analysis,

804 Desh Bandhu Singh- Data curation, Project administration, Software, review & editing

805 Gopal Nath Tiwari - Review & editing

Figures



Figure 1

Experimental setup of SEBWP of single slope type integrated with evacuated tubular collectors

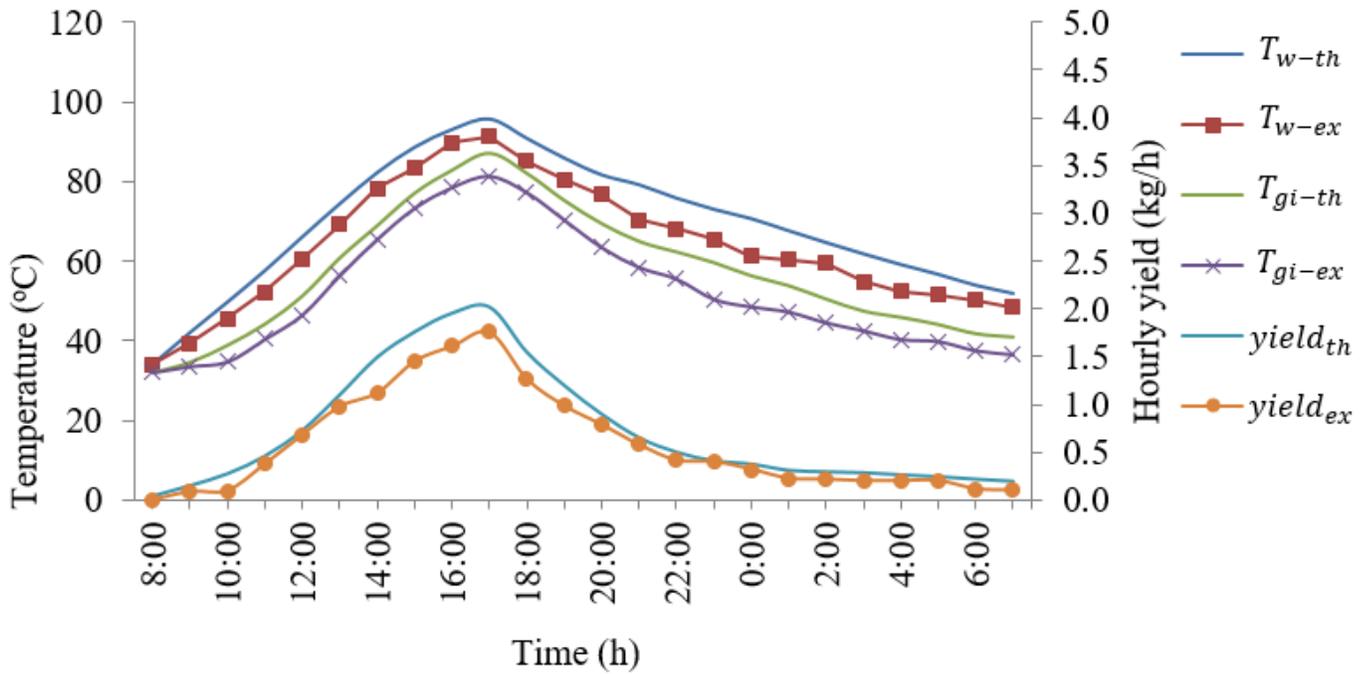


Figure 2

Validation of values of T_w , T_{gi} and hourly yield on April 29, 2019 for SEBWP of single slope type integrated with evacuated tubular collectors

