

Effect of Injection Timing In Reducing The Harmful Pollutants Emitted From CI Engine Using N-Butanol Antioxidant Blended Eco-Friendly Mahua Biodiesel

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Abstract

Consuming non-renewable energy sources has genuine and long-standing unfavorable consequences on people's health, neighborhood networks and ecosystems, and also on the worldwide atmosphere. The burgeoning demand and use of diesel engines in various fields cause emission of exhaust gases like NO_x and CO that lead to serious environmental pollution and hazards like global warming, respiratory problems, and so on, has necessitated a reduction in the use of diesel and addition of suitable biodiesel. Mahua biodiesel blend has also been considered as a safe renewable fuel for conventional engines. This is due to its desirable properties such as rapid growth rate, higher productivity, and the ability to utilize CO₂ into fuel. The introduction of an antioxidant, preferably n-butanol eliminates these harmful emissions from the diesel engine. In this experimental investigation, mahua biodiesel blend mixed with n-butanol has been used as test fuel in a conventional engine. Because of the special character n-butanol has been chosen for use with the mahua biodiesel blend. About 20–30 vol.% of n-butanol has been blended with 80 vol.% of diesel and tested. The manufacturer set injection timing was 23°CA bTDC. The injection timing is preferably between 21°CA bTDC and 25°CA bTDC. Nitrogen oxides and carbon monoxide are reduced by 49 percent and 5.88 percent, respectively, when a blend of B20 + D80 + 30% n-butanol is used at 21°CA bTDC relative to diesel fuel. The smoke and hydrocarbon emission of blend B20 + D80 + 30% n-butanol at 25°CA bTDC is reduced by 40% and 38.07%, respectively, related to diesel. The brake thermal efficiency for entire injection timing has been identified to be increased correlated to all other test blends. The brake thermal efficiency in blend B20 + D80 + 30% n-butanol of 25°CA bTDC is increased by 15.30%, when compared with diesel fuel. These promising results assure that mahua biodiesel blend containing antioxidant would be eco-friendly fuel.

I. Introduction

Experts are scrambling to find suitable replacement energy options, which have grown in importance due to rising energy needs and pollution. [1]. A few environmental issues emerge because of the utilization energy sources in IC engines. Biodiesel is a fuel delivered from vegetable oils utilizing a couple of catalysts. Biodiesel is an alternative solution to the problem faced by most developing countries [2]. Biodiesel has been select by many kinds of research as far as maintainability, financial, and environmentally friendly behavior [3]. To produce sustainable biodiesel on a large scale, it is imperative to understand the advantage of utilizing minimal effort non-edible oil that can cut down the expense of delivering biodiesel [4]. Rudolf diesel, with his engine powered by peanut oil, wrote, "The use of vegetable oil in diesel engine fuel may appear to be inconsequential now, but in the future it might turn out to be just as essential as conventional oil." [5]. It is notable that non-renewable energy sources are effectively approachable in a few locations around the globe. Several researchers have been making systematic effort to utilize plant oils as engine fuel. This will also lessen the need for fossil fuel and emission of harmful wastes into the atmosphere, because they are organic and emissions-less. The raw vegetable oils have the highest significance and exist as a promising substitute for fossil fuels [6]. They have high viscosity, lower heat capacity, and are well known for gum formation, auto oxidation, and lower engine

durability. Without any changes to the engine, the use of these oils may lead to poor performance and engine damage. Contamination forms and adheres to piston rings due to the direct usage of vegetable oil. It will be necessary to put in place some preparatory activities in order to make vegetable oils a viable fuel. Several methods are available for making vegetable oils usable, Among these, transesterification is the most popular method of extracting biodiesel.

In a previous study, the palm and jatropha biodiesel are extracted from the transesterification process. The combination of palm biodiesel and jatropha biodiesel each 5% when mixed with 90% of diesel and palm biodiesel and jatropha biodiesel each 10% when mixed with 80% of diesel gave better performance when compared with other tested fuel. These two fuels blend shows slightly increased brake specific fuel consumption (BSFC) then correlated with diesel. Apart from this the emission characteristics have been reduced mainly because of more injection of fuel through the inlet manifold. The oxides of nitrogen (NO_x) and hydrocarbon (HC) emission for blend of palm and jatropha biodiesel each 5% mixed with 90% of diesel were decreased by 9.53% and 3.69%, respectively, when used in conjunction with mineral diesel fuel. The carbon monoxide (CO) and carbon dioxide (CO₂) of blend palm and jatropha biodiesel each 5% mixed with 90% of diesel were decreased by 20.49% and 5.69%, respectively, when used in conjunction with mineral diesel fuel due to the proper combustion and shorten ignition delay [7].

The rape seed oil was separated by transesterification process and mixed with 80% and 90% of base fuel and fuel mixes were prepared, such as B10 and B20, respectively, at a different speed to run the base diesel engine. At 1800 rpm the mix B10 demonstrated the diminished BSFC when contrasted with the diesel fuel due to the lesser heating value and improved the ignition process. The emission of CO and HC were cut down when the fuel blends of B10 were used at 75% of load condition at 3000 rpm, owing to the oxygen concentration of biodiesel and appropriate ignition timing. This is mainly due to proper blending of air–fuel through the ignition procedure [8].

The rapeseed oil and mahua oil were mixed equally and a new biodiesel was prepared and this biodiesel has been mixed with the various proportion of diesel having varying percentage of 20%, 40%, 60%, and 80%, respectively. The blend B20 gave the superior performance and this value near to the base fuel diesel. The brake thermal efficiency (BTE) was decreased by 2.79%, when used in conjunction with mineral base fuel due to the inferior CV and greater viscosity. The emission of CO, smoke opacity, and HC were decreased by 20.66%, 6.9%, and 8.56%, respectively, individually when contrasted with perfect diesel fuel owing to the full combustion and shortened ignition delay. The concentrations of NO_x were found to be more by 3.71% with B20, when contrasted with slick diesel fuel due to higher temperatures produced obtained during the combustion process [9].

In this investigation, the sunflower and soybean oil were mixed 50% equally and new biodiesel has been prepared. It is mixed with various proportions of diesel preferably 10%, 20%, and 30%, respectively. The blends showed slight decrease in cylinder pressure and HRR by 8.1% and 7.2%, respectively, when compared with neat diesel fuel, because of the lesser CV and lower ID. The CO concentration was decreased by 33.8% when contrasted with flawless diesel fuel due to better combustion and lesser IT.

When compared to pure diesel fuel, the HC emission was shown to be reduced using the B30 fuel mix. The NO_x and BSFC were increased by 0.98%, 2.5%, and 11.43% with the fuel blend B30 when compared with the neat base fuel. It is owing to the higher ignition delay and lack of oxygen availability of the biodiesel [10].

Nowadays, many studies have been done focusing on biodiesel with some antioxidant agents like BHT, BHA, n-butanol, n-octanol, and so on, to improve performance and decrease emission attributes of the diesel engine. When using the antioxidants there is a drastic decrease in NO_x and a slight increase in BTE.

In this investigation, *Calophyllum inophyllum* was chosen as the biodiesel, mixed with antioxidants, preferably BHT 500 ppm of dosage, and nanoadditives of titanium oxide of 100 ppm dosage and was used in the direct ignition diesel engine. The inclusion of nanoparticles and BHT into the fuel showed a slight increase in BTE by 4% and 2%, respectively, then correlated to the neat diesel fuel due to the oxygen availability in the nanoparticles and higher the density of the fuel. The CLME with BHT of fuel blend showed an increase in emission of HC and CO by 35% and 12.76%, respectively, when compared with pure C100 due to the availability of free radical surface and decreased oxidation of HC. The NO_x concentration was drastically reduced by 11.85% when adding the antioxidants (BHT) with the fuel, when compared with other tested fuels due to the oxidation during the combustion process [11].

In this study mahua oil extracted by transesterification process was mixed with the antioxidants (BHT) in various proportions, such as 5%, 10%, and 15%, and was then used for investigation. The BTE was found to be slightly increased by 3.42% with the blends B85, when contrasted with the raw diesel fuel. The blend B85 showed a decrease in HC and smoke opacity by 37.63% and 2.65%, respectively, when contrasted with the mineral fuel, because of the greater oxygen content available in the fuel and proper combustion. The concentration of NO_x and EGT were found to increase in the B85 blend by 45.87% and 27.33%, respectively, when contrasted with diesel fuel, this was mainly because of the high temperature produced during the combustion process [12].

The coconut biodiesel extracted by the transesterification method is mixed with the antioxidants such as BHT and BHA at a dosage of 2000 ppm and used for investigation. The results showed a significant decrease in NO_x and a slight increase in BTE. Addition of BHA in B20 blend showed a drastic decrease in NO_x by 7.78%, when correlated with the base fuel, because of phenolic hydroxyl present in the antioxidant and oxygen availability in the biodiesel. The blend B20 with BHA showed a decreased emission of CO and smoke opacity by 18.39% and 32.43%, respectively, when correlated with neat base fuel mainly due to the oxygen availability and also due to an increase in the C–C bond. The B20 blend containing BHT showed the highest HC emission of 27.65%, when correlated with diesel fuel, mainly due to the free radical formation [13].

In another study, the Annona biodiesel and three different antioxidant agents like p-phenylenediamine (PPDA), alpha-tocopherol acetate (AT) and l-ascorbic acid (LA) were investigated. The antioxidants were

used in a different dosage of 50, 150, 250, 350, and 450 mg and mixed with biodiesel. It was found that all the antioxidants when mixed with the biodiesel showed slight decrease in emission of NO_x. Among all the dosage of antioxidants, 250 mg showed a significant decrease in NO_x by 24.7%, 22%, and 23.8%, respectively, when compared with base fuel, and this is mainly due to the greater cetane number and lesser ignition delay and oxygen availability in the biodiesel. Finally, the results concluded that biodiesel containing 250 mg of PPDA showed drastic reduction in NO_x, when correlated with the other two antioxidants [14].

The researchers are also focusing on diesel engine parameters like injection timing, injection pressure, and so on. When using antioxidants mixed with B20 during the injection timing there was a significant decrease in NO_x improvement in the performance of the engine, and also improvement in combustion, when compared with the normal base engine.

In another study, the *Syzygium cumini* oil containing biodiesel was used and experiments were performed at different injection timing and pressure. The blends tested were B30, B70, and B100 to operate the diesel engine. The blend B30 and diesel increased the BTE by 16.68% and 17.85%, respectively, when the injection timing was 21°CA bTDC. When the injection timing was increased, the emission of HC and CO in the blend B30 was decreased by 46.15% and 15.9%, respectively, when compared with other test fuels. In advanced injection timing greatly reduced the smoke opacity emission by 28.7% in B30 blend, when compared with other test fuels. During the advanced injection timing NO_x emission was high, when compared with the normal diesel engine. There was a remarkable reduction in emission of harmful wastes when the injection timing was increased [15].