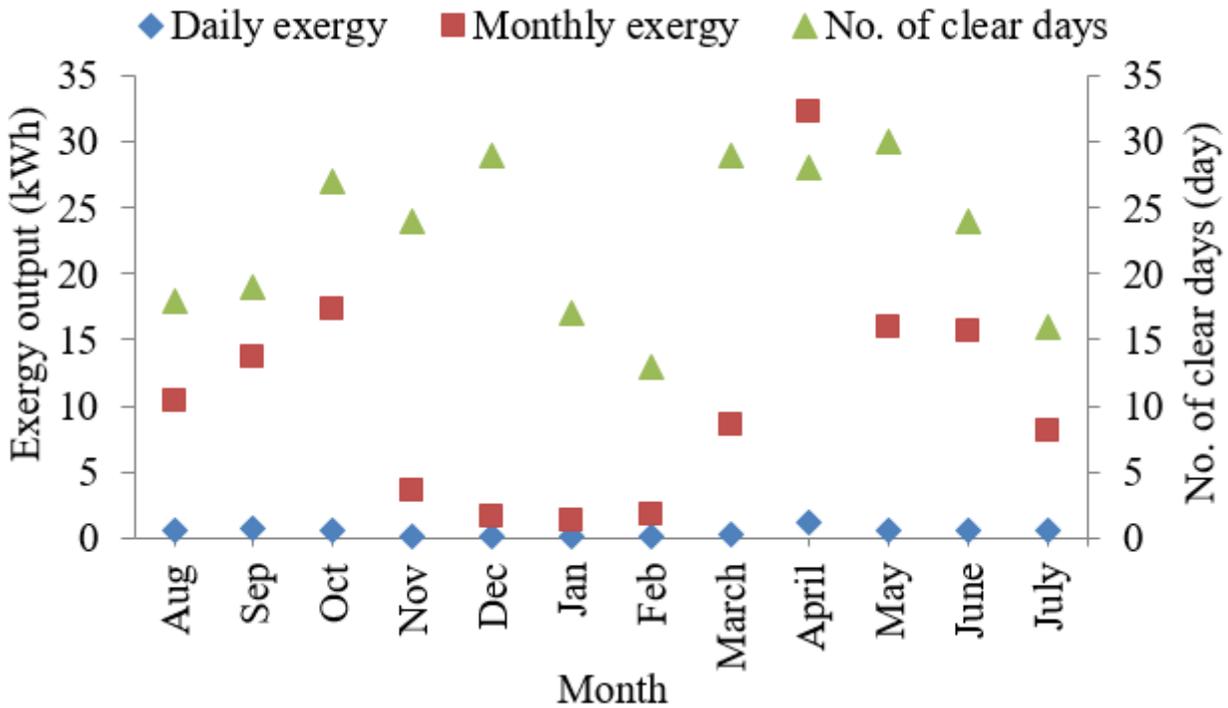


Figure 3

Variation of monthly exergy output for SEBWP of single slope type integrated with evacuated tubular collectors