In another study, the idea was to vary injection timing of the base engine to reduce the NO_x emission. By delaying injection to 21°CA bTDC from the usual 24°CA bTDC, the hybrid biofuel (jatropha oil and rubber seed oil) was utilized in a single-cylinder base engine. Along with pure diesel fuel, B20 (biodiesel-20 percent), B40 (biodiesel-40 percent), and B60 (biodiesel-60 percent) mixes were utilized. By using this hybrid biofuel, this study article demonstrated the effect of injection time modification in a base engine. The experiments examined the performance and emission characteristics of BSFC, NO_x, CO, and unburned hydrocarbon (UHC). It was found from the results that SFC for B20 blend was lesser than for raw base fuel, while B40 and B60 blends had slightly increased values but were similar to the B20 blend. It was also seen that the CO and UHC emissions were decreased by increasing biodiesel blends in the fuel mixture, but NO_x emissions were higher by increasing biodiesel blends in the fuel mixture [16].

For their study, the authors used mineral oil, gasoline, biodiesels, and various alternative fuels. When the diesel fuel exhibited improved injection timing was seen that BTE was greater with diminished fuel consumption and minimal HC and CO emissions. However, NO_x emission was found to be higher. Retarded injection time resulted in a reduction in cylinder pressure, which further led to reduced peak temperature resulting in decreased NO_x emission. During the retarded injection timing, the fuel injection happened slowly because of which the duration of combustion and cylinder peak pressure were lowered. Due to this reason fuel efficiency was also reduced. On the contrary, advanced injection timing enhanced

the combustion process and the fuel efficiency was also higher. It was also found that oxidation capacity was increased with increased cylinder temperature. It further resulted in a decreased O₂, carbon (C), CO, and HC emissions and EGT. Normally, diesel–biodiesel fuel blends gave higher HC and CO emissions. However, in the case of retard injection timing the NO_x emissions were minimized [17].

In this study, the diesel is mixed with n-octanol with various percentages of 10%, 20%, and 30%, respectively. The experiments also used EGR at a rate of 10%, 15%, and 20% with injection timing advanced and retarded conditions. The best position of brake thermal efficiency is obtained by diesel with a blend of 10% n-octanol in 10% EGR at 23°C bTDC. There is the simultaneous reduction of smoke, NO_x, and CO of blend diesel with 10% n-octanol + 10% EGR in advanced injection timing of 47.77%, 21.08%, and 18.76%, respectively, due to the antioxidants agents its react as a catalyst more oxygen content available in the antioxidants proper combustion takes places and shorten ignition delay [18].

The novelty of work is clear from the above literature review that the various additives added to biodiesel led to improved efficiency and decreased harmful environmental emissions. The modification of the injection timing and the introduction of the fuel additives resulted in the diesel engine with superior output and emission characteristics. The present work was conducted with various injection timings such as 21°C bTDC for retardation, 23°C bTDC for standard and 25°C bTDC for advanced. The mahua biodiesel is mixed with the diesel and additives, they are prepared the three blends for investigation of M100 (mahua raw oil), M20 (mahua oil 20 vol.% + diesel 80 vol.%) and NBM (mahua oil 20 vol.% + diesel 80 vol.% + n-butanol 30 vol.%). These blends are used in the current investigation with varying injection timing (advancing and retardation) and then compared with mineral diesel fuel.

Table 7 shows that the comparison of various ignition timing for different fuel blends from the previous literature survey. The table uses two arrow symbols, one is upward arrows and the other one is downward arrow. The upward arrow symbol indicates increased performance and emission characteristics while the downward arrow symbol indicates decreased performance and emission characteristics.

Table 8 shows the comparison of different antioxidants used in diesel engines with different fuels. The table uses two arrow symbols, one is upward arrow and the other one is downward arrow. The upward arrow symbol indicates increased performance and emission characteristics and downward arrow symbol indicates decreased performance and emission characteristics.

Ii. Outline Of Mahua Biodiesel

Two types of mahua variety namely *Madhuca longifolia* and *Madhuca indica* are seen in India, especially in the dry lands and wastelands. The seeds are commonly known as Indian spread tree seeds. The specific gravity of mahua raw oil is 9.11%, that is, higher than base diesel fuel. The specific gravity of mahua oil was 15.23 times more than the base fuel at 40°C. The specific gravity of mahua oil diminished

fundamentally with increase in temperature to 80°C. By extending the degree of diesel in fuel blends. The seeds contain between 30% – 40% greasy oil called mahua oil, which is attractive and is used in the production of a variety of products, for example, chemicals and glycerin. The oil cake is used as biomanure, characteristic compost, and as feed for fish and steers. The leaves are used as feed and as green manure. The blooms are used for evacuating ethanol, which is used in making country liquor. Figure 1 depicts the fatty acid concentrations of mahua biodiesel.

Production of Mahua Biodiesel

Figure 2 shows the oil extraction process of mahua oil (up to 50% oil) from mahua seeds. The seeds are first broken and flaked during the industrialized extraction, and the resultant particles are then oven-cooked. At 63°C the cooked flakes are broken and extracted using hexane as solvent. The resulting meal of mahua oil contained less than 1% of oil. When the crops are crushed in smallholder fields, they yield an energy-dense mahua seed cake containing up to 17% oil. The natural morphology of the mahua oil plants and their seeds are the accessibility and ignition attributes like CV, cetane number, octane number, flash point, fire point, viscosity, and density. It easily mixes with base fuel. The utilization of mahua oil as a base fuel as alternative in the compression ignition engine has now increased and showed more prominent and significant results with greater population and sensational development rate. The properties of the fatty acid composition in the mahua oil are shown in Table 1.

Table 1
Fatty acid composition for mahua biodiesel

Fatty Acid	Structure Number	Structure Formula	Weight (%)
Compounds of palmitic acid	16.2	$C_{16}H_{32}O_2$	24.8
Compounds of stearic acid	18.4	$C_{18}H_{36}O_2$	22.5
Compounds of arachidic acid	20.7	$C_{20}H_{40}O_2$	1.5
Compounds of oleic acid	18.1	$C_{18}H_{34}O_2$	37.5
Compounds of linoleic acid	18.2	$C_{18}H_{32}O_2$	14.3

iii. Preparation Of Fuels

Figure 3 shows the fuel preparation chart for mahua biodiesel. Biodiesel is extracted from the mahua plant, using the method of transesterification. The biodiesel density and viscosity are very similar to that of base fuel. The introduction of antioxidant (n-butanol) is to eliminate the harmful gaseous emission in the base engine. About 30% of antioxidant (n-butanol) is mixed with the 20% of mahua biodiesel and 80% of base fuel by stirring. This study was conducted with various injection timings like 21°CA bTDC for retardation, 23° CA bTDC for standard, and 25°CA bTDC for advanced. The mahua biodiesel is mixed with diesel and additives. Three blends have been prepared for investigation, and they are M100 (mahua

raw oil), M20 (20 vol.% mahua oil + 80 vol.% diesel), and NBM (20 vol.% of mahua oil + 80 vol.% of diesel + 30 vol.% of n-butanol). The findings of the tests were compared to those obtained from petroleum diesel fuels. Table 2 depicts the properties of various fuels used in this study.

Table 2
Properties of fuels

Property	Diesel	n-butanol	M20	M100	NBM
Density kg/m ³	835	812	862	912	872
Viscosity at 40°C mm ² /sec	3.1	3.8	3.6	4.1	3.7
Flash point °C	56–58	35–37	59–62	61–68	54–58
Calorific value MJ/kg	48	17	38	37	41
Oxygen (wt.%)	0	12.3	2.5	3.1	14.5
Low heating value MJ/kg	44.45	37.9	42.12	41.45	40.12

Iv. Engine Set-up

The Table 3 below demonstrates the basic engine's specs. The experimental arrangement comprises a single-cylinder, water-cooled in DI diesel engine. It can produce 4.4 kW at a base engine speed of 1500 rev per min and engine is coupled with dynamometer. The engine entry side consists of an anti-pulsing drum, a fresh air heater, and a device for measuring entry temperature. The tail pipe of the base engine consists of EGT devices, a tail pipe gas analyzer, and a smoke meter. The testing equipment also comprises a different fuel estimation measuring device to measure the utilization of mahua biodiesel blends. The test rig is fitted with a 64-bit data acquisition (DAQ) framework to gain crank angle and cylinder pressure data. The parameters for the combustion are found using the data systems. The AVL 444 flue gas analyzer is utilized to assess the visibility of the smoke. In this system, the uncertainty analysis is performed. Figure 4 shows the experimental set-up.

Table 3
Engine specification

Parameter	Specification
Type of engine	Kirloskar TV-1
Stroke	Four
Cylinders	Single
Bore (mm)	110
Compression ratio	17.50
Maximum engine power (kW)	5.20 kW/7 HP@1500 rpm
Fuel type	Diesel
Starting	Hand start
Injection	Direct
Coolant	Water-cooled
Maximum engine speed	1500 rpm
Engine volume	0.661

V. Analysis

a. Uncertainty Analysis

All measurement equipment must undergo a precision uncertainty analysis. It was sourced from a variety of manufacturers. Uncertainty inquiry was used to rectify errors that occurred as a result of environmental circumstances, worker considerations, and assessment. Normal characteristics were determined by re-directing each examination several times to ensure the accuracy of the findings. We noticed the levels of the errors and the quantities of experiments that were compatible with the empirical errors that happened throughout the analytical technique. The uncertainty analysis is summarized in Table 4.

Table 4
Results of uncertainty analysis.

No.	Parameters	Systematic Errors (\pm)
1	Speed, rpm	± 1
2	Load, N	± 0.2
3	Time, seconds	± 0.1
4	Brake Power, kW	± 0.5
5	Temperature, °C	± 1
6	Pressure, bar	± 1
7	NO _x , ppm	± 9
8	CO, %	± 0.03
9	CO ₂ , %	± 0.03
10	Unburnt hydrocarbon, ppm	± 10
11	Smoke, HSU	± 1

b. FTIR Analysis

At the SRM institute of technology in Chennai, Fourier transform infrared (FTIR) spectroscopy study was performed. The FTIR analysis was performed in this study using a Nicolet, ThermoScientific model IS 10 equipment. The machine's range was determined to be between 3885.85 and 467.13 cm⁻¹. The study of the presence and ranges of several utility bands was finished in FTIR. Calculating the utility and vibrations of biodiesel allows for an estimation of its quality and unsaturated fat methyl ester content. FTIR research may also be used to evaluate samples as small as 10 m. The smaller sample sizes enable the identification of particles, fibers, residues, and films. The degrees of oxidation or fixation of some polymers may be determined using FTIR analysis, which is comparable to the estimation of impurities or additional chemicals. The FTIR analysis of mahua biodiesel is shown in Fig. 5. Table 5 illustrates the results of an FTIR study of the band structure.

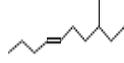
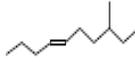
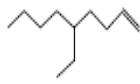
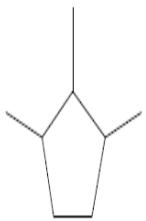
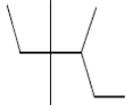
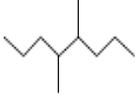
Table 5
FTIR analysis of band structure

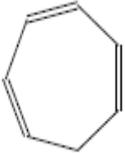
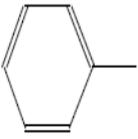
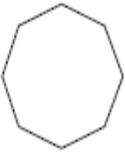
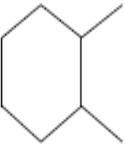
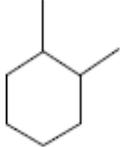
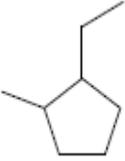
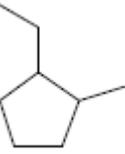
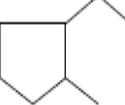
No	Range of the Value	Band Value	Bond Structure
1	4123.03	4300 - 3900	HC
2	3431.45	3900 - 3200	NH
3	3145.34	3200 - 2920	NH
4	2645.23	2920 - 2530	CH
5	2312.67	2530 - 2210	CH
6	2198.78	2210 - 1970	CH
7	1865.23	1970 - 1724	OH
8	1523.67	1724 - 1430	CO
9	1320.13	1430 - 1210	CO
10	989.34	1210 - 934	CO
11	778.91	934 - 710	CBr
12	487.12	710 - 420	Cl

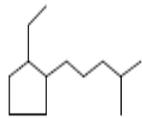
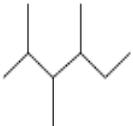
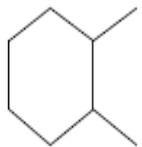
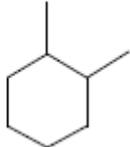
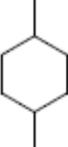
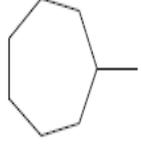
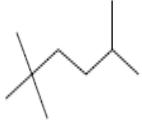
c. GC-MS Analysis

The full transesterification of triglycerides into methyl ester fatty acid was investigated using a gas chromatograph-mass spectrometer (GC-MS) study and the results were shown in Fig. 6. Thermo GC-MS Trace Ultra hardware and software version 6.0 were used in this study to conduct GC-MS analysis, which was combined with data acquired from the 2.0 framework. The methyl ester fatty acid exemplified the feasibility of using biodiesel as an alternative to non-renewable energy sources. In this study, 1 mL of biodiesel was run through a Perkin Elmer GC-MS section, and nitrogen was used as the transporter gas at a flow rate of 1.0 mL/min. The pinnacles obtained for the test were identified by the biodiesel's contrasting n-hexane standard. The GC-MS of aromatic compounds were shown in the Table 6.

Table 6: GC-MS of aromatic compounds

S.No	Retention Time	Component Name	Area (%)	Chemical Structure	Chemical Formula
1	1.2	4-DECENE, 8-METHYL-,	3.54		C ₁₁ H ₂₂
2.	1.4	4-DECENE, 8-METHYL-	4.34		C ₁₁ H ₂₂
3.	1.6	4-DECENE, 8-METHYL-	6.87		C ₁₁ H ₂₂
4.	1.8	5-ETHYL-1-NONENE	7.67		C ₁₁ H ₂₂
5.	2.0	1-OCTENE, 3,7-DIMETHYL-	7.98		C ₁₀ H ₂₀
6.	2.2	CYCLOPENTANE	8.43		C ₅ H ₁₀
7.	2.4	HEXANE, 3,3,4-TRIMETHYL-	8.89		C ₉ H ₂₀
8.	2,6	OCTANE, 4,5-DIMETHYL	9.34		C ₁₀ H ₂₂
9.	2.8	HEPTANE, 3,3-DIMETHYL-	9.68		C ₉ H ₂₀
10.	3.0	2-HEPTANONE, 5-METHYL-	9.94		C ₈ H ₁₆ O
11.	3.2	1,3,5-CYCLOHEPTATRIENE	10.22		C ₇ H ₈

					
12.	3.4	:1,5-HEPTADIEN-3-YNE	10.49		C7H8
13.	3.6	BENZENE, METHYL-	10.95		C7H8
14.	3.8	:1,5-HEPTADIEN-3-YNE	11.21		C7H8
15.	4.0	CYCLOOCTANE	11.68		C8H16
16.	4.2	CYCLOHEXANE, 1,2-DIMETHYL-, CIS-	11.94		C8H16
17.	4.4	CYCLOHEXANE, 1,2-DIMETHYL-, TRANS-	12.24		C8H16
18.	4.6	CYCLOPENTANE, 1-ETHYL-2- METHYL-	12.54		C8H16
19.	4.8	CYCLOPENTANE, 1-ETHYL-2- METHYL-, CIS-	12.95		C8H16
20.	5.0	COMPNAME:CYCLOPENTANE, 1- ETHYL-2-METHYL-	13.12		C8H16
21.	5.2	:1-ETHYL-2-(4- METHYLPENTYL)CYCLOPENTANE	13.54		C13H26

					
22.	5.4	HEXANE, 2,3,4-TRIMETHYL-	13.97		:C9H20
23.	5.6	OCTANE, 4-METHYL-	14.12		C9H20
24.	5.8	CYCLOHEXANE, 1,2-DIMETHYL-, CIS	14.54		C8H16
25.	6.0	CYCLOHEXANE, 1,2-DIMETHYL-, TRANS-	14.86		C8H16
26.	6.2	CYCLOHEXANE, 1,4-DIMETHYL-, CIS-	15.14		C8H16
27.	6.4	CYCLOHEPTANE, METHYL-	15.42		C8H16
28.	6.6	HEXANE, 2,2,5-TRIMETHYL-	15.85		C9H20
29.	6.8	HEXANE, 2,2,5-TRIMETHYL	16.12		C9H20

Vi. Results And Discussion

A. Performance Characteristics

a. NO_x Emissions

Figure 7 illustrates the variation in NO_x emission of the tested fuel under full load conditions. In particular, the diesel engine produces lower nitrogen oxides at the starting load level, and it increases at full load level. Nitrogen oxides rise slowly and eventually correspond to neat base fuel owing to the enhanced oxygen availability in the fuel due to the addition of n-butanol antioxidants. [27]. NBM showed reduced nitrogen oxides compared with pure diesel fuel, M100, and other tested fuel mixtures at part load and full load status of oxide of nitrogen producing results at 342 and 1054 ppm, respectively. It is due to the change in temperature induced by vaporization of n-butanol which reduced the latent heat and greater oxygen availability, which caused high peak temperature during improved combustion [28]. The NBM 23^oCA bTDC blend showed decreased NO_x content by 28.34% then correlated with the neat base fuel. The NDM 25^oCA bTDC blend showed slightly increased emission by 5.38% when compared with the diesel fuel. The NBM (21^oCA bTDC) showed a 32.17% decline in NO_x emission relative to diesel fuel attributable to the anti-oxidant involvement of oxidative amine reaction preventing peroxy-free radical formations. These peroxy free radical formations form the focal point of higher emissions of NO_x. A free radical is the oxidizing agent of molecules that determines the oxidation reaction rate [29].

b. Smoke Opacity

Figure 8 demonstrates the diversity in smoke intensity and all tested fuels concerning full load conditions. The smoke opacity emission in clean biodiesel is high as compared with all measured fuels. The mahua biodiesel had higher toxicity for smoke in previous studies. Since n-butanol antioxidants are blended with mahua biodiesel and diesel fuel, the intensity of smoke decreased, because of the ascent in fuel oxygen in the mixes and the accessibility of oxygen content even in fuel-rich regions. The smoke emission of M20 and NBM at 21^oCA bTDC is slightly decreased by 2.5% and 1.25%, respectively, when correlated with neat base fuel and raw oil is increased in the smoke emission. The reason may be attributed to the higher cetane amount of diesel and the ignition delay period. The smoke opacity of NBM with injection timing 23^oCA bTDC and 25^oCA bTDC was seen to be decreased by 32.5% and 40%, respectively, when correlated with neat pure base fuel. It produces low smoke emissions as equated with all the other high engine load fuels. The explanation for low emission is due to engine operated with optimum 25^oCA bTDC injection timing, which improved the dwelling time for fuel and air blending leading to clean and assured combustion. Such combined results gradually diminish the release of smoke with respect to the timing of advance injection [30].

c. CO Emissions

Figure 9 demonstrates the emission of CO and all measured fuels under the full load condition. Emission of CO is seen throughout the working of diesel engine due to inadequate combustion produce by the unavailability of the oxygen. If the CO emission of the engine is increased and the n-butanol additives are added to the fuel, the low CO is emitted throughout the diesel engine [31]. The blend NBM (21^oCA bTDC) produces 29.41% (high-level of CO) related to diesel fuels; this is attributable to little combustion duration and need of arrangement time in the combustion bowl. When the blend of NBM with injection timings, 23^oCA bTDC and 25^oCA bTDC is used, it is found that CO emission has been reduced by 8.6% and

16.53%, respectively, when correlated with the neat base fuel. Due to the introduction of n-butanol additives and high temperatures at high loads levels, the blend NBM (250CA bTDC) has low emission of CO (0.14%), relative to diesel fuels. Prolonged delay time boosts fuel spraying pattern and atomization and thus increased burning occurring in the container [23].

d. HC Emissions

Figure 10 demonstrates the diverse in the emission of HC and all the measured fuels concerning the full load condition. Considering the proximity of unsaturated HC which is strong during most of the combustion cycle, the HC emission is usually high for neat biodiesel than diesel fuel [32]. Owing to inadequate burning and lower delay time, the blend NBM (21⁰CA bTDC) provided 10.31% higher level of HC emission, when correlated with neat base fuel resulting in poor atomization. Standard diesel reports low HC emissions at peak load and normal timing. Compared with standard diesel fuel, the blend NBM (25⁰CA bTDC) offered low HC emission of 38.12%, when correlated with the base fuel due to advanced ignition timing and quicker ignition delay[33]. It might be due to improvement in the evaporation cycle and further enhancement during the preparing of the air–fuel mixture, contributing to clean combustion eventually [34].

B. Performance Characteristics

a. BTE

The BTE variations with respect to load are shown in Figure 11 with the contradiction effect of different injection timings with M100, M20, and NBM fuel. The lower thermal efficiency of the raw mahua biodiesel was then contrasted with all other tested fuels, as mahua displayed a reduction in performance since additional energy is required to breakdown huge HC chain and substantial aromatic substances affecting mahua biodiesel [35]. Compared with all other measured fuels, the brake thermal performance was increased by including the additives in the fuels. It is associated to retarding the timing of fuel injection in the engine at 210CA bTDC, 230CA bTDC, and 250CA bTDC. There was an inadequate period for air–fuel blending to delay the fuel injection and inadequate combustion was noted [36]. The increase in BTE of NBM with injection timings, 210 CA bTDC, 230CA bTDC, and 250CA bTDC were found to be improved by 3.53%, 10.12%, and 15.56% then correlated with neat base fuel. That was due to the ample residence time in the cylinder for preparing the air–fuel mixture with a longer combustion period [37].

b. Brake Specific Energy Consumption

Figure 12 illustrates the pattern for the brake specific energy consumption (BSEC) variance with load for different injection timings and with M100, M20, and NBM (21⁰CA bTDC) fuels. The calorific value plays a significant part in this investigation. Generic diesel fuel at standard 23⁰CA bTDC injection time demonstrated low BSEC on a par with all other fuels [38]. This can be ascribed as equated with all other fuels to a higher fuel heating benefit. NBM (21⁰CA bTDC) has higher BSEC levels. It is due to the retarded injection pressure of fuel at 21⁰CA bTDC. Although the retardation cycle had a greater combustion

temperature and higher pressure, it does not have enough time for full burning [26]. Therefore the more quantity of fuel was consumed during most of the retardation cycle. The NBM (250CA bTDC) has fewer BSEC compared with NBM (210CA bTDC) and NBM (230CA bTDC). It is due to the efficient use of fuel by keeping the cycle of burning very close to fuel, which would increase the evaporation in the tank, hence lower BSEC was observed [39].