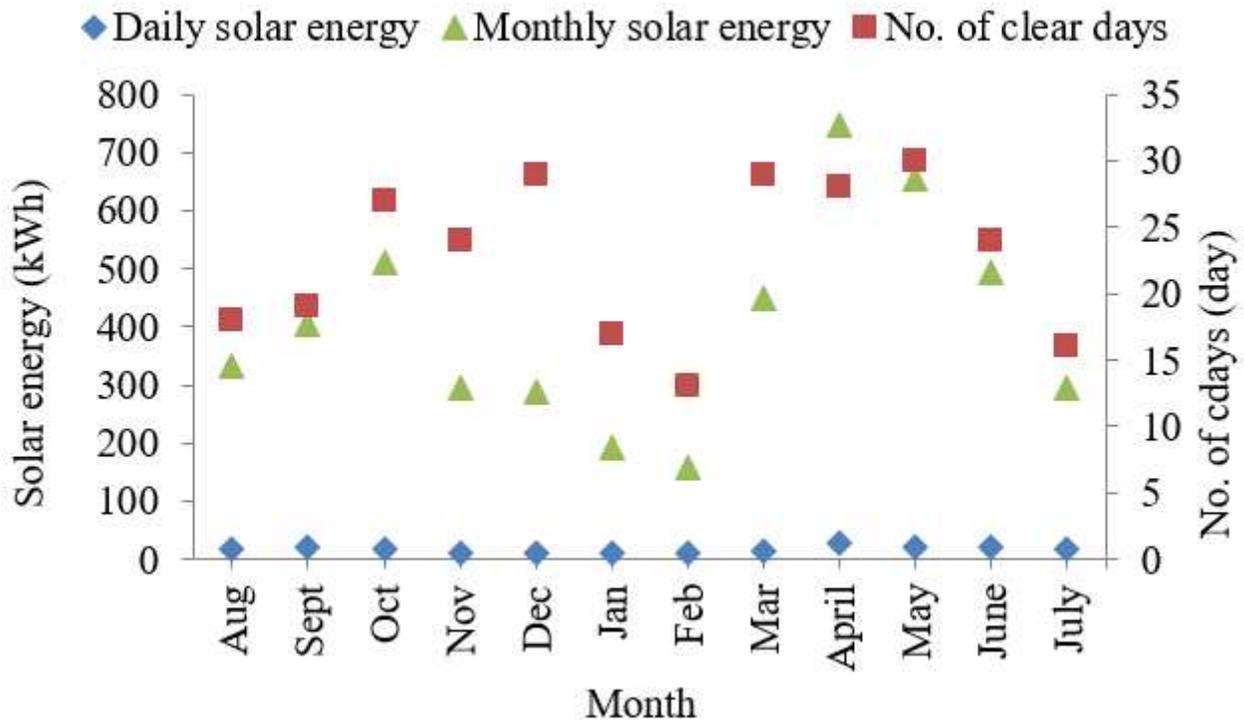


Figure 4

Variation of monthly solar energy falling on the surface of SEBWP of single slope type integrated with evacuated tubular collectors

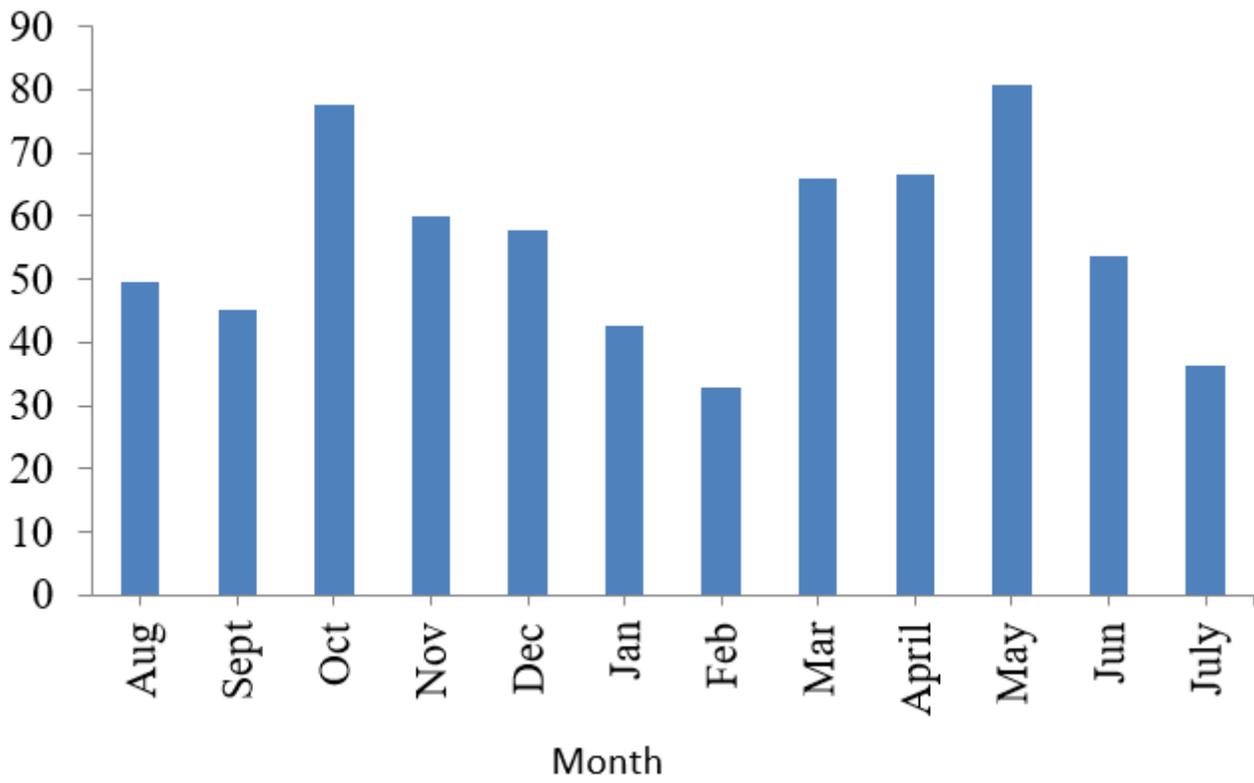


Figure 5

Variation of monthly exergy loss for SEBWP of single slope type integrated with evacuated tubular collectors