C. Combustion Characteristics

a. Cylinder Pressure

Deviations between the cylinder pressures are based on various crank angles for NBM, M100, and M20 fuels followed by diesel fuel at NBM 23⁰CA bTDC, NBM 21⁰CA bTDC, and NBM 23⁰CA bTDC have been shown in Figure 13 respectively. Relative to all other fuels, the NBM 21⁰CA bTDC showed low combustion pressure peaks.

$$dQ_c = du + w + dQ_h \quad (1.1)$$

$$(dQ_c)/dt = P dv/dt + mc_v dT/dt + (dQ_h)/dt \quad (1.2)$$

$$(dQ_c)/dt = P dv/dt + mc_v d/dt [PV/mR] \quad (1.3)$$

$$(dQ_c)/dt = P dv/dt + c_v/R d/dt [P dv/dt + V dp/dt] \quad (1.4)$$

$$c_v/R = 1/(\gamma - 1) \quad (1.5)$$

$$(dQ_c)/dt = \gamma/(\gamma - 1) P dv/dt + 1/(\gamma - 1) V dp/dt \quad (1.6)$$

The factors for the low cylinder pressure are due to poor atomization and less time of preparation at lower injection timing levels. At NBM 25⁰CA bTDC fuel gives inflated cylinder pressure on par with all the injection timings and fuels [40]. It is because of enhanced air-to-fuel mixing, evaporation rate and ignition delay due to 25⁰CA bTDC IT. The diesel fuel at 23⁰CA bTDC slightly increases the cylinder pressure when compared with NBM at 21⁰CA bTDC. Due to variations in injection timing and properties, such as heating value and cetane number, the pressure was found to increase. To conclude, NBM 25⁰CA bTDC can be recommended for the diesel engine powered by NBM [41].

Vii. Conclusions

The experimental investigation attempted a novel approach of using mahua biodiesel with n-butanol antioxidant fuel in DI diesel engine. The following are the inferences of the experiments:

NBM (21⁰CA bTDC) showed an appreciable reduction (32.17%) of NO_x, when correlated with neat base fuel. The reaction with aromatic amines effectively stopped the formation of peroxy free radicals.

The smoke emissions were reduced by 15.56% when the blends NBM 25^oCA bTDC were used when correlated to neat fuel at full load conditions. Because of oxygen availability in the fuel, decrease in dwelling time during the combustion process is possible.

The CO emission was found to be decreased by 8.6% and 16.53% when NBM with injection timings 23^oCA bTDC and 25^oCA bTDC, this is because of the effect of antioxidants and shorten ignition delay.

A significant reduction of hydrocarbons was observed when NBM blends at 25^oCA bTDC when correlated to neat base fuel, because of the improvement in the evaporation cycle and more oxygen content available in the fuel.

A remarkable increase in BTE has been found when NBM was used as fuel with injection timings 21^oCA bTDC, 23^oCA bTDC, and 25^oCA bTDC. The increase was found to be more by 3.53%, 10.12%, and 15.56%, respectively, when compared with neat diesel fuel.

NBM at 25^oCA bTDC gave inflated in-cylinder pressure, which is on par with standard injection timing with other fuels, which is because of enchanted air–fuel mixing, evaporation rate, and ID.

As a concluding remark, the tremendous reduction in emission parameters which is vital for the current scenario with the beneficial increase in BTE proves NBM with antioxidants to be a promising alternative fuel source for a diesel engine without any modification.

Table 9 shows the comparison results for different injection timing performance and emission characteristics. In the table shows two arrow symbols, one is upward arrow and the other one is downward arrow. The upward arrow symbols show increase in injection timing performance and emission characteristics and the downward arrow symbols show decrease in injection timing performance and emission characteristics. Figure 14 shows the graphical picture of the comparison results.

Declarations

Permissions

The authors have authorization to gather *Madhuca longifolia* and *Madhuca indica*. We have all permits and we certify handlings of plants were carried out in compliance with applicable rules and laws.

Viii. References

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Supplementary

Due to technical limitations, Table 7, 8 and 9 are only available as a download in the Supplemental Files section.

Figures

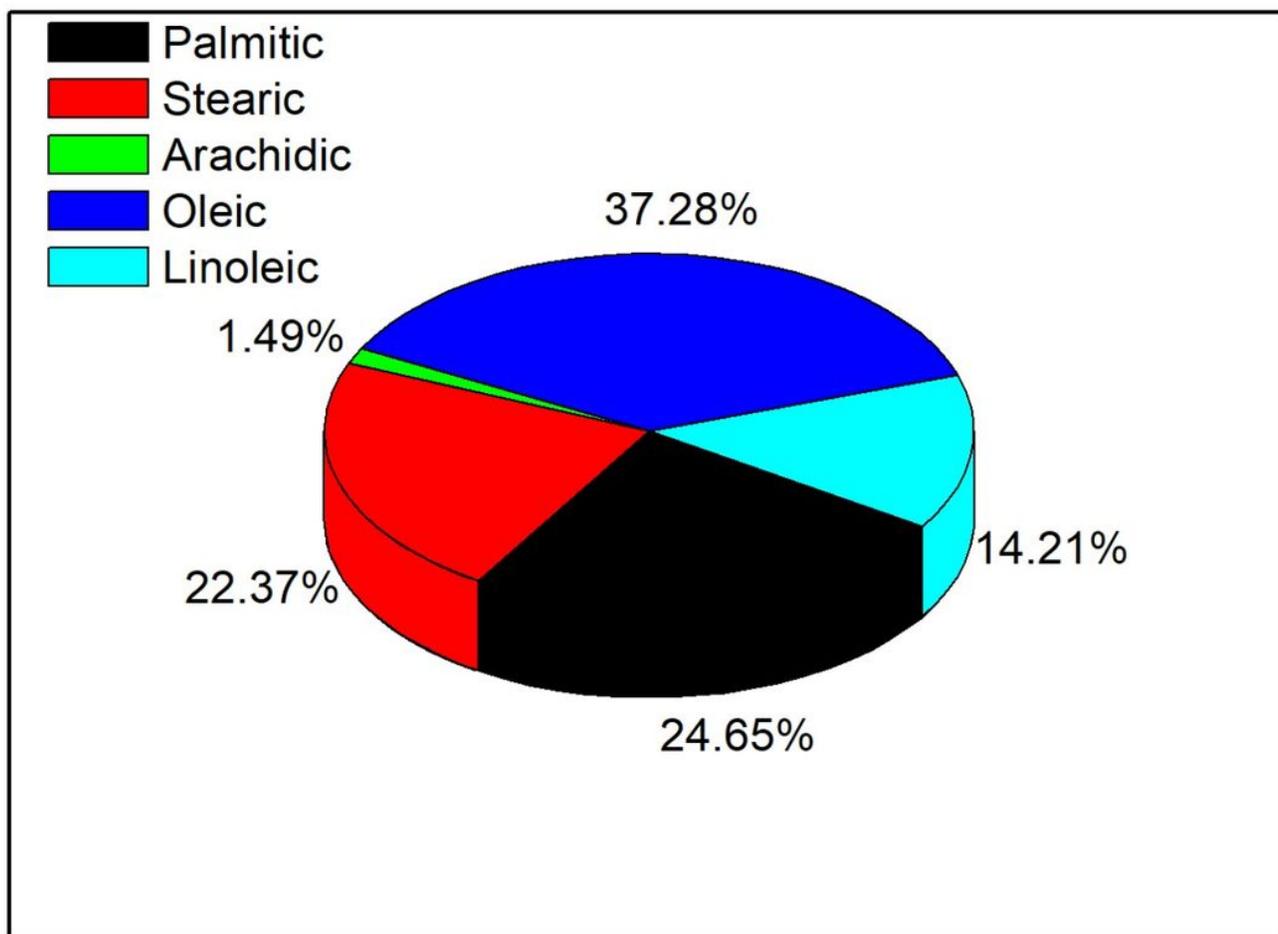


Figure 1

Fatty acid composition

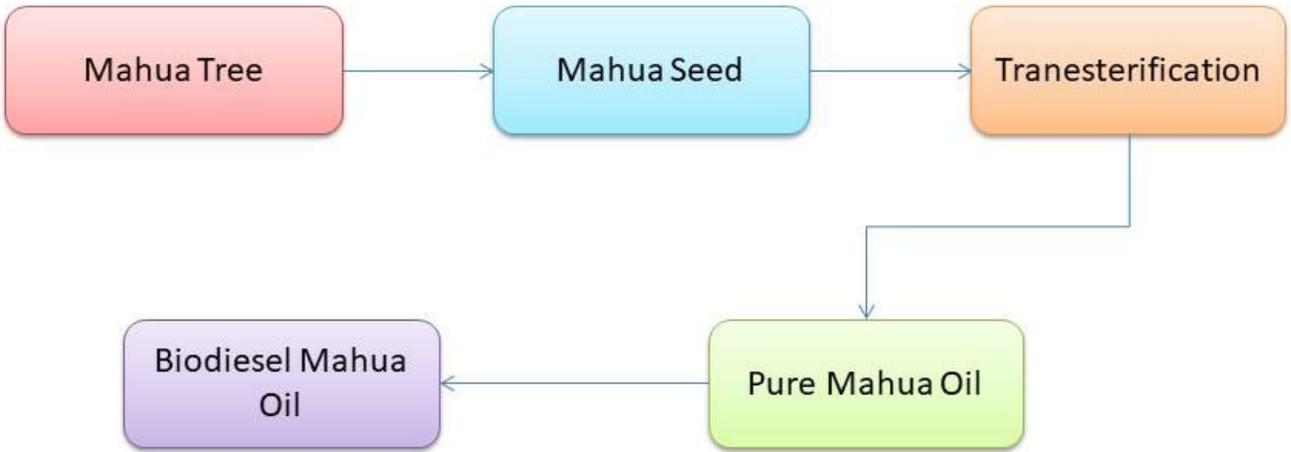


Figure 2

Oil extraction process

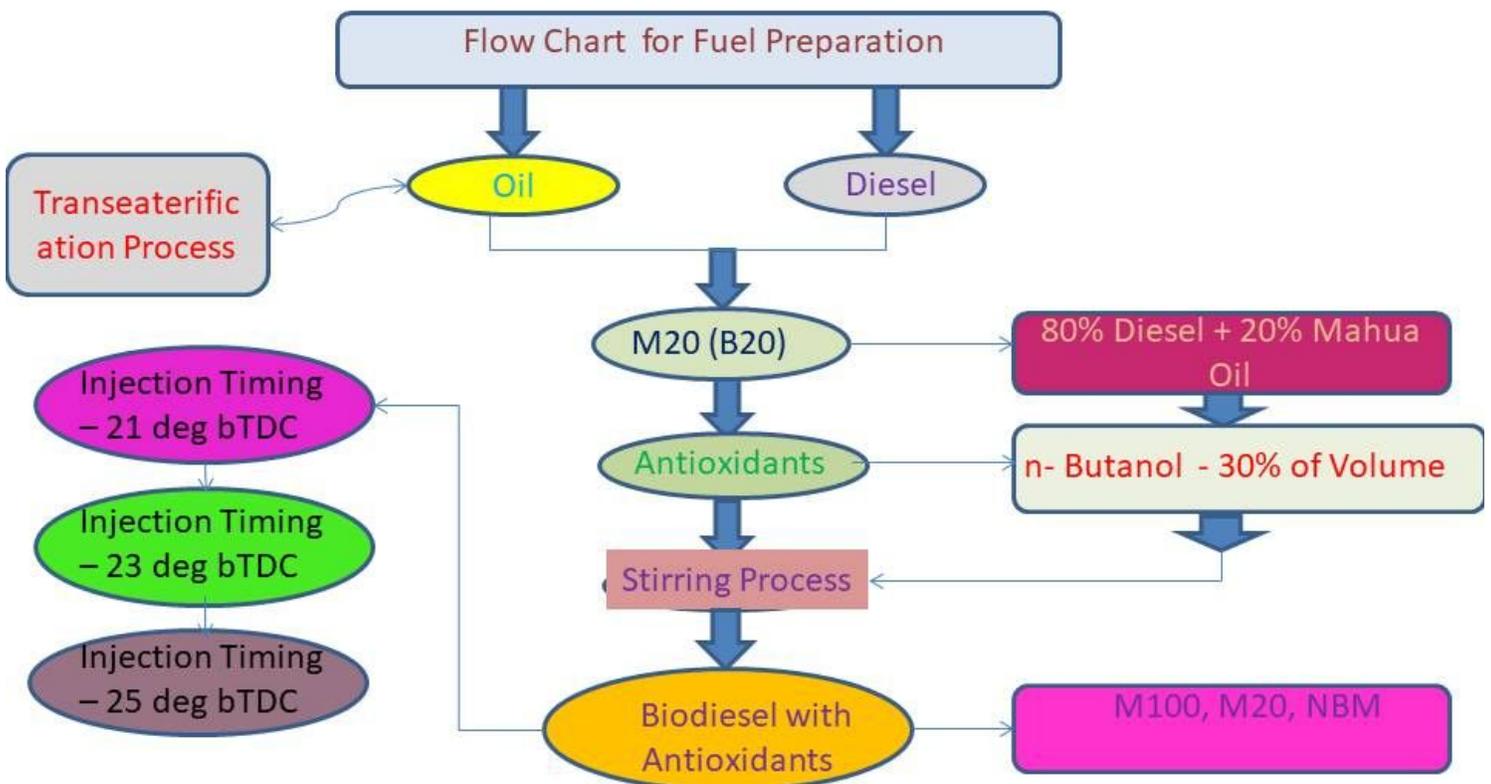


Figure 3

Fuel chart depicting preparation of mahua seed fuel

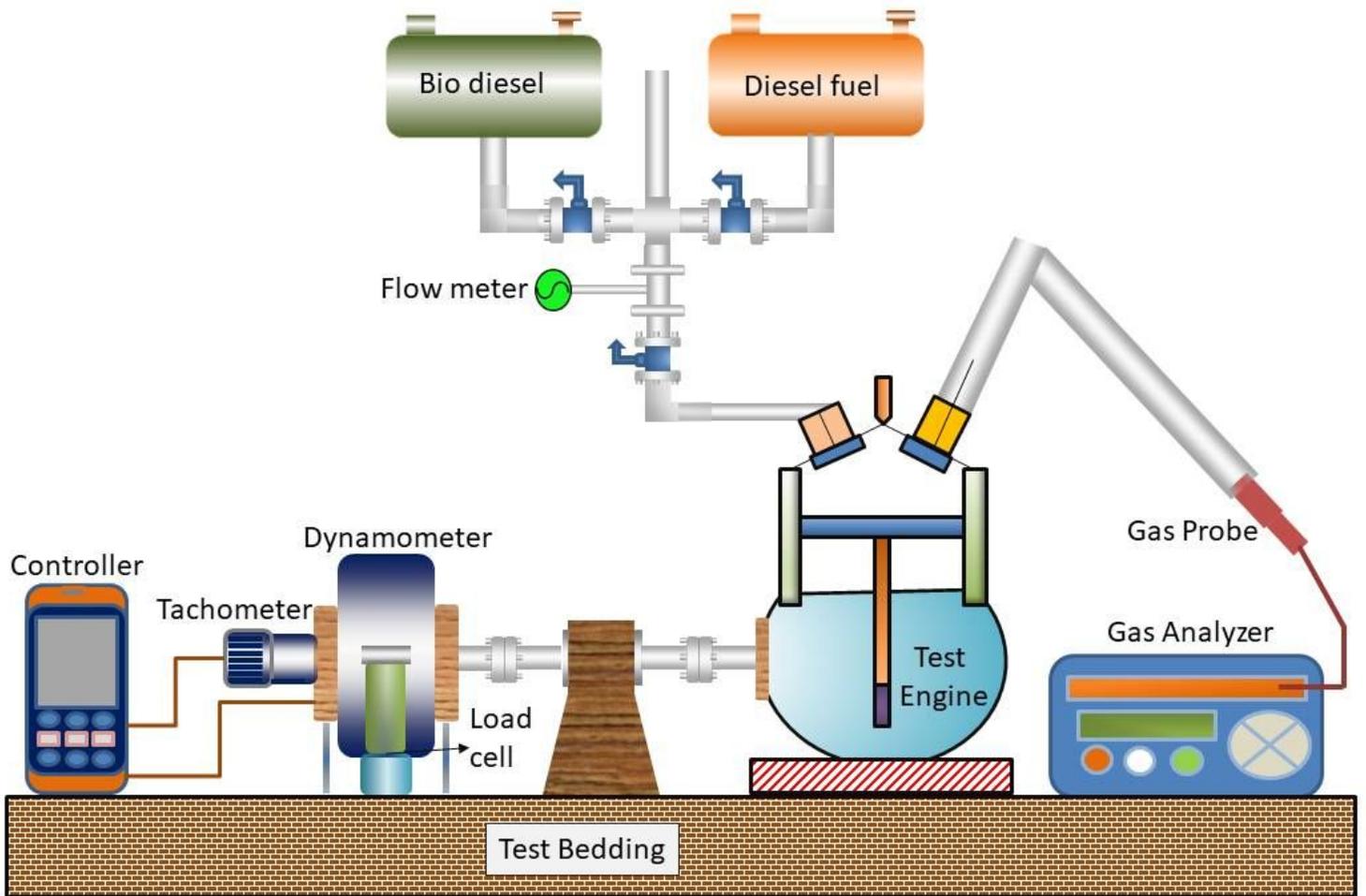


Figure 4

Engine set-up

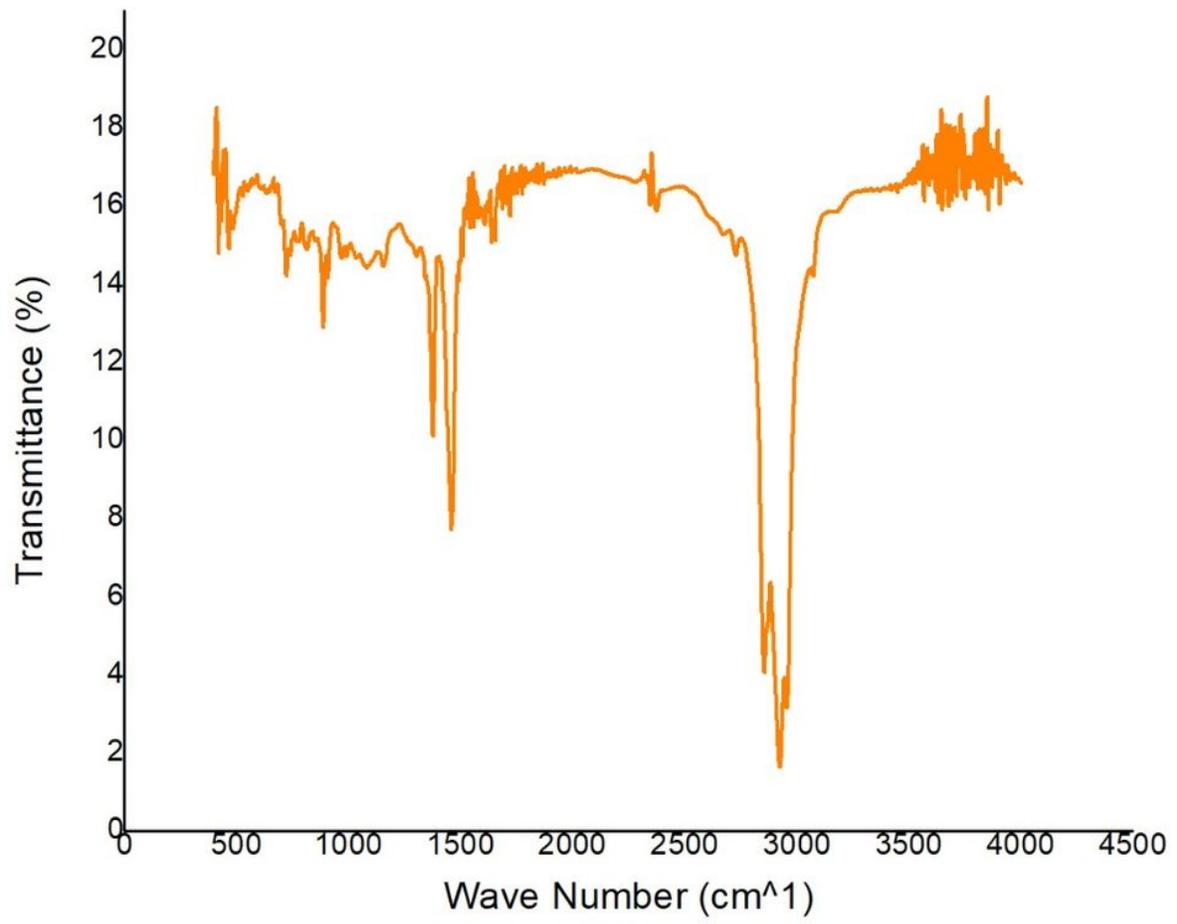


Figure 5

FTIR Analysis of Mahua Biodiesel

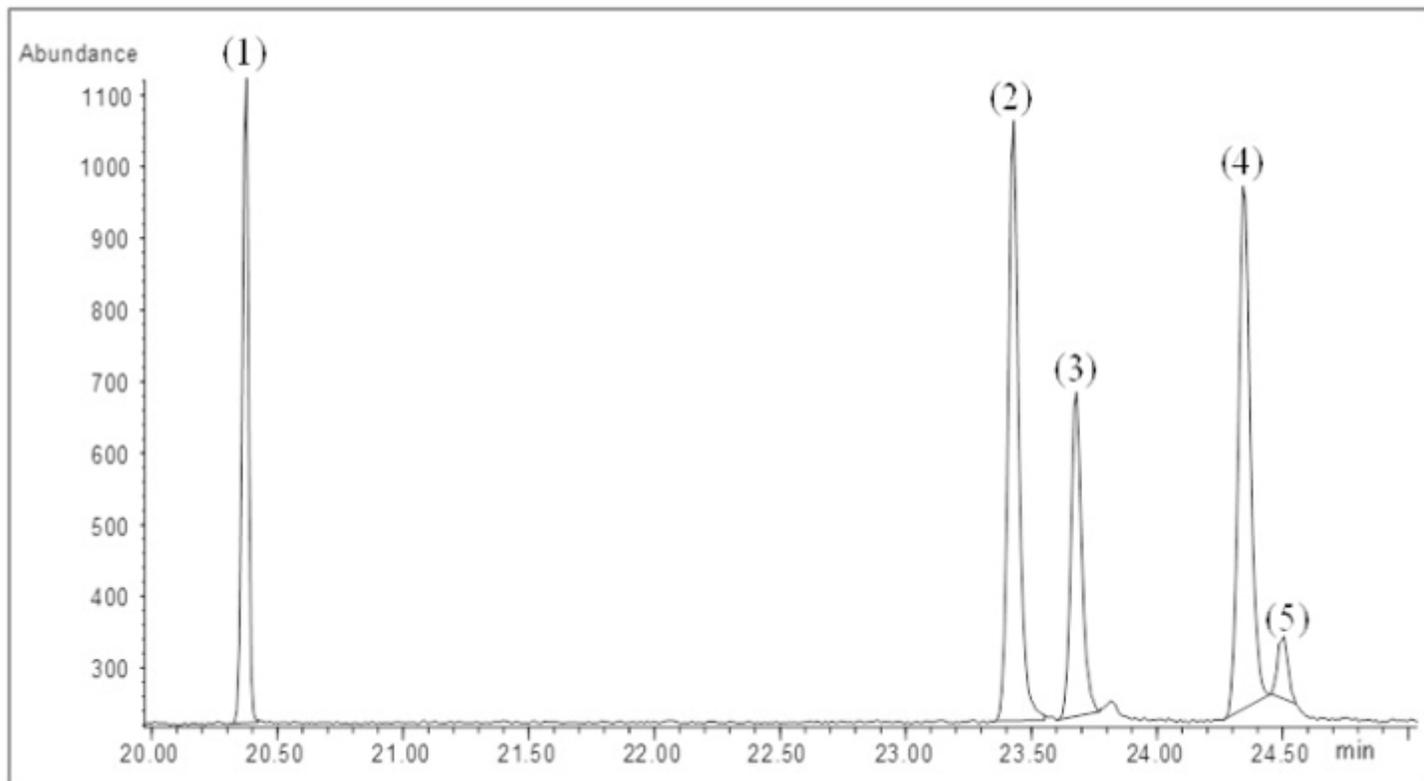


Figure 6

GC-MS of mahua biodiesel

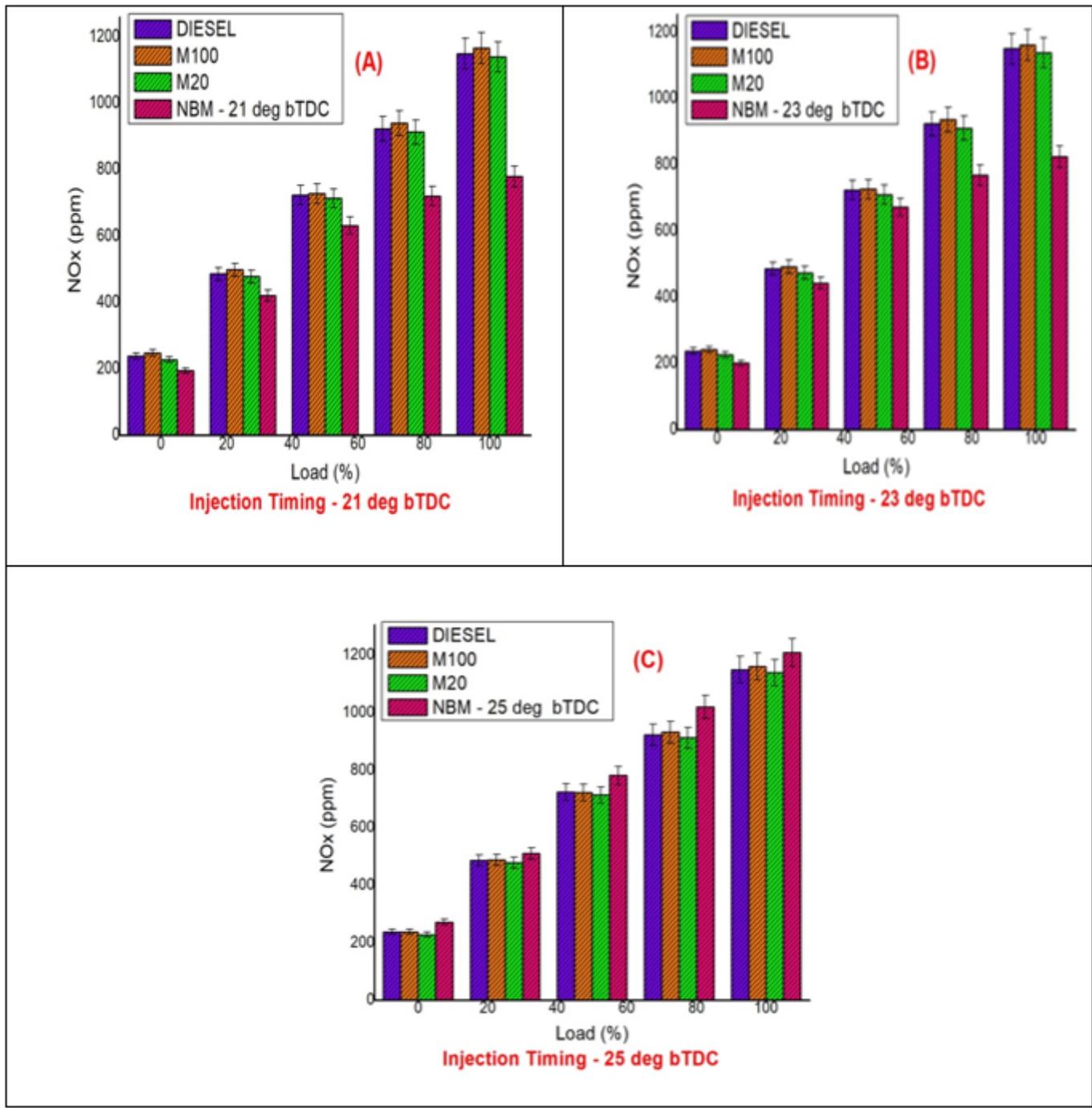


Figure 7

Variation of load vs. NOx at various blends (a) 21°CA bTDC, (b) 23°CA bTDC, and (c) 25°CA bTDC

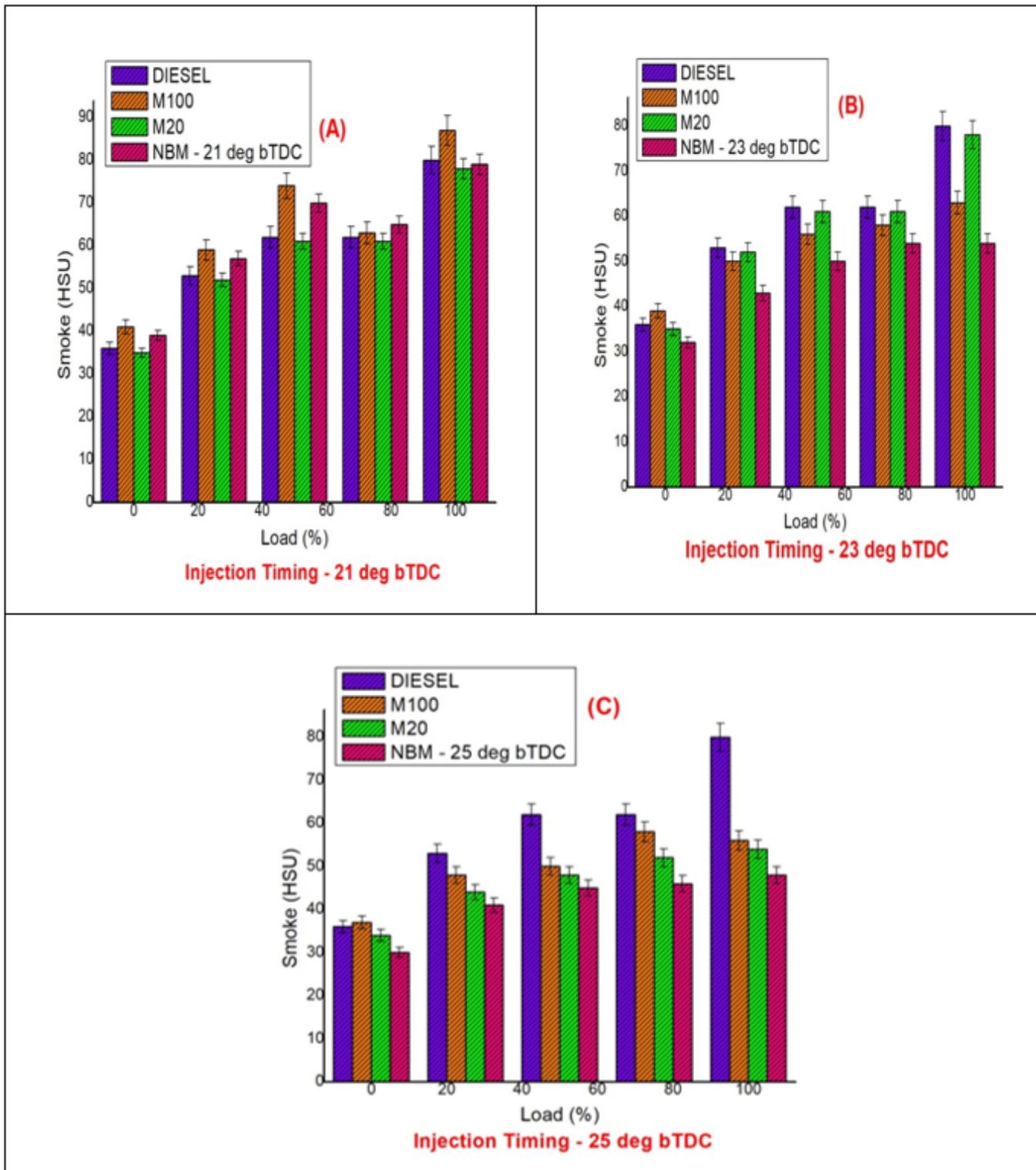


Figure 8

Variation of load vs. smoke opacity at various blends (a) 21°C bTDC, (b) 23°C bTDC, and (c) 25°C bTDC

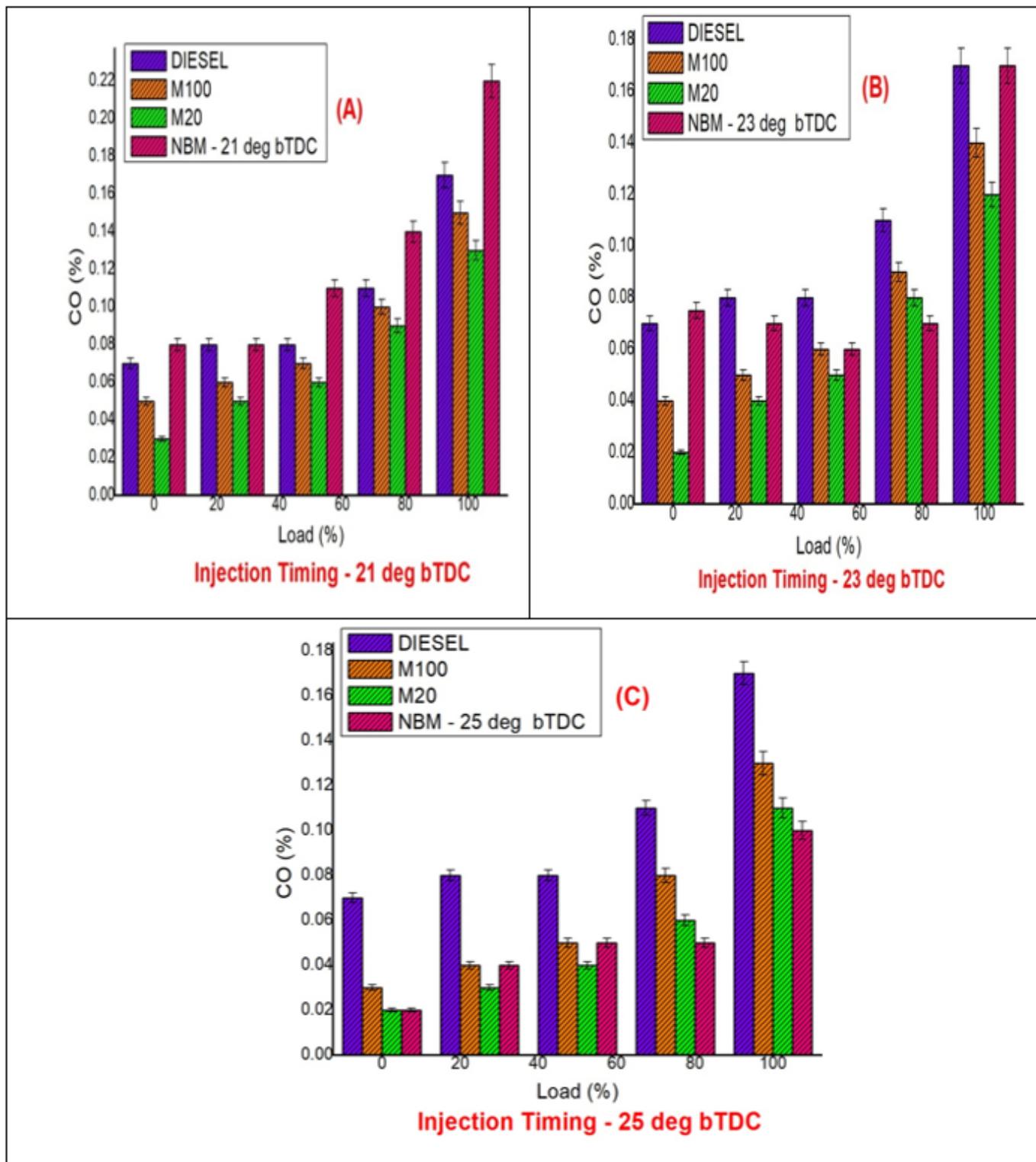


Figure 9

Variation of load vs. carbon monoxide at various blends (a) 21°CA bTDC, (b) 23°CA bTDC, and (c) 25°CA bTDC

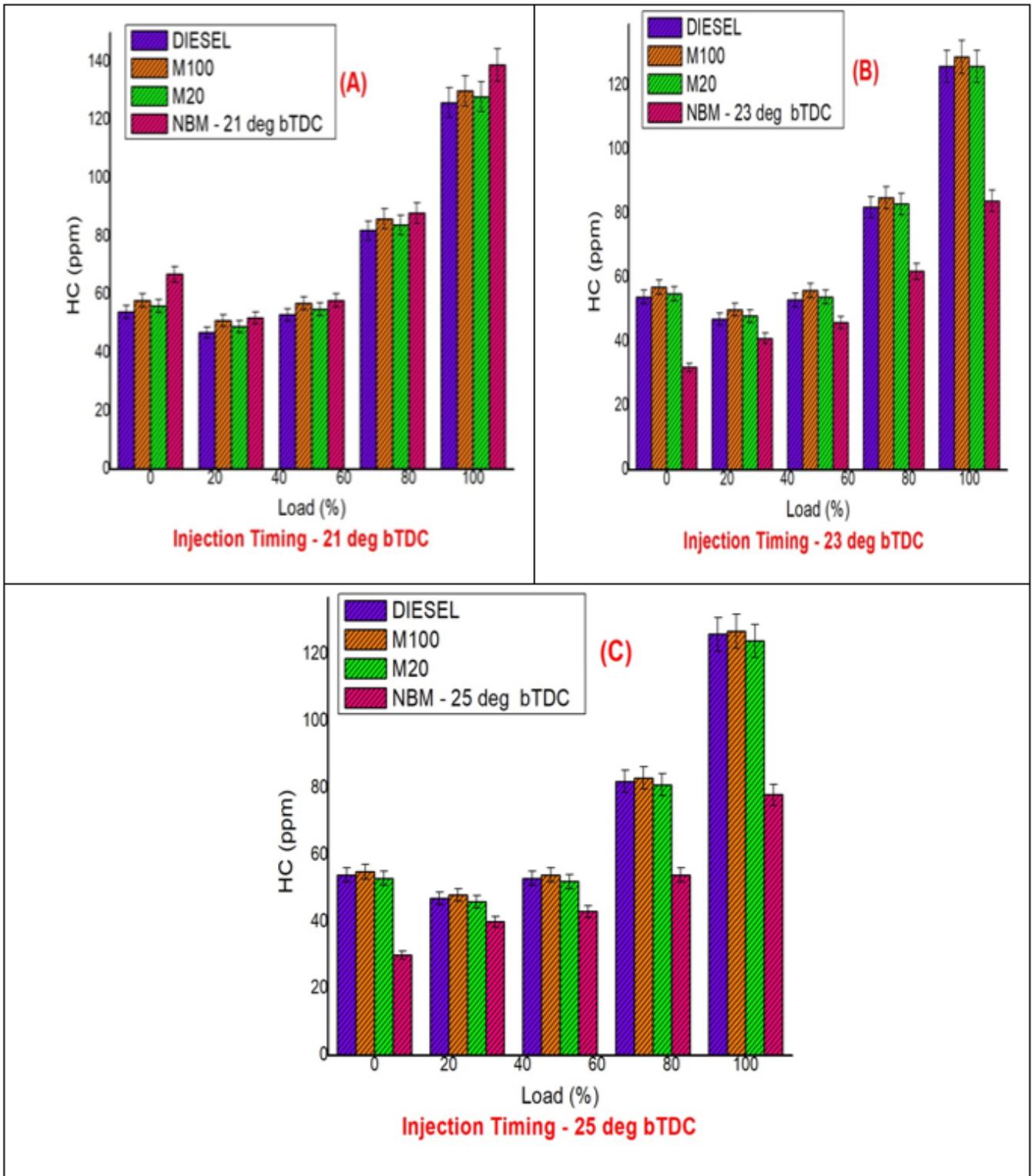


Figure 10

Variation of load vs. hydrocarbon at various blends (a) 21°CA bTDC, (b) 23°CA bTDC, and (c) 25°CA bTDC

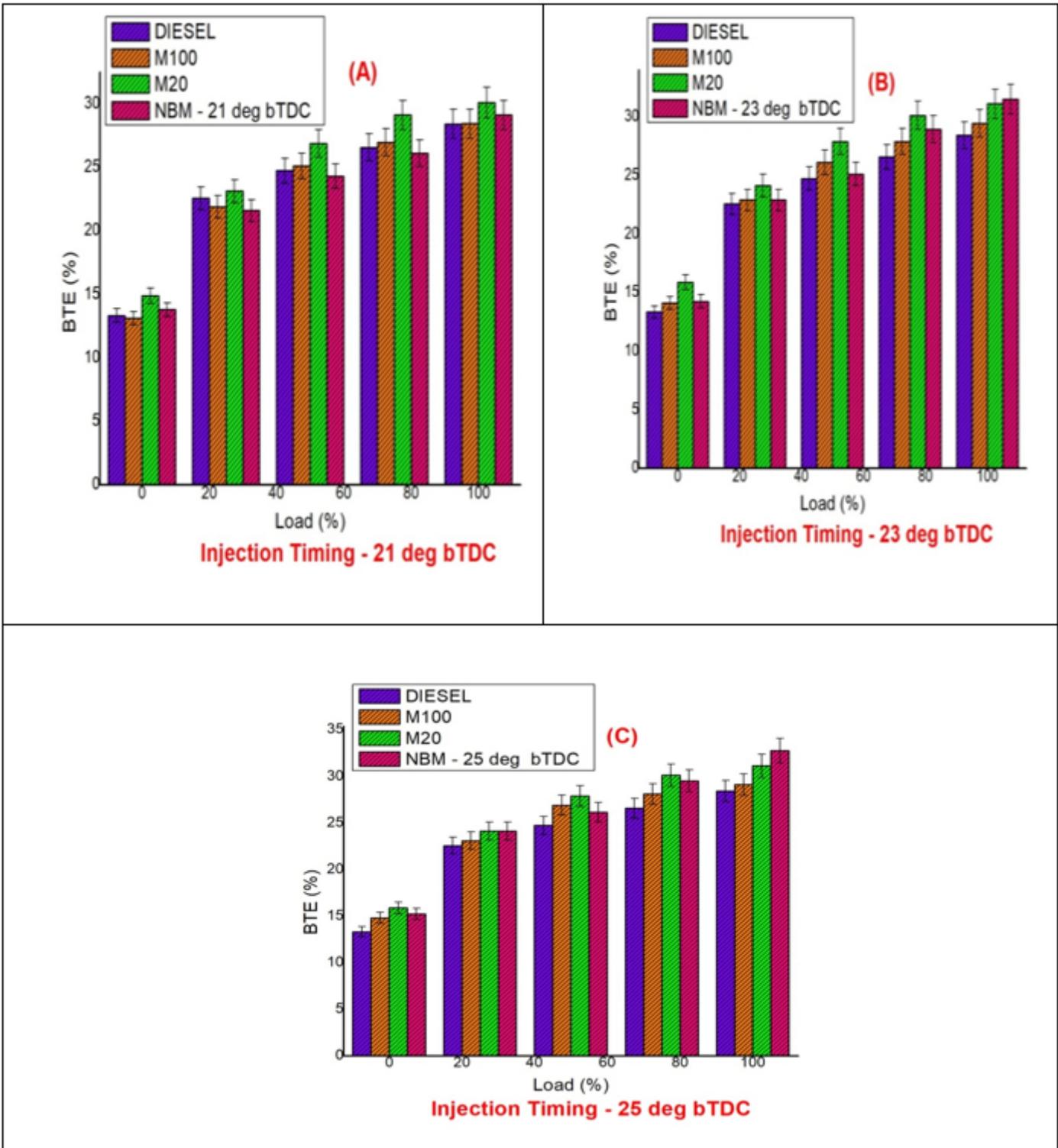


Figure 11

Variation of load vs. BTE at various blends (a) 21°CA bTDC, (b) 23°CA bTDC, and (c) 25°CA bTDC

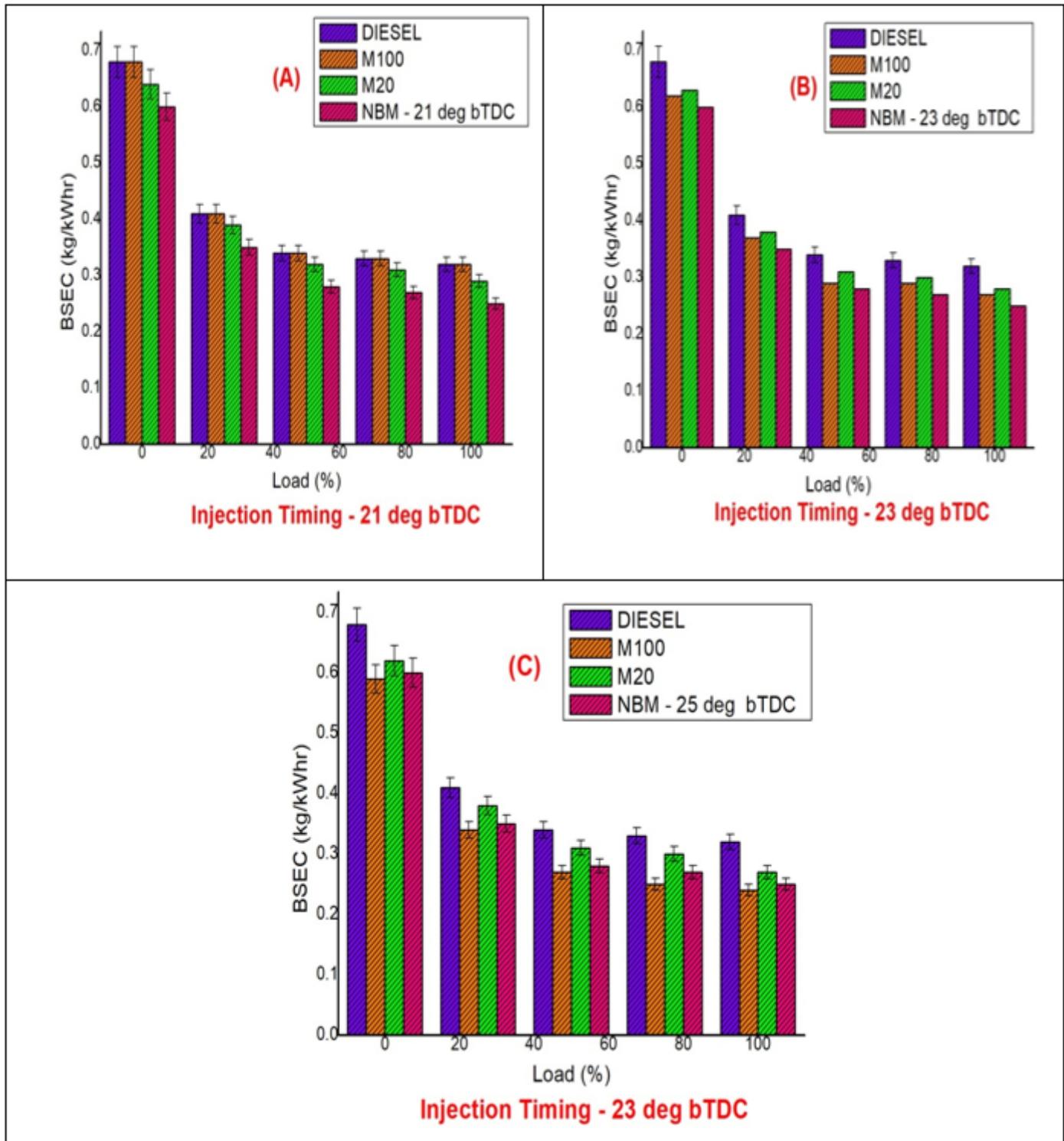


Figure 12

Variation of load vs. BSEC at various blends (a) 21°C bTDC, (b) 23°C bTDC, and (c) 25°C bTDC

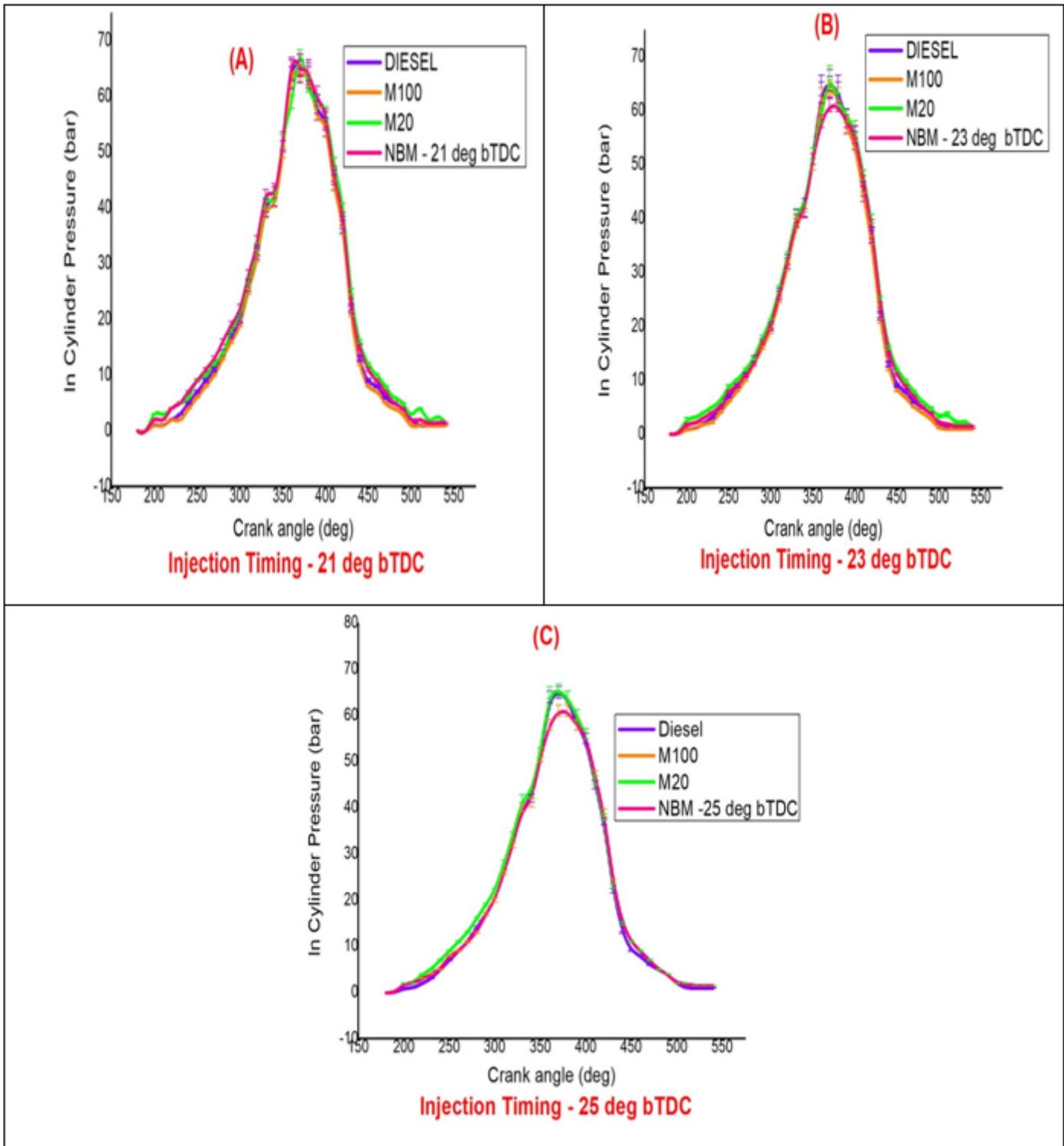


Figure 13

Variation of crank angle vs in-cylinder pressure at various blends (a) 21°CA bTDC, (b) 23°CA bTDC, and (c) 25°CA bTDC

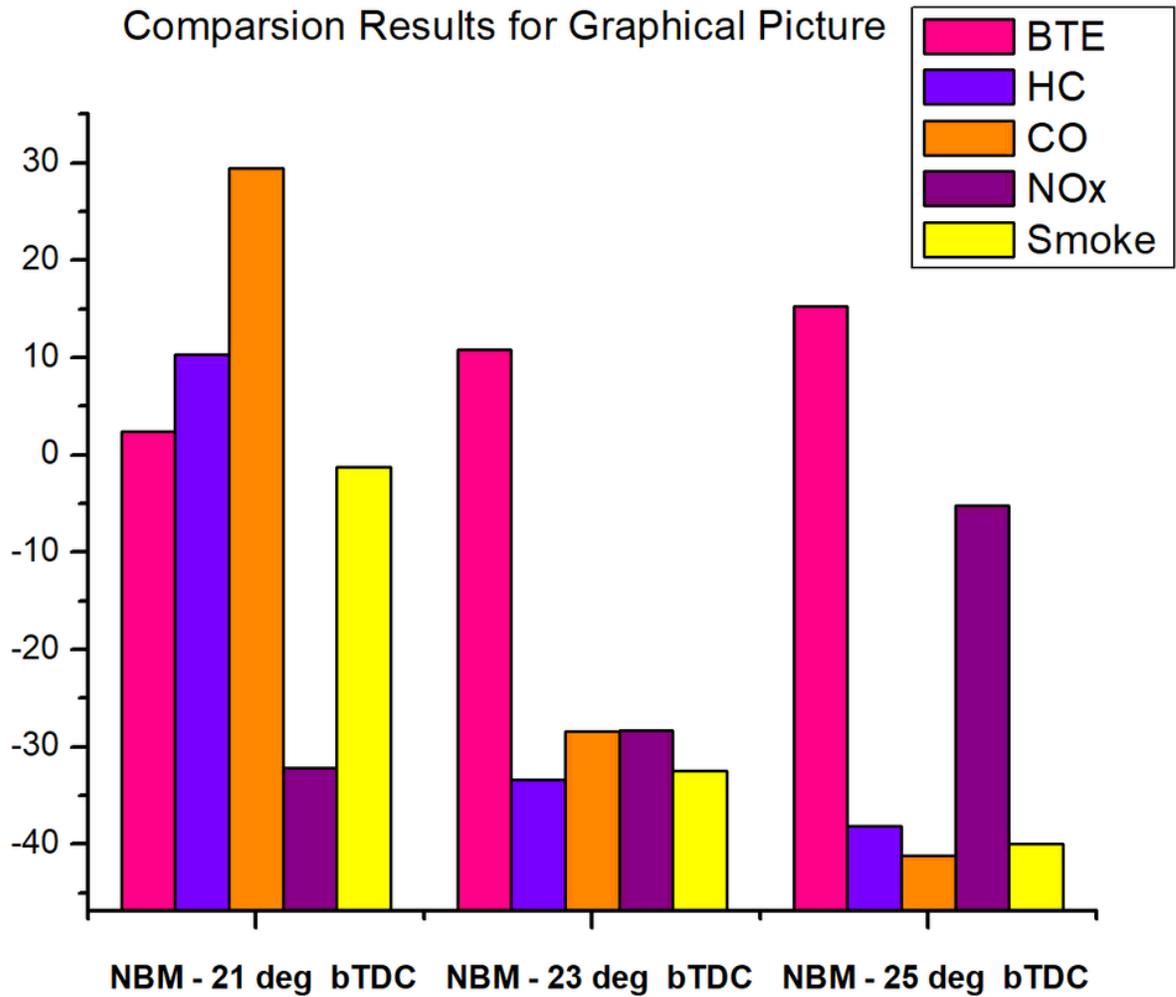


Figure 14

Graph depicting the comparison results of performance and emission characteristics

Supplementary Files

This is a list of supplementary files associated with this preprint. Click to download.

- [Table78and9.docx](#